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THERMOELECTRICALLY COOLED CLOUD PHYSICS EXPANSION CHAMBER

Final Technical Report

by

Richard J. Buist

January 31, 1977

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Sponsored by

George C. Marshall Space Flight Center
National Aeronautics and Space Administration
Marshall Space Flight Center, Alabama 35812

Contract No.
NAS8-31633

BORG-WARNER THERMOELECTRICS
Wolf & Algonquin Roads
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PREFACE

This report describes the development of the thermoelectric cooling of an expansion chamber for use in cloud physics experimentation. Work was performed at Borg-Warner Thermoelectrics, Des Plaines, Illinois under contract #NAS8-31633 awarded by the George C. Marshall Space Flight Center, Huntsville, Alabama. The contract was administered by C. A. Law, Contracting Officer and B. Elkins, Contracting Officer's Representative.

The author wishes to acknowledge the many significant contributions by Borg-Warner Thermoelectrics personnel: Dr. J. S. Lee in the area of heat sink design, thermal modeling, and acceptance testing; R. W. Duenn who served as Project Manager and Production Manager; B. Peters, L. Shaw, and L. Black in the area of hardware production; F. Smith in quality control; and K. Ryan, Administrative Assistant.

Finally, the author is especially grateful for the many helpful discussions and guidance by G. Barr and J. Moses, Contracting Officer's Technical Representative, of the Marshall Space Flight Center.

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1. SUMMARY

The major objective of this program was to provide means of controlling the temperature of an expansion chamber for use in cloud physics experimentation. The original concept as proposed by MSFC was to attach single stage thermoelectric modules in a close-packed configuration over the entire surface of the chamber.

The pursuance of this program involved four distinct phases of activity and conceptualization. Phase 1 was characterized by the development of a mechanical design, thermal modeling, and computer programming. Initial calculations were based on the approach as outlined in the original MSFC RFP -- that of applying single stage modules directly to the chamber. This approach proved to be marginal requiring utilizing high quality thermoelectric materials, the maximum power budget, and the most optimistic space lab water supply conditions.

Phase 2 added a thermoelectric pre-cooler between the space lab water supply and the heat sinks of the chamber thermoelectric modules. Although greatly improving the system steady-state performance, the cool-down time was not acceptable to MSFC.

Phase 3 eliminated the pre-cooler and replaced the single stage chamber thermoelectrics with multi-stage cascades. The cool down time was enormously improved without sacrificing the steady-state performance improvement made in Phase 2. However, this approach put a large ΔT across the thermoelectrics in contact with the chamber and thereby augmented potential isothermal problems.

It was at this point in the program that BWTE was made aware of a given transient cooling requirement. This requirement was not part of the contract but was of the overall program. Attempts to simulate the desired transient performance led to the conclusion that this condition could not be met under the imposed constraints. A simple calculation from the cooling slope required and the mass of the chamber revealed that a heat extraction rate of 2960 watts from the chamber at -10°C (14°F) was required. The maximum heat extraction rate of even a 5-stage thermoelectric optimized for that temperature using the full 1.5 kw power budget was less than 500 watts.

The fourth and final phase of the program involved a complete change in the scope of effort. A contract modification was made redefining the objectives in a revised statement of work. In general, this involved the design and fabrication of a thermoelectric chiller for use in cooling a liquid reservoir. This removed responsibility for cool down of the chamber, itself, from BWTE.

A subsequent reservoir chiller was designed, fabricated, tested, and delivered to MSFC. An acceptance test was designed complete with a reservoir thermally similar to that anticipated for the end item usage. Test results established the accuracy of the thermal model as well as unit compliance with all specifications as stated in the modified contract. Finally, calculations were made using this thermal model in order to predict the unit performance under various other conditions.

2. INTRODUCTION

Thermoelectrics are finding increased utilization in aerospace applications as a result of characteristics particularly compatible with requirements. Thermoelectrics have the advantage of being able to heat, cool, and modulate temperatures by a simple feed back input current control. Having no moving parts, these control systems are generally lighter, smaller-sized, and are potentially more effective and reliable than competing temperature control systems.

The concept of applying thermoelectrics to cloud chambers was introduced by Dr. Kassner of the University of Missouri-Rolla. Borg-Warner Thermoelectrics (BWTE) supplied the thermoelectric modules and provided some technical assistance for this application. BWTE had and still has reservations about achieving the degree of isothermal precision required over a large set of thermoelectrics. However, BWTE has made every effort to be fully responsive to program requirements even though not formally committed contractually. Indeed, design conceptualization underwent several phases in response to Marshall Space Flight Center's (MSFC) requests even though each concept technically met all contract requirements. The final phase involved a redefinition of the scope of effort and corresponding contract modification. All requirements of the modified contract were met by BWTE.

3. PHASE 1 - MODULES

The initial phase of this program was discussed in detail in the BWTE proposal dated May 28, 1975 submitted in response to MSFC RFP #8-1-5-60-01462. Basically, it involved the thermal/mechanical design of a system for cooling an expansion cloud chamber using single stage thermoelectric modules (see Figures 3.1 and 3.2).

3.1 Mechanical Design

Thermoelectric modules were evenly distributed over the surface of the cloud chamber. On sections 2 and 3, six modules were assumed on each of the 32 flats, or a total of 192 modules per section. Maintaining the same "packing density" of thermoelectrics, 64 modules were placed on each end of the cloud chamber, sections 4 and 5 as well as on the middle section 1. Half modules were used around the viewports and on the cloud chamber ends and were included in the total module count. The total number of modules in the system was, therefore, 576.

3.1.1 Heat Sink Considerations - For maximum isothermal conditions, counterflow aluminum heat sinks were chosen. Three separate heat sinks were considered to cover each of the 32 flats around the periphery of the cloud chamber, one heat sink each on sections 2 and 3, and one on section 1. The water inlet and outlet was provided on each end of the chamber. Additional heat sinks covering the modules on the ends of the chamber were also fed from the water input at the

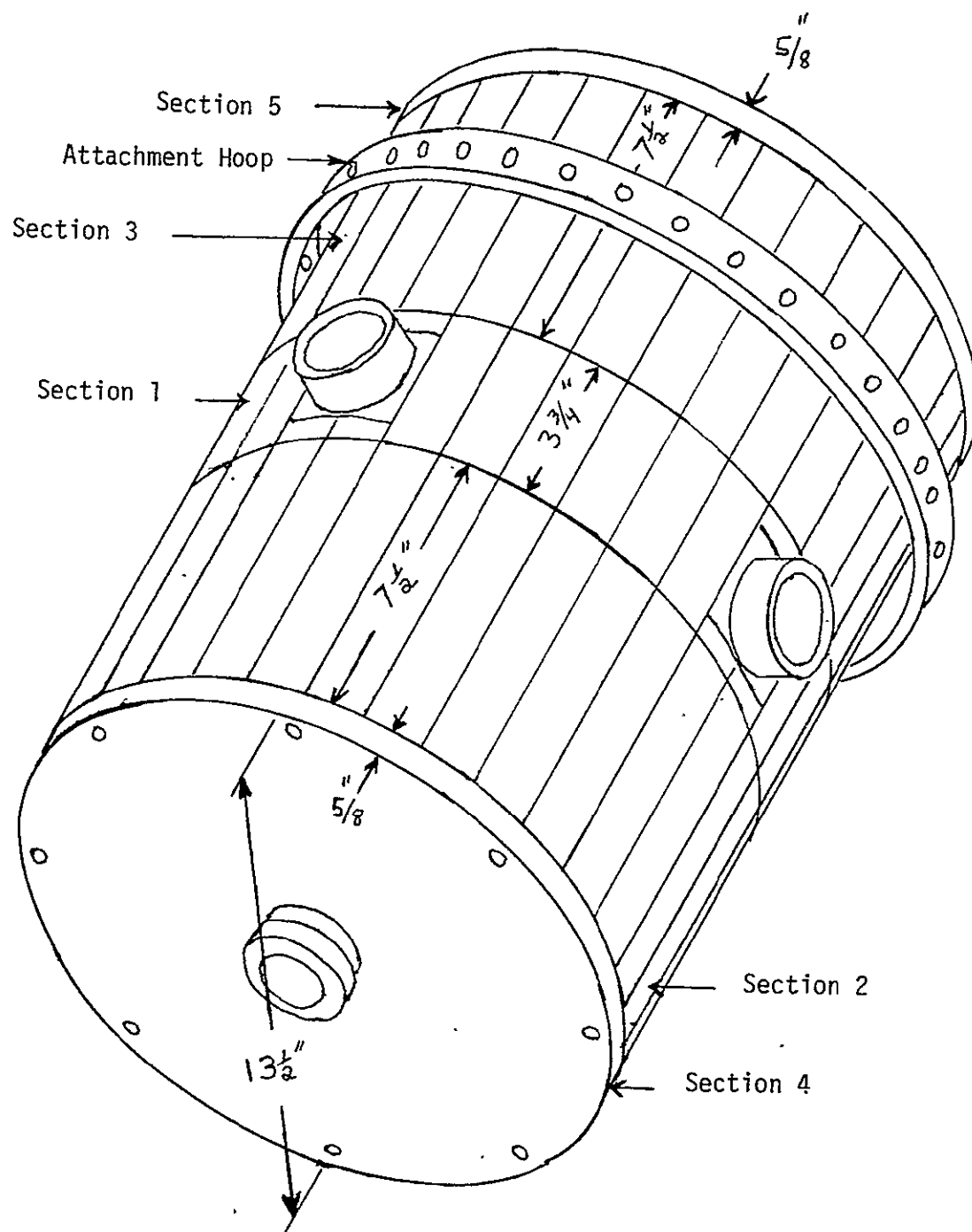


FIGURE 3.1 Expansion Cloud Chamber

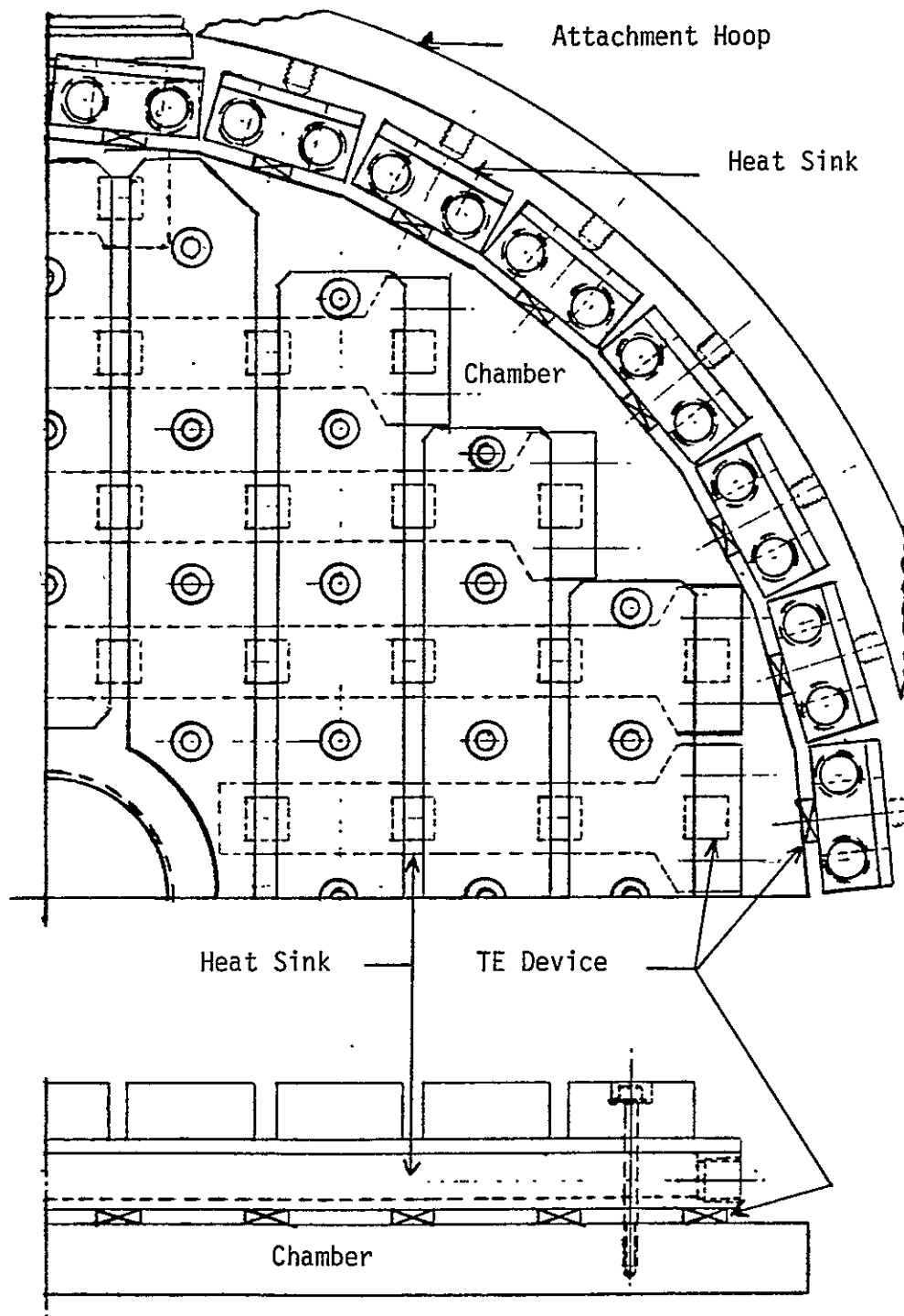


FIGURE 3.2 Method of TE Device Attachment

respective ends. To provide parallel flow equalizing pressure drop and flow rate in each section, a "gas burner type" manifold for the input and output water was provided at each end of the chamber.

3.1.2 Physical Attachment - To minimize the thermal attachment load a "hoop" type compressive mounting system was developed for this application. A series of cylindrical hoops with inside dimensions approximately 1/2" larger than the outside dimension of the heat sinks, drilled and tapped to hold set screws, were provided. A set screw in the hoop was positioned over each heat sink. Tightening the set screws evenly provided balanced clamping forces to each heat sink. Using this system there was no thermal contact between the heat sinks and the cloud chamber, and consequently no attachment heat load. On the ends of the chamber the mounting system is more difficult, so 128 1" long insulated stainless steel screws were provided to mount the heat sinks. With this mounting design concept developed, the initial system thermal analysis and module design was initiated.

3.2 Thermal Design

3.2.1 Requirement Summary

3.2.1.1 Chamber Temperature - The chamber temperature was to be operated at a maximum temperature of 110°F and a minimum of -15°F.

3.2.1.2 Heat Sink - The heat exhausted by the thermo-

electrics was to be dissipated by a heat sink which was, in turn, cooled by a flow of water. The water inlet temperature was 60 to 80°F. The water flow rate was 100-300 lbs/hr.

3.2.1.3 Supply Voltage - The available ranges of supply voltage were 15-22 volts, or 32-40 volts. Maximum system power budget was 1.5 kilowatts.

3.2.1.4 Performance Prediction - The cool down rate and minimum temperature obtainable with the proposed system was to be estimated.

3.2.1.5 Thermal Loads - The thermal load distribution from various sources was to be determined together with a development of techniques to minimize these various thermal loads.

3.2.1.6 Module Design - The optimum number of modules to be series-connected was to be determined. The smallest adequate lead wire size was to be used.

3.2.2 Thermal Load Calculations - Initial calculations of the heat sink water temperature rise at maximum input power indicated a minimum of 20 to 60°F depending on water flow rate. Consequently, equations were developed for each source of thermal load as a function

of the cold chamber temperature, the heat sink temperature, and ambient temperature.

3.2.2.1 Radiation - The view factor of the radiation surfaces was assumed to be 1.0. The emissivity of these surfaces was assumed to be 0.4. The effective emissivity of the two plates was calculated according to equation 3.2.2.1.1:

$$3.2.2.1.1 \quad \epsilon = \frac{1}{\frac{1}{\epsilon_1} + \frac{1}{\epsilon_2} - 1} = 0.25$$

where: $\epsilon_1 = \epsilon_2 = 0.4$

The equation for the total radiative thermal load on the cold chamber is given in equation 3.2.2.1.2:

$$3.2.2.1.2 \quad QR = \sigma \epsilon A_o (THHS^4 - TCCH^4)$$

where: $\sigma = 3.659 \times 10^{-11} \frac{\text{Watts}}{\text{in}^2 \text{K}^4}$

TCCH = Chamber Temperature

THHS = Heat Sink Temperature

A_o = Chamber Outside Area

3.2.2.2 Convection/Conduction - The ambient air pressure for convection/conduction loss calculations was assumed to be 15 psia. The thermal load for this condition was calculated by simply assuming air conduction loss between the plates. It was determined that natural convection losses in this air space were negligible compared with conduction losses. The equation for air conduction loss is given in equation 3.2.2.2.1:

$$3.2.2.2.1 \quad QC = \frac{A_o}{L} \kappa_a (THHS - TCCH)$$

where: L = Distance Between Plates

κ_a = Thermal Conductivity in Air

3.2.2.3 Windows - The proximity of the windows to the heat sinks was such that the heat sink temperatures were not instrumental in the thermal load due to the windows, i.e. the thermal load due to the windows was a function of the ambient temperature and the chamber temperature. The equation for the heat load due to windows is given in equation 3.2.2.3.1:

$$3.2.2.3.1 \quad QW = h A_w (TAMB - TCCH)$$

where: $h = 2 \text{ BTU/hr ft}^2 \text{ } ^\circ\text{F}$

$A_w = 169 \text{ in}^2$

$TAMB$ = Ambient Temperature

3.2.2.4 Lamps - Two 30-watt lamps were assumed with 89% transmission losses. Therefore, the thermal load due to the lamps was 6.6 watts constant for the system.

3.2.2.5 Attachment - The mechanical description formulated in Section 3.1 indicated the usage of 128 one inch long stainless steel #4 machine screws to mount the thermoelectric modules in compression between the heat sinks and chamber on either end of the chamber. The following equation was generated which expresses the heat short or thermal load due to these stainless steel screws:

$$3.1.2.2.5.1 \quad QA = N_S \kappa_S \frac{A_S}{L_S} (THHS - TCCH)$$

where: N_S = Number of screws

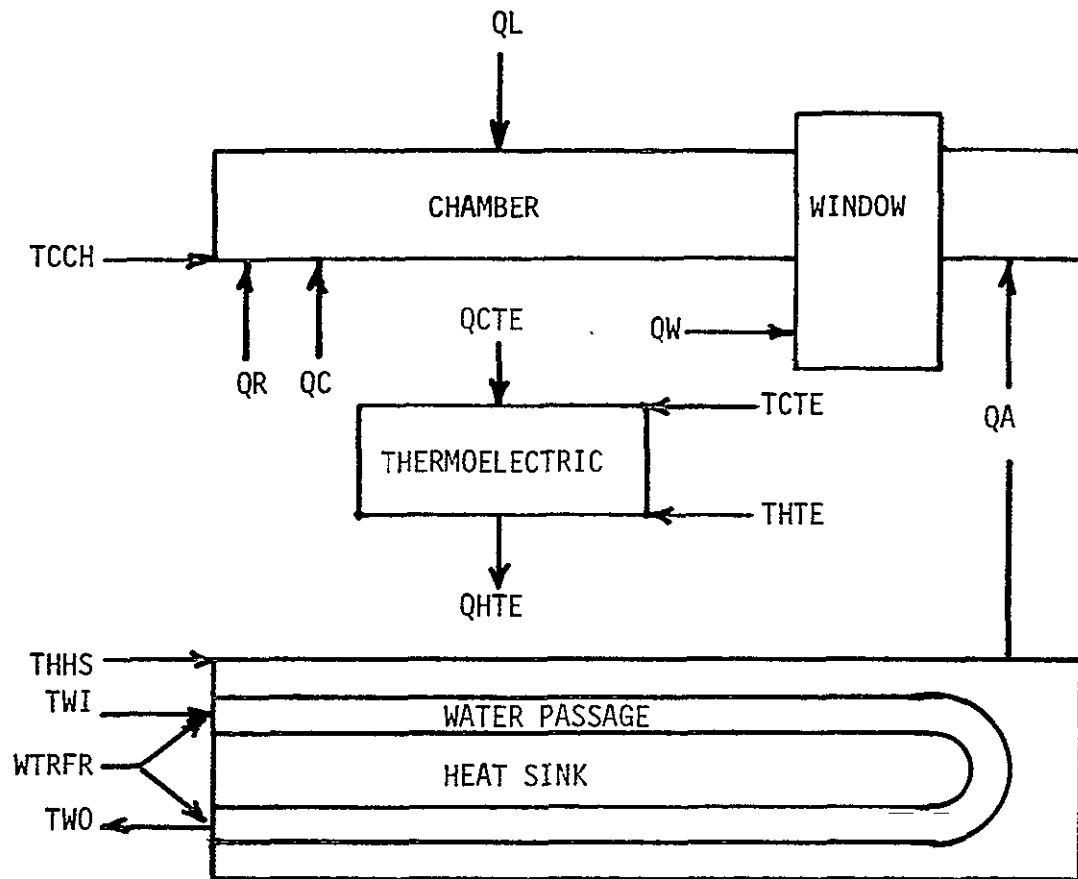
κ_S = Thermal conductivity of screws

A_S = Cross-sectional area of one screw

L_S = Length of screw

3.2.3 Design Model - The mathematical model used for design calculations is shown in Figure 3.2.3.1. Indicated thermal loads were calculated according to the equations developed in Section 3.2.2. The water flow passage was designed to form a single loop to minimize thermal gradients in the heat sinks. A nominal value of 5W/°C/in was assumed for the thermal contact resistance between the modules and mounting surfaces.

Thermal analysis of the system indicated a significant heat sink limitation resulting in a relatively high heat sink temperature rise in the maximum power condition. Consequently, it was discovered that the combination of highest flow rate, 300 lb/hr, and lowest temperature, 60°F, of coolant water together with the optimized application of the highest quality thermoelectric material was required to meet the low temperature specification. The results of the design calculations are given in Table 3.2.3.1:



THERMAL LOADS

QR - Radiation Load
 QC - Conduction Load
 QL - Lamp Load
 QW - Window Load
 QA - Attachment Load

TEMPERATURES

TCTE - Cold Side of Thermoelectrics
 THTE - Hot Side of Thermoelectrics
 TCCH - Chamber

HEAT SINK

TWI - Inlet Water Temperature
 TWO - Outlet Water Temperature
 • WTRFR - Water Flow Rate

HEAT FLUX

QCTE - Heat Pumped by Thermoelectric Module
 QHTE - Heat Exhausted by Thermoelectric Module

FIGURE 3.2.3.1 Thermal Design Model

Table 3.2.3.1
THERMOELECTRIC MODULE SYSTEM DESIGN DATA

<u>Mechanical Description</u>		<u>System Power Conditions</u>	
HT SNK (WT*MIN/F)	= 202.00	APPLIED VOLTAGE	= 20.0
CHAMBER (WT*MIN/F)	= 255.00	INITL CURNT (AMPS)	= 90.6
INSD AREA (SQ IN)	= 907.00	FINAL CURNT (AMPS)	= 74.1
OUTSD AREA (SQ IN)	= 1120.00	INITL POWER (WATTS)	= 1811.1
HEAT SINK EFF	= 0.95	FINAL POWER (WATTS)	= 1481.2
WTR FLW RT (LB/HR)	= 300.00		
<u>Thermoelectric Parameters</u>		<u>Thermal Loads (Watts)</u>	
NUMBER OF STRINGS	= 18.000	RADIATION	= 45.40
MODULES/STRING	= 32.000	CONV/COND	= 193.41
COUPLES/MODULE	= 7.000	WINDOWS	= 51.52
T.E. L/A (/CM)	= 7.820	ATTACHMENT	= 20.42
T.E. COP	= 0.216	LAMPS	= 6.60
		TOTAL	= 317.42
<u>Temperatures (Deg F)</u>			
CHAMBER	= -14.91		
T.E. COLD	= -15.11		
T.E. HOT	= 82.35		
HEAT SINK	= 81.29		
WATER IN	= 60.00		
WATER OUT	= 80.23		

3.3 Transient Performance Calculations

A computer simulation program was generated with the constraints outlined in the thermal model shown in Figure 3.2.3.1. The pertinent results are shown in the transient cool down performance curve of Figure 3.3.1. It is observed that the chamber temperature is very nearly at equilibrium within one hour after power turn-on, and is

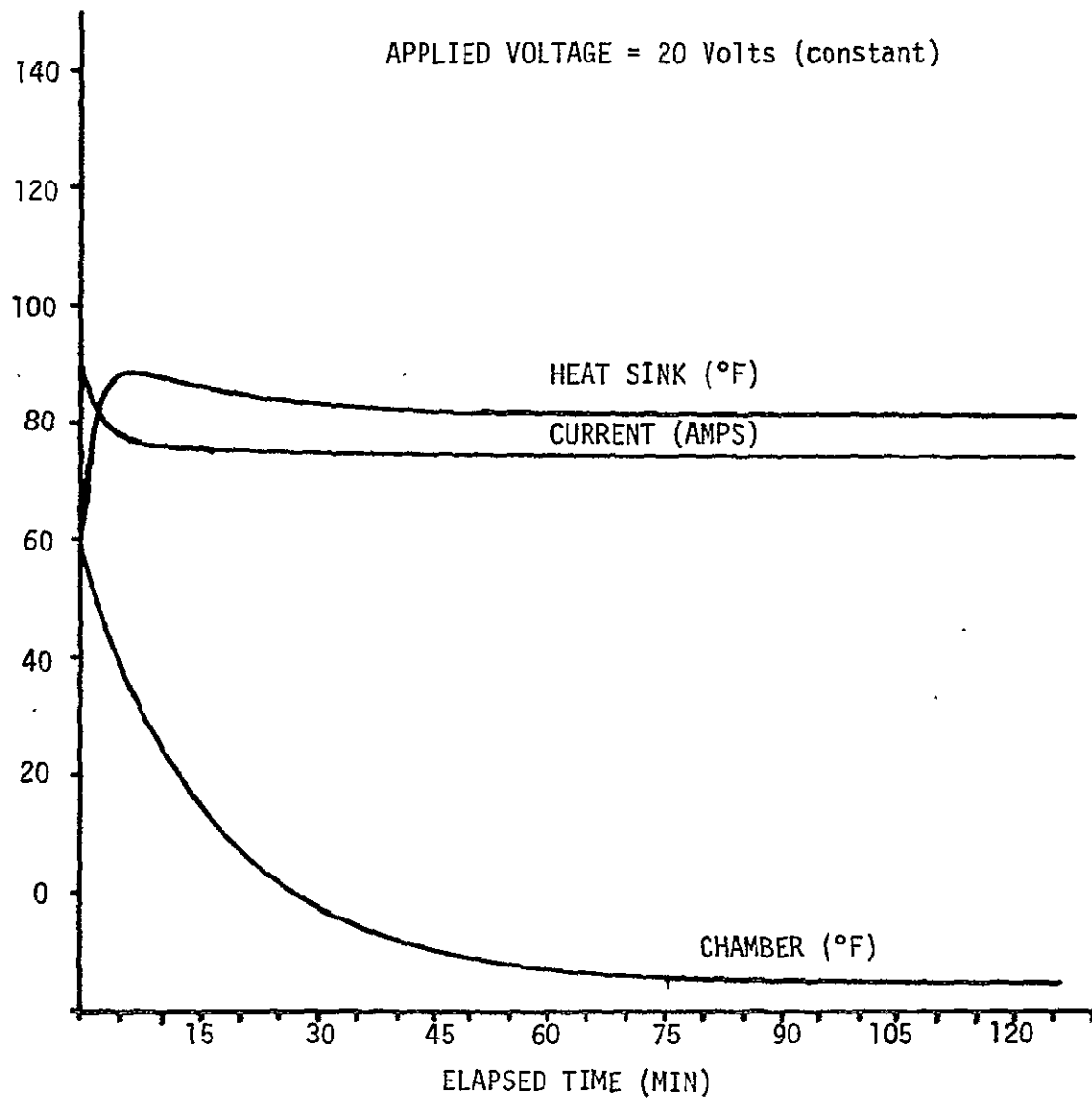


FIGURE 3.3.1 Transient Cool-Down Performance
(Single-Stage TE)

truly at equilibrium within two hours after initial turn-on. Figure 3.3.2 illustrates the build-up of the thermal load components as a function of time after initial power turn-on.

3.4 Conclusions

The concern for the design generated in this initial phase of the contract was that the assumed conditions of 60°F space lab water at 300 lb/hr flow rate and full power budget of 1.5 kw could not be realized simultaneously on a continual basis in practice. Furthermore, concern was expressed by BWTE that the large module ΔT required (97.5°F) would augment thermal variance between modules. Therefore, a pre-cooler concept was introduced to correct these shortcomings and a second phase of the program was initiated.

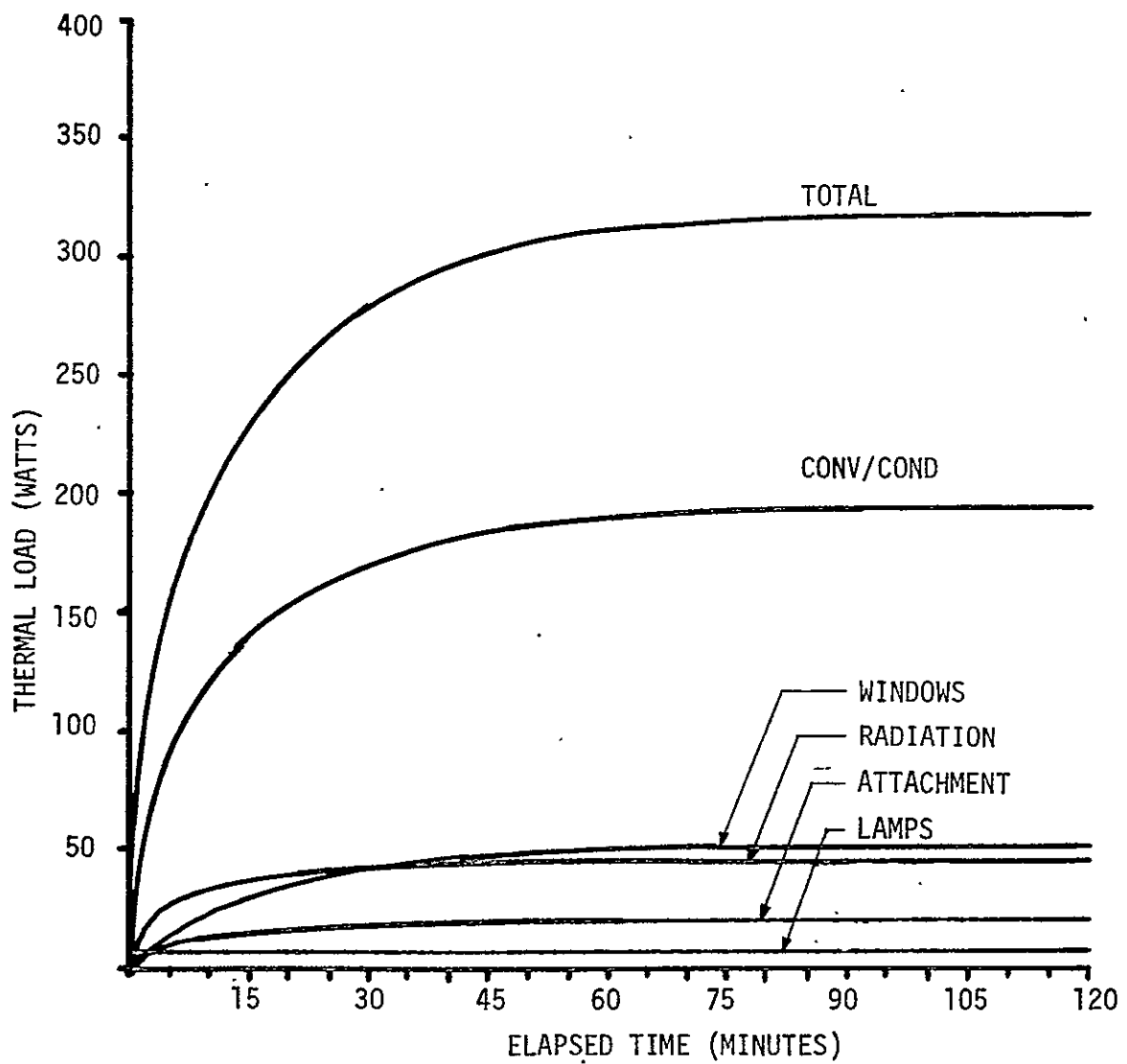


FIGURE 3.3.2 Transient Thermal Loads on Chamber

4. PHASE 2 - PRE-COOLER PLUS MODULES

The concept of pre-cooling the liquid coolant in the cloud chamber heat sinks was studied. The advantage of this approach was that the heat sink temperature could be maintained closer to the desired temperature in the cloud chamber using the thermoelectrics on the cloud chamber as trimmer coolers rather than primary coolers. This would result in much closer temperature control of the cloud chamber.

The thermal model is shown in Figure 4.1. In order to eliminate as many variables as possible the space lab flow rate-specific heat product was set to 200 BTU/hr°F by MSFC. The effect of chamber inlet temperature on the design of the pre-cooler and chamber thermoelectrics is shown in Figure 4.2. It is clear from this figure that the optimum chamber heat sink inlet temperature was approximately 20°F. Therefore, all subsequent calculations were made using this value.

Calculations were made on the effect of heat sink effectiveness on the system design. It was clear from those results that neither the chamber heat sink nor the pre-cooler effectiveness had a very dramatic effect on the input required. Therefore, the most reasonable combination of these values was chosen for subsequent calculations, namely 0.80 for both heat sinks.

The predicted performance of the pre-cooler/chamber system is shown in Figure 4.3. Since the size and weight of the pre-cooler were unknown, several cases were generated to illustrate the effect of the

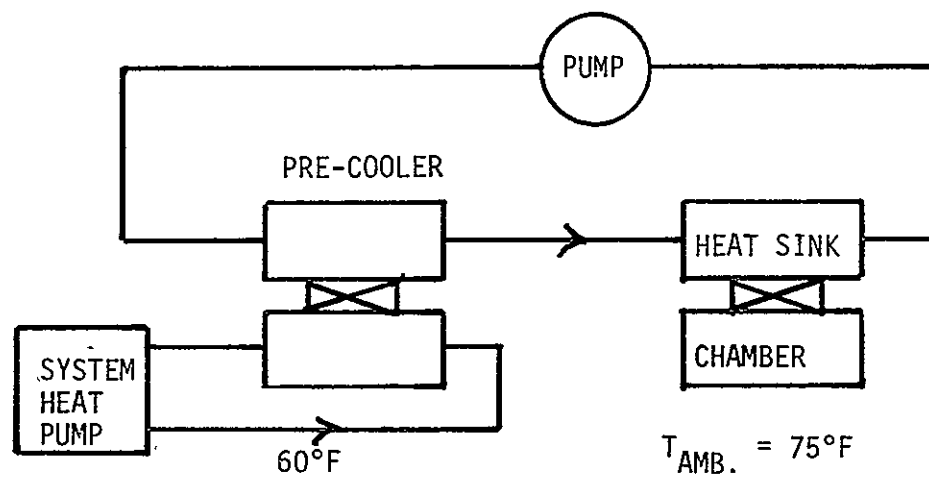


FIGURE 4.1 Model for Pre-Cooler/Chamber System

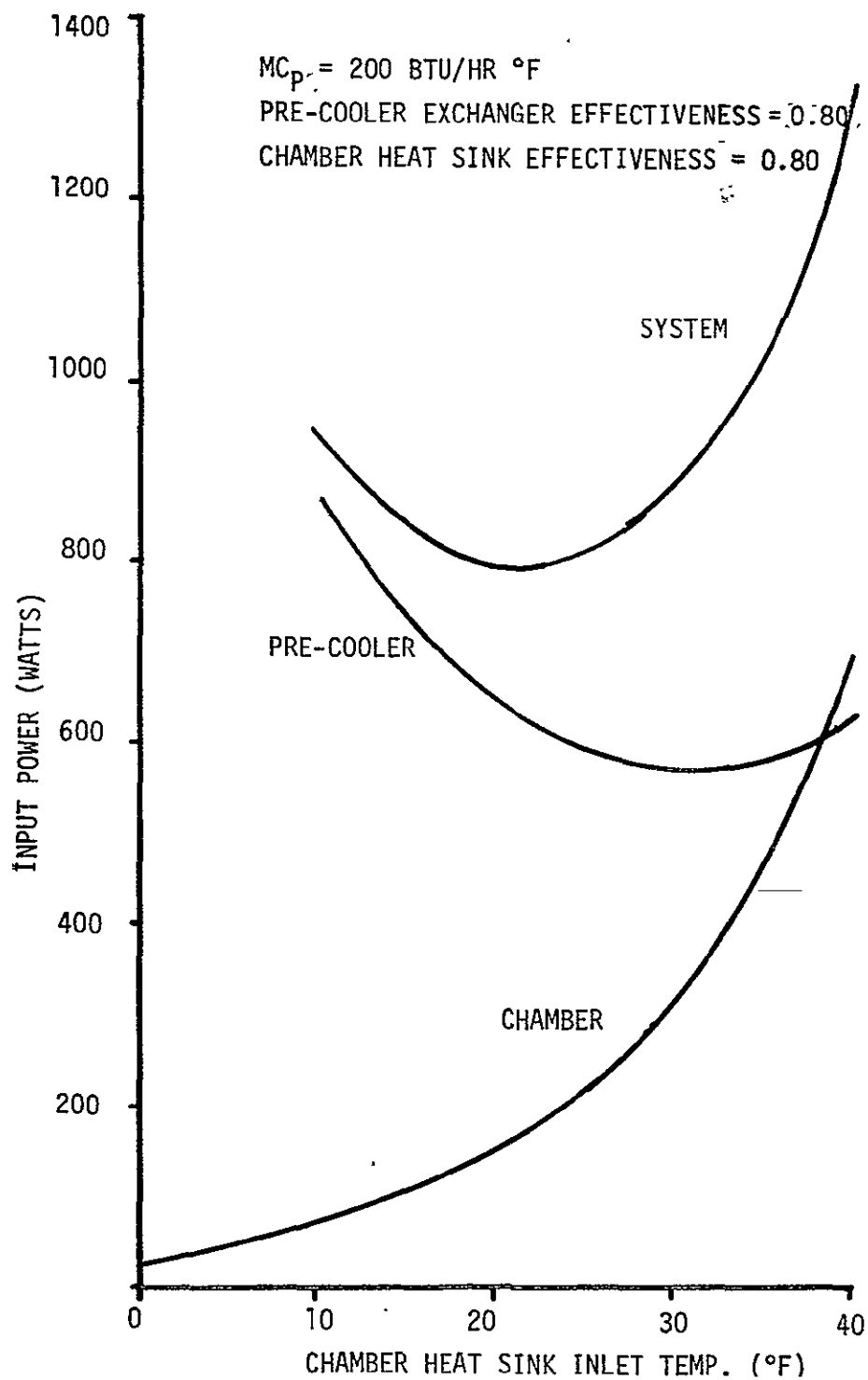


FIGURE 4.2 Effect of Chamber Inlet Temperature

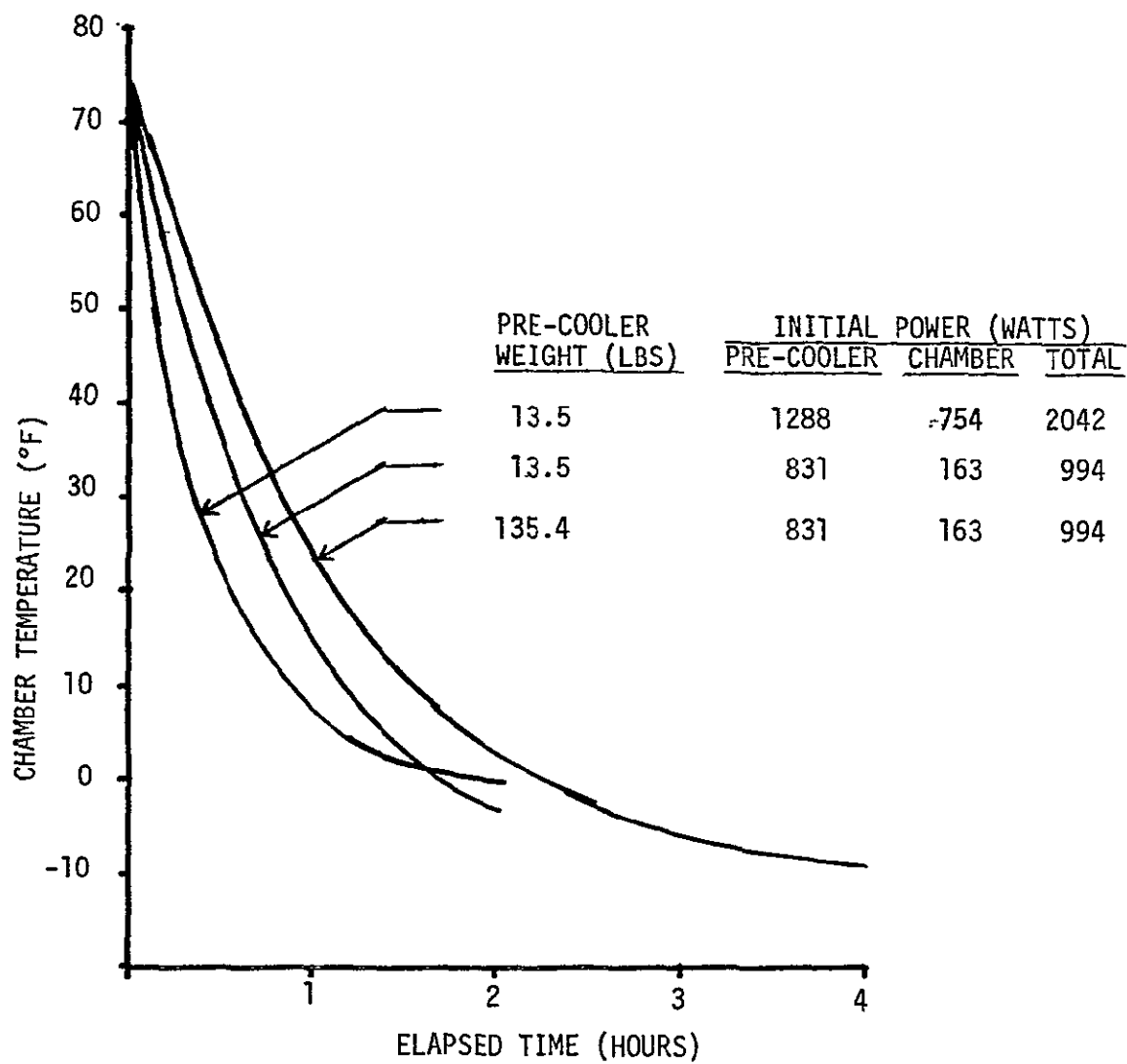


FIGURE 4.3 Pre-Cooler System Transient Performance

pre-cooler weight. Additionally, the input power to the pre-cooler as well as the chamber was increased to the largest acceptable values to yield the fastest cool down times achievable with this systems approach.

Since the cool down time was approximately 2-4 hours, an alternate design approach was considered. That is, the limiting factor in cool down time was the large thermal mass of the pre-cooler water loop. In the alternate design, the pre-cooler is completely eliminated and a staged thermoelectric cooler is used between the chamber heat sink and the chamber wall. This 3rd phase of activity is described in the next section.

5. PHASE 3 - CASCADES

5.1 Design and Performance Calculations

The configuration studied in this phase of the program was identical to that described in Phase 1 with one exception: The thermoelectric module as shown was replaced with a multi-stage thermoelectric cascade. Results of the effect of thermoelectric cascading on steady-state performance are shown in Figure 5.1.1. It is observed that the two-stage case required approximately the same input power as the optimized pre-cooler/chamber system described in Phase 2. Indeed, thermoelectrically they are nearly equivalent.

Transient cooling characteristics of an optimized 2 and 3-stage cooler are shown in Figures 5.1.2 and 5.1.3, respectively. It is observed that, in spite of the higher efficiency of the three-stage cooler, the transient performance in the region of interest is practically identical. This is due to the added mass and distributed heat pumping of the 3-stage unit versus the more compact 2-stage unit. Consequently, subsequent efforts concentrated on the two-stage design.

Further calculations showed that a slight improvement in the initial cooling transient was evidenced by increasing the thermal mass of the heat sink. Of course, a more dramatic effect could be obtained by decreasing the thermal mass of the chamber. The calculations for a chamber with 1/2 the mass of the original design (wall thickness approximately 0.31" versus 0.625") are shown in Figure 5.1.4 for a two-stage cascade. This represented the best performance that could

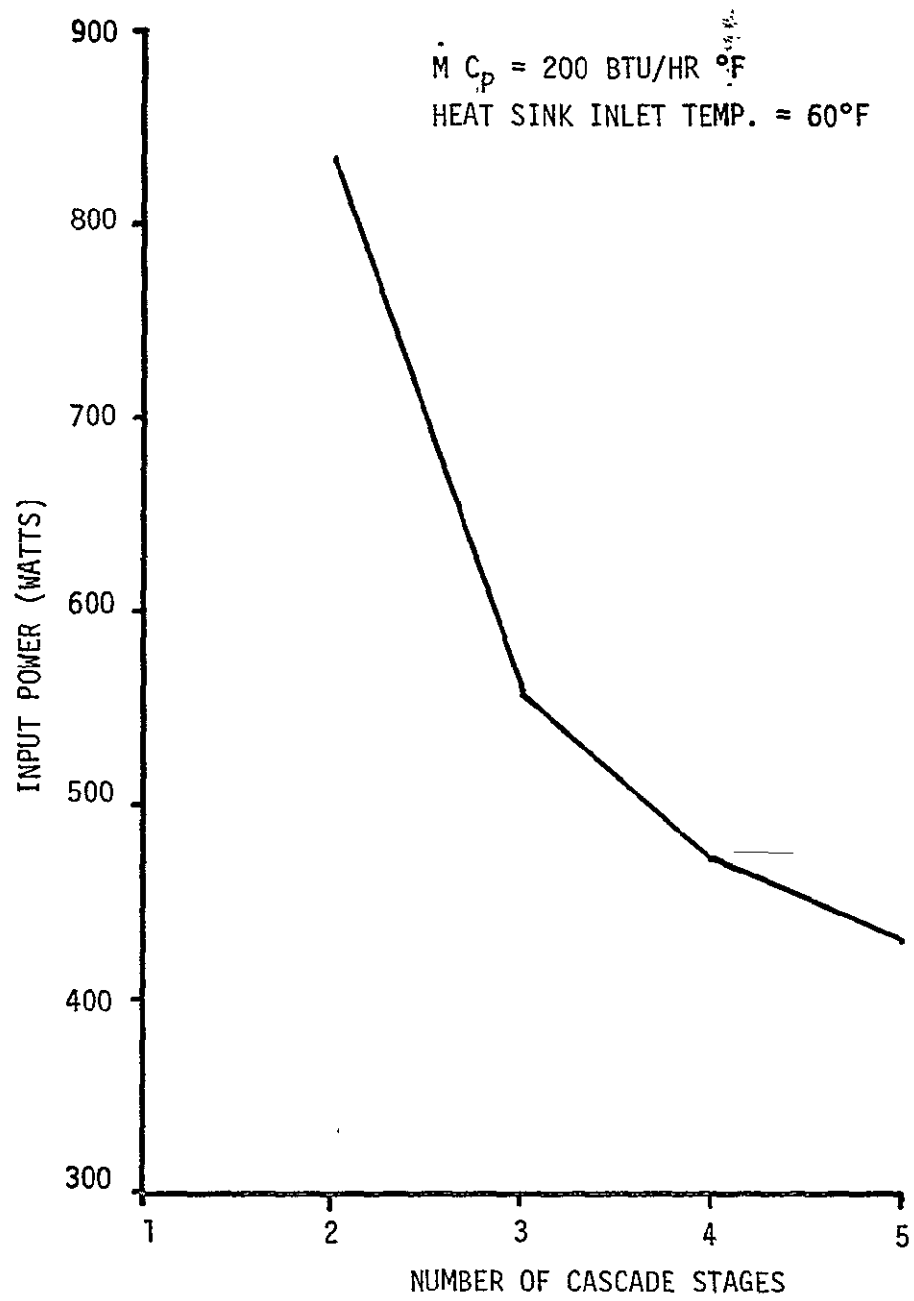


FIGURE 5.1.1 Effect of Thermoelectric Cascading
(No-Pre-Cooler)

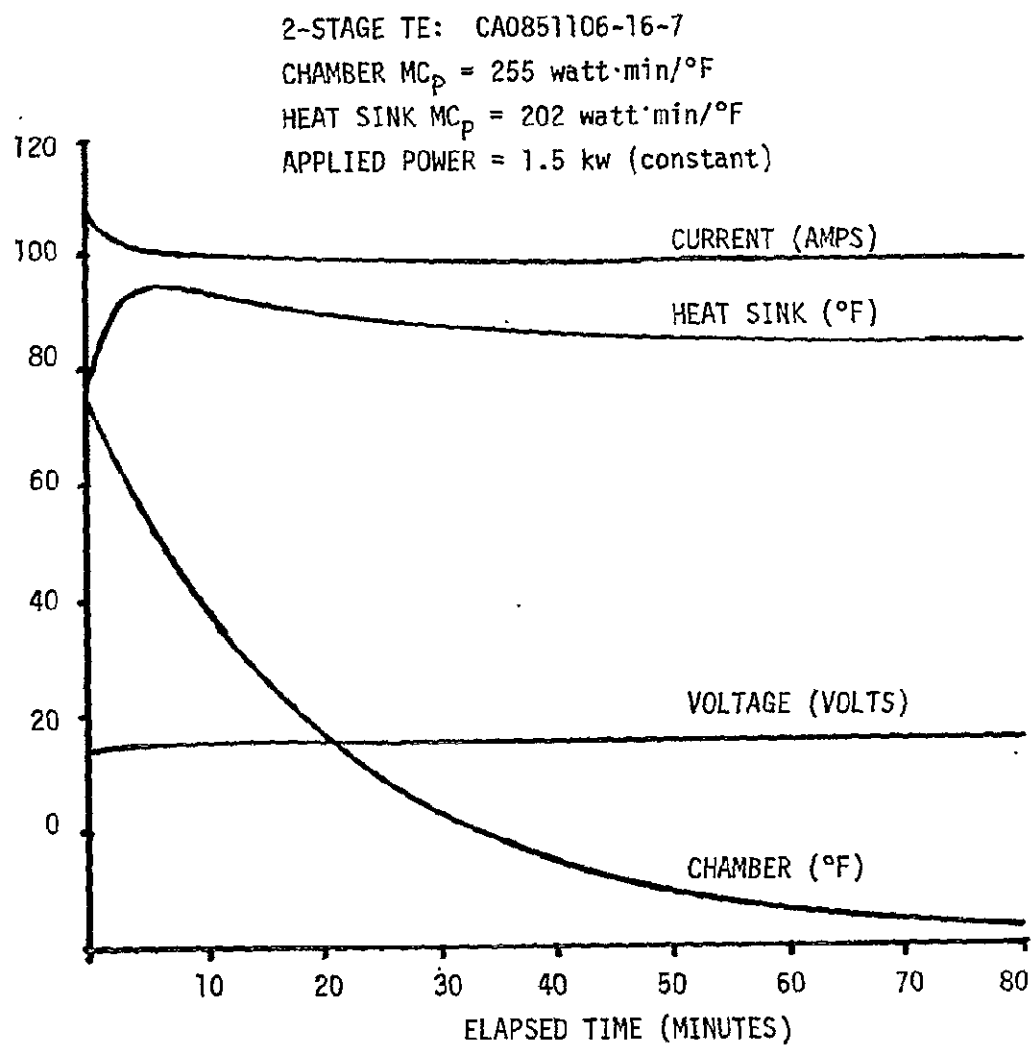


FIGURE 5.1.2 Transient Performance -
Two Stage Cascade

3-STAGE TE: CA0550514-34-16-11
CHAMBER $MC_p = 255 \text{ watt}\cdot\text{min}/^\circ\text{F}$
HEAT SINK $MC_p = 202 \text{ watt}\cdot\text{min}/^\circ\text{F}$
APPLIED POWER = 15 kw (constant)

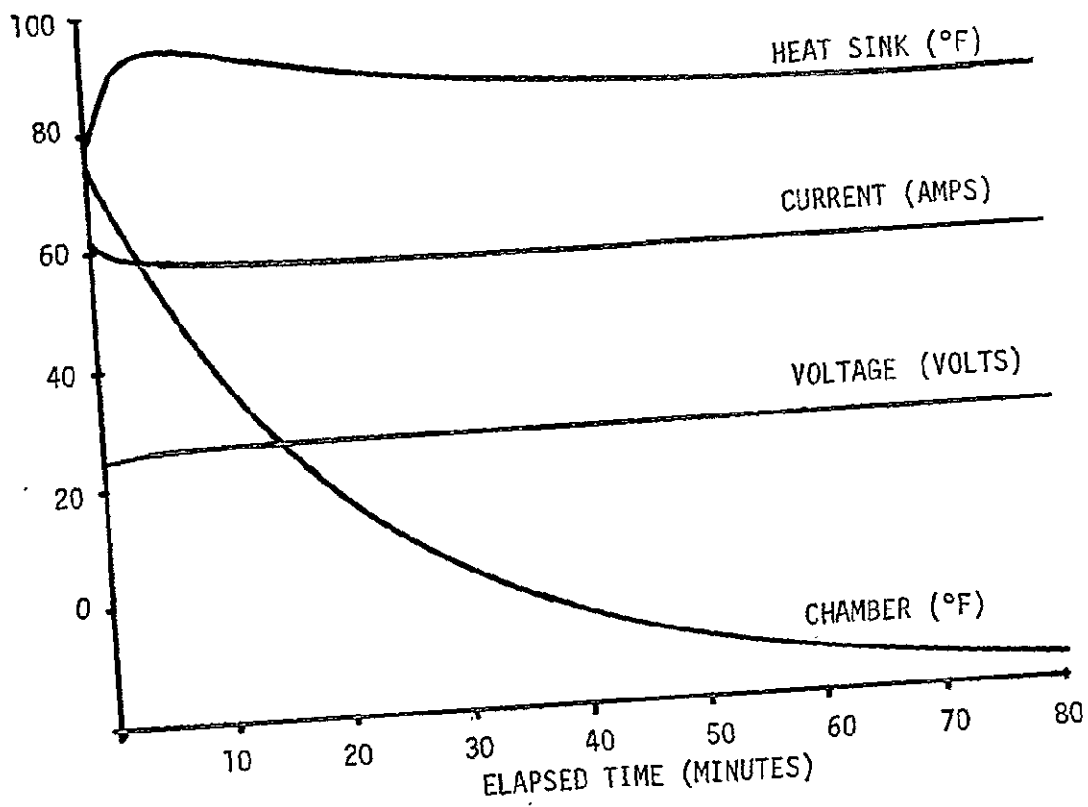


FIGURE 5.1.3 Transient Performance -
Three Stage Cascade

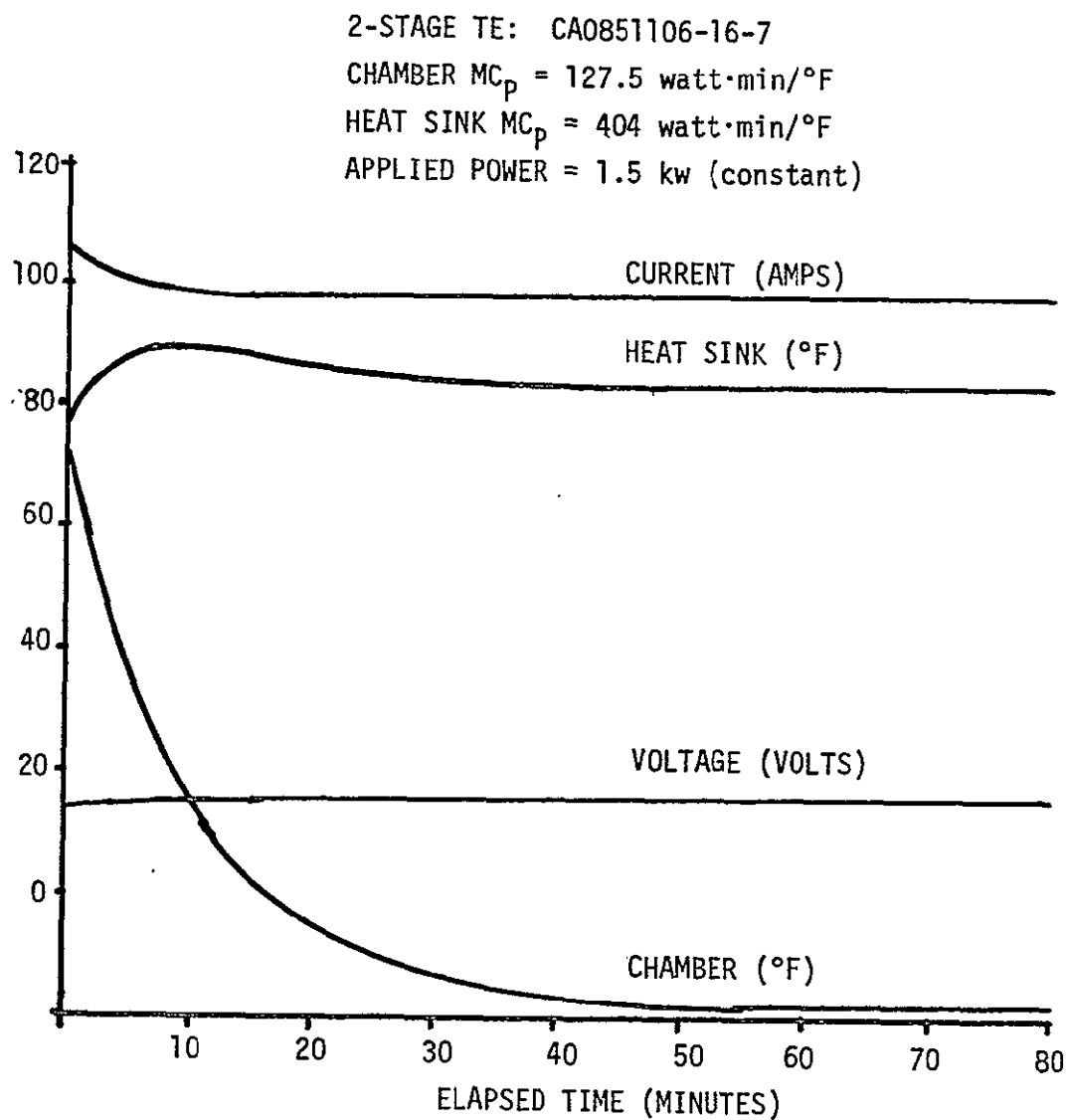


FIGURE 5.1.4 Transient Performance - Two Stage Cascade
With Reduced Chamber Mass

be expected under the imposed constraints. Any further reduction in wall thickness decreases the "thermal filtering" and jeopardizes the ability to achieve desired isothermal conditions.

5.2 Hardware Delivery

In anticipation of utilizing similar coolers in the final design, MSFC placed an order for 40 two-stage cascades, Model #952-16-7, for use in a test designed by MSFC personnel. These coolers were different from those in Figures 5.1.2 and 5.1.4 in thermoelectric pellet dimensions. That is, the 952-16-7 cooler is fabricated from pellets 0.055" x 0.055" in cross-section by 0.063" high; whereas the CA0851106-16-7 cooler is fabricated from pellets 0.085" x 0.085" in cross-section by 0.1106" high.

5.3 Transient Specifications

It was at this point in the program that BWTE was first made aware of certain transient cooling specifications required for the proper execution of the cloud physics experiments. These desired cooling specifications are shown in Figure 5.3.1. This data was entered into the computer program and a simple thermostatic approach was applied. That is, the cooler power was full on if the chamber temperature was above the curve and completely off otherwise. The results of these calculations are shown in Figures 5.3.2 and 5.3.3 for the two respective cool down specifications. (The observed oscillations were arti-

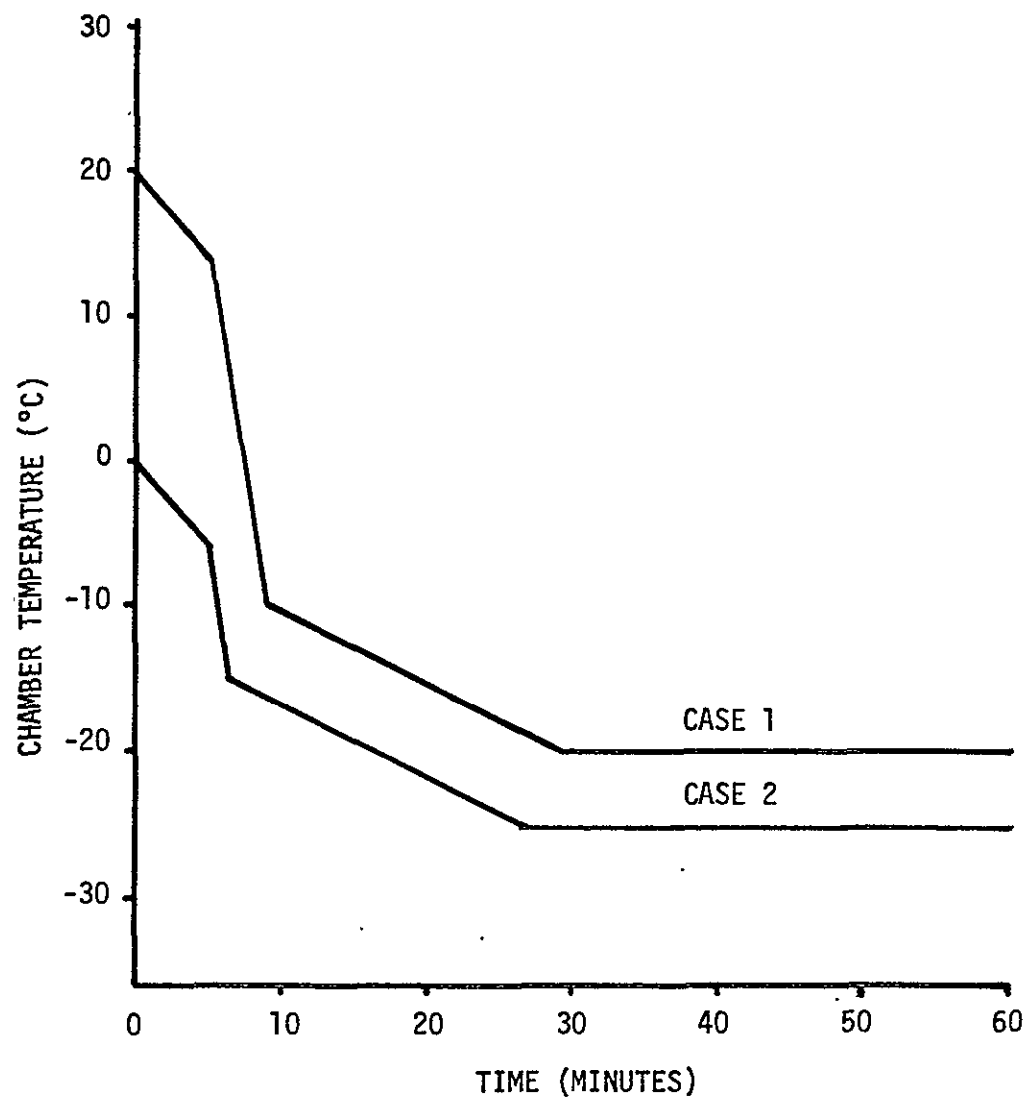


FIGURE 5.3.1 Transient Cooling Specifications

Thermostatically Applied Power = 1500 Watts_i
 Chamber MCp = 127.5 Watt·Min/°F
 Heat Sink MCp = 404.0 Watt·Min/°F
 Flow Rate MCp = 300 BTU/HR °F at 60°F

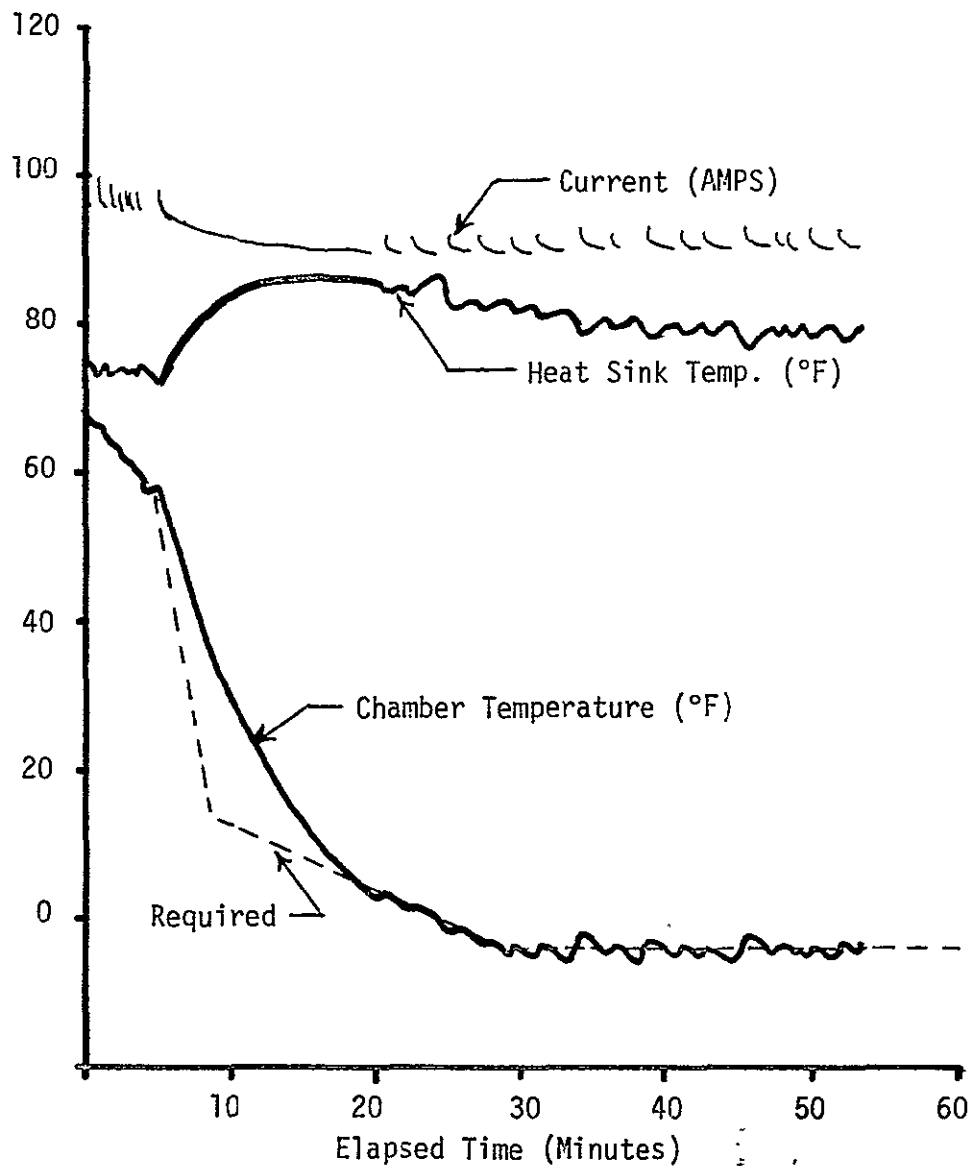


FIGURE 5.3.2 Transient Performance of Thermoelectric
 Cloud Chamber, Case I

Thermostatically Applied Power = 1500 Watts
Chamber MCp = 127.5 Watt·Min/°F
Heat Sink MCp = 404.0 Watt·Min/°F
Flow Rate MCp = 300 BTU/HR °F at 60°F

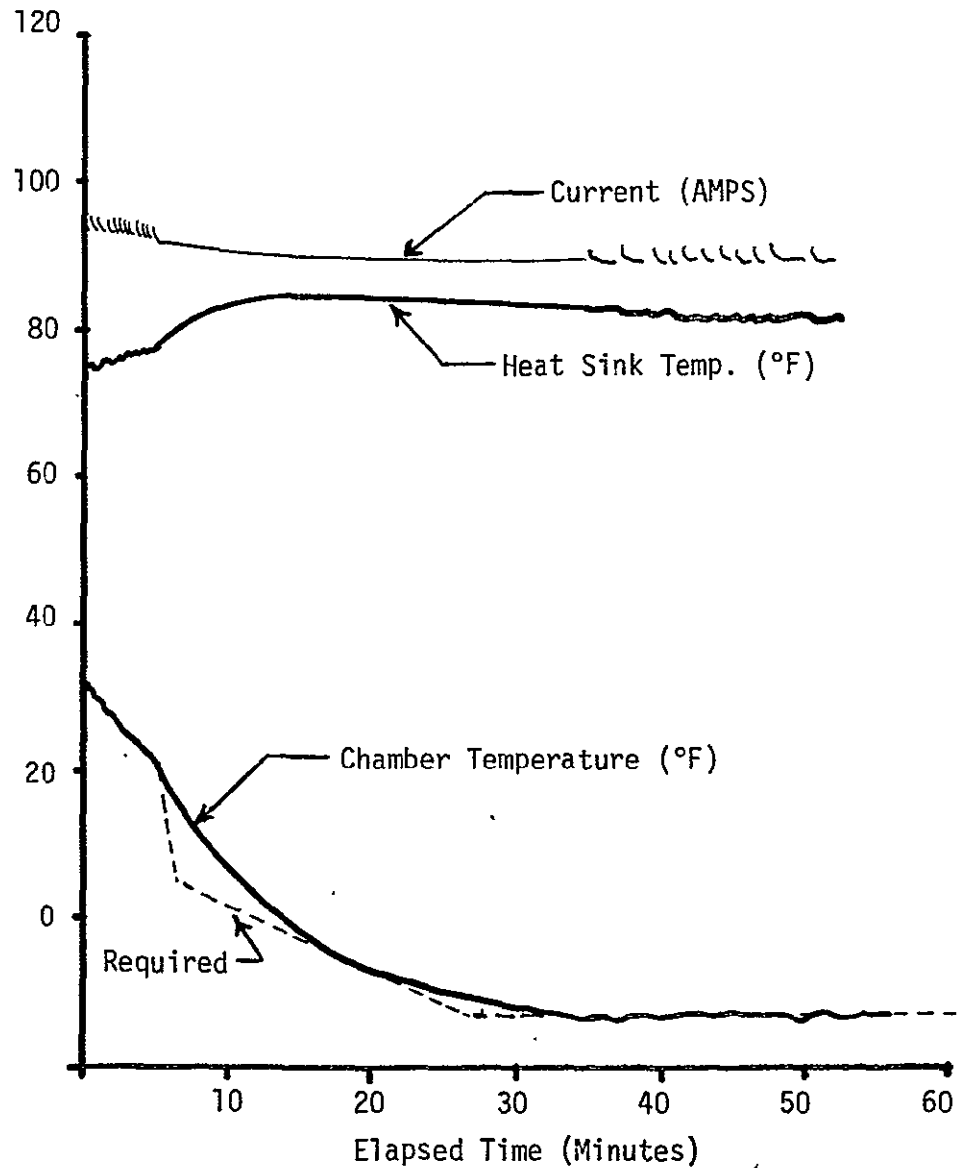


FIGURE 5.3.3 Transient Performance of Thermoelectric Cloud Chamber, Case 2

facts of the program due to arbitrary time interval selections.)

Every attempt was made to maximize performance: The full power budget of 1.5 kw, the maximum heat sink flow rate of 300 BTU/hr°F, the minimum heat sink flow temperature of 60°F, a "double-sized" heat sink mass, and a "minimum" chamber wall thickness of 0.31 inches. Nevertheless, the steeper portions of the spec. curves could not be met in either transient case.

Furthermore, fundamental calculations based solely on the required cool down rate and the chamber mass revealed that 2960 watts of heat pumping was required down to 14°F chamber temperature. Consequently, the required Coefficient of Performance $COP = \frac{\text{heat pumped}}{\text{input power}}$, was approximately 2.0. This value was totally unattainable by any thermoelectric device operating over the required 66°F temperature differential.

It was concluded from the above calculations that thermoelectrics could not provide the degree of cooling on a real time basis to meet the desired transient condition given in Figure 5.3.1. Although this transient was not a contractual requirement on the part of BWTE, MSFC decided to seek yet another approach. This involved a totally different method of cooling the chamber and a redefinition of the scope of effort required by BWTE. This fourth and final phase of this contract is discussed in the following section.

6. PHASE 4 - RESERVOIR CHILLER

This final phase of the contract required a contract modification to formally redefine the scope of effort. In general, the revised statement of work was concerned with the thermoelectric cooling of a liquid reservoir which was to be used, in turn, to cool the chamber. The cool down of the chamber itself was no longer the responsibility of BWTE.

6.1 Design

The conceptual schematic of the expansion chamber thermal control system is shown in Figure 6.1.1. The thermoelectric chiller on which BWTE was to concentrate its activities is shown in more detail in Figure 6.1.2. Due to the constraints of funding and delivery schedule, it was mutually decided that BWTE utilize approximately 1100 watts input power and simply design the chiller to extract the maximum heat with the cold side at -30°C and the hot side at $+30^{\circ}\text{C}$.

6.1.1 Thermoelectric - A two-stage design was chosen for this application. The optimum couple ratio from the bottom stage to the top stage was 2 to 3 depending on the final operating temperatures of the respective sinks. The best mechanical versus thermal trade-off in design was to use a standard 12 x 12 T.E. pellet array as the bottom and a standard 8 x 8 T.E. pellet array as the top stage. These stages had different heights but identical width envelope

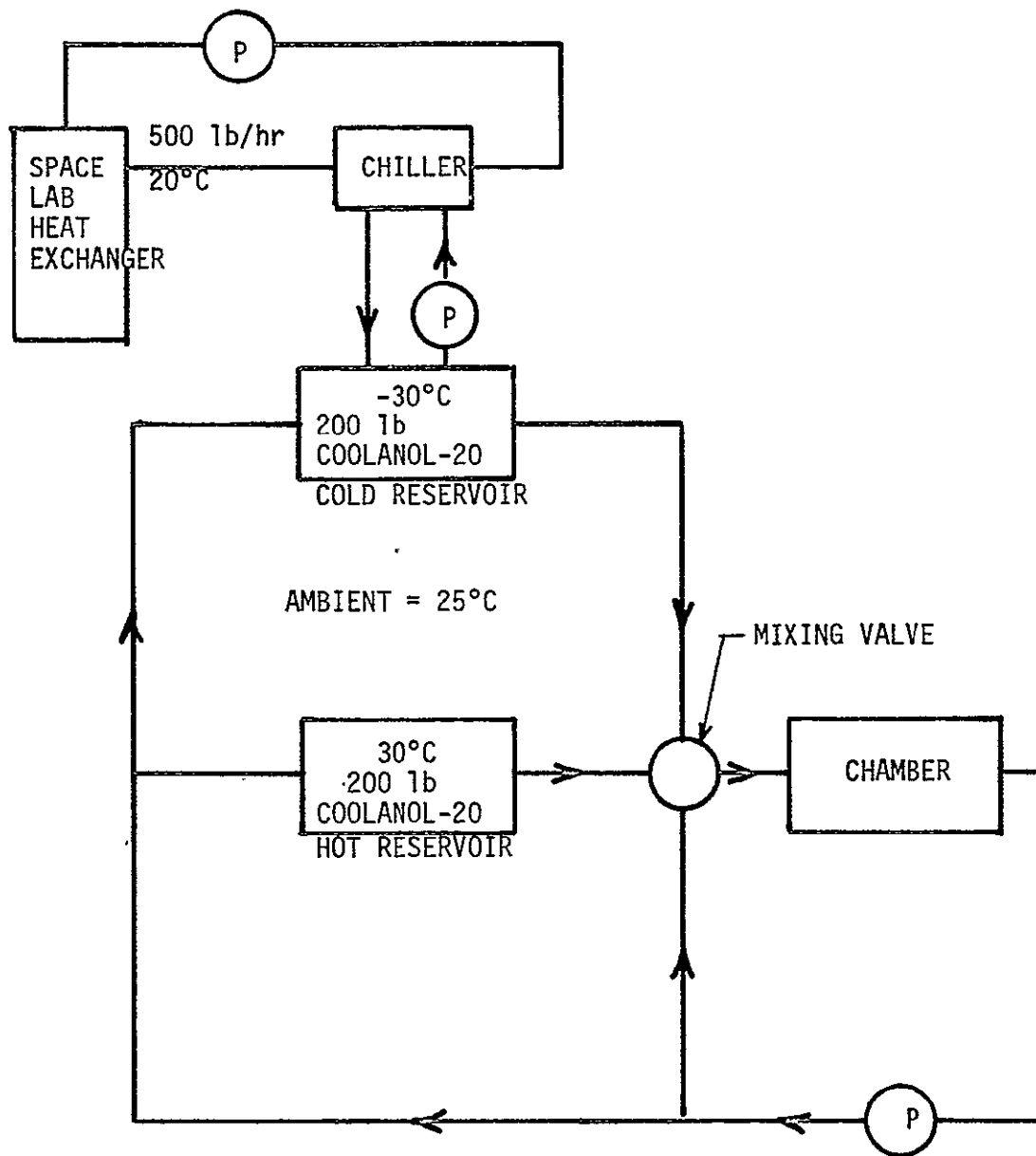


FIGURE 6.1.1 Conceptual Schematic of Chamber Thermal Control System

dimensions so that the assembled cascade had the overall appearance of a single parallelepiped module as opposed to the normal pyramid structure. The resulting couple ratio was 70/31 or 2.26 in good agreement with optimization requirements.

The resulting optimum input power for each thermoelectric cascade was 5.75 volts at 3.0 amps. Since the maximum allowed system voltage was 22 volts, three cascades in series were selected yielding a system voltage of 17.3 volts. An even number of these 3 cascade sets was required due to the symmetry of the design (Figure 6.1.1.1). Consequently, 18 sets were chosen yielding an input power requirement of 17.3 volts at 54.0 amps. The calculated heat pumping at the design point was 2.5 watts per cascade or 135 watts for the system. In the turn-on condition when the entire system was at ambient, the heat extraction rate was about 500 watts. The net heat extracted from the fluid, of course, was slightly lower than these values due to thermal losses.

The electrical resistance was measured on each cascade manufactured. The observed range was 1.58 to 1.70 ohms. The two resistance extremes were chosen, in addition to two other cascades selected at random, for zero load ΔT measurements. The results are given in Table 6.1.1.1:

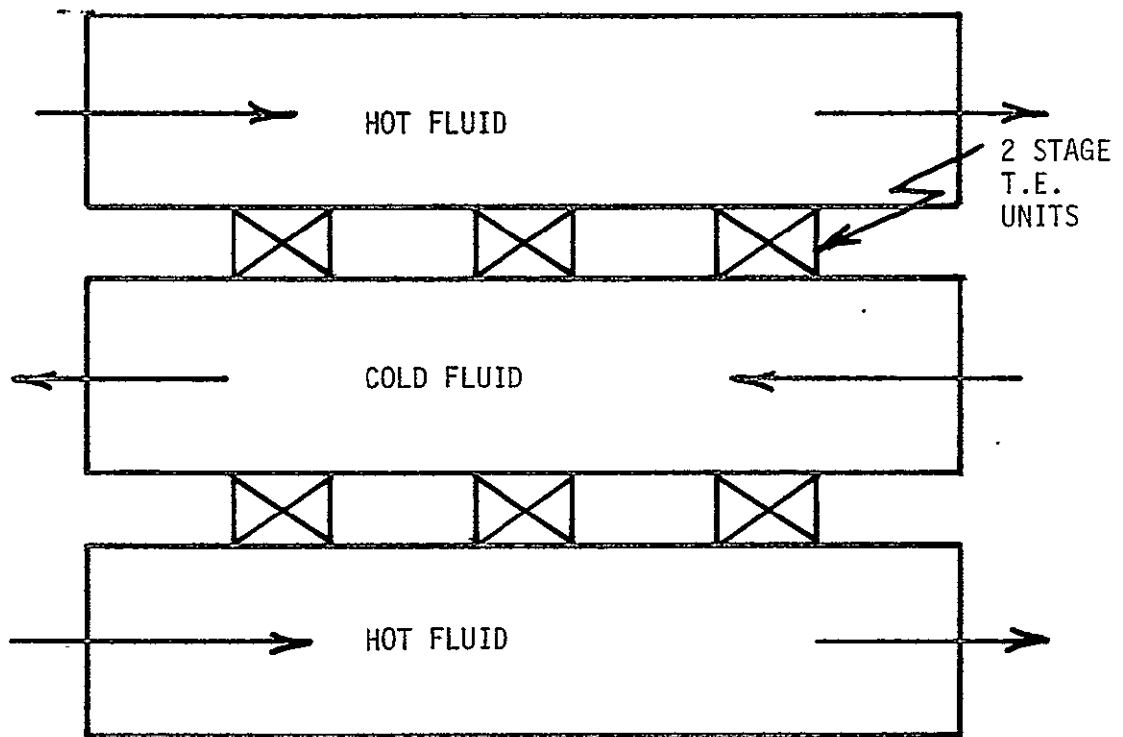


FIGURE 6.1.1.1 Concept of Fluid Chilling Device

Table 6.1.1.1
CASCADE TEST RESULTS, HEAT SINK = 35°C

<u>Serial No.</u>	<u>I(amps)</u>	<u>V(volts)</u>	<u>$\Delta T(^{\circ}\text{C})$</u>	<u>Q(watts)</u>	<u>Res. (Ω)</u>
8	4.08	8.37	90.5	0.0	1.70
17	4.10	7.60	87.0	0.0	1.58
25	4.02	7.95	90.5	0.0	1.65
39	4.07	7.90	91.7	0.0	1.62
Predicted	4.05	7.50	89.0	0.0	1.58
Predicted Opt.	3.03	5.75	65.0	2.5	1.58

The observed performance variance within this set of cascades was due most likely to thermoelectric material parameter variance. However, none were rejected since the average of the coolers tested exceeded the expected cooling performance.

6.1.2 Heat Sinks - The envelope dimensions of the heat sinks were designed to be consistent with the layouts of the thermoelectric designs. Finned passages were used such that the maximum expected ΔT between the sinks and the outlet fluid was 5.8°C and 1.2°C for the hot and cold sinks, respectively.

6.2 Assembly

A complete set of drawings was generated governing the fabrication of the thermoelectric chiller. The main assembly drawing was given

the BWTE number: 2600-0018, dated 7/2/76. The various stages of assembly are depicted in Figures 6.2.1 through 6.2.4.

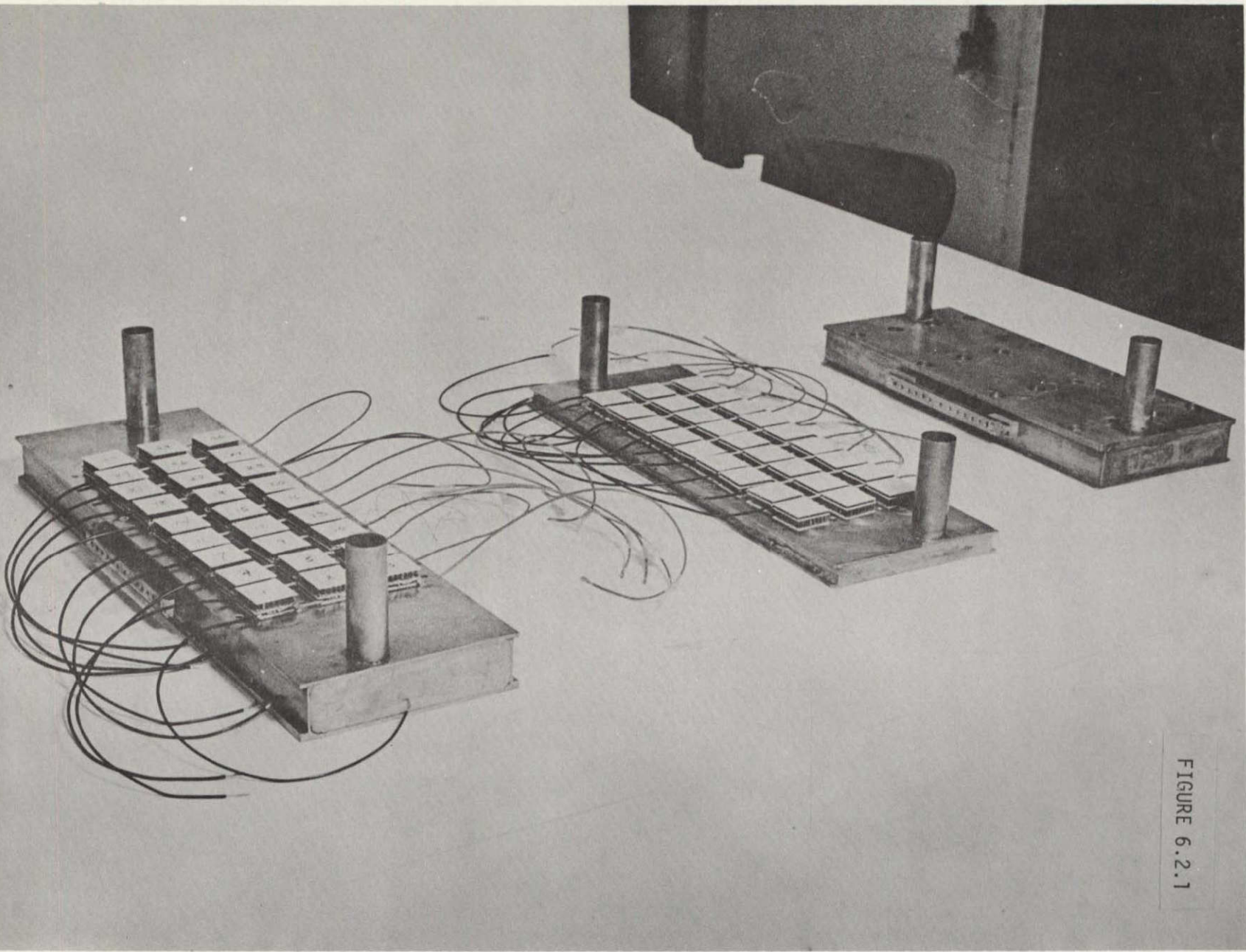
6.3 Acceptance Test

An acceptance test was designed as shown schematically in Figure 6.3.1. The 12" x 18" x 18" reservoir was insulated with 4" of polyurethane. The cold fluid piping was insulated with 2" (on radius) of Armstrong insulation with $K = 0.0208 \text{ BTU/hr ft } ^\circ\text{F}$. The total length of the piping was 5.2 ft. Figures 6.3.2 and 6.3.3 are photographs of the system assembled as ready for test.

A zero load ΔT test was made prior to the system simulation test. This test differed from the system test in that no cold fluid was allowed into the chiller. This test, of course, had a much faster transient and allowed a determination of the quality of the thermoelectric chiller apart from the system parameters. Detailed test measurement data sheets from both tests are given in Tables 6.3.1 and 6.3.2.

6.4 Analysis - The calculated heat loss conductance of the cold system was 0.4859 watts/ $^\circ\text{C}$ with 0.2536 watts/ $^\circ\text{C}$ due to the reservoir and 0.2323 watts/ $^\circ\text{C}$ due to the piping. The thermal capacitance of the cold system minus the chiller was 2560 watt min/ $^\circ\text{C}$. The heat sinks plus the water contained in them was 242.7 watt min/ $^\circ\text{C}$. The cold sink alone (zero load test) was 33.8 watt min/ $^\circ\text{C}$ and filled with the ethylene glycol-water mixture was 49.9 watt min/ $^\circ\text{C}$.

FIGURE 6.2.1



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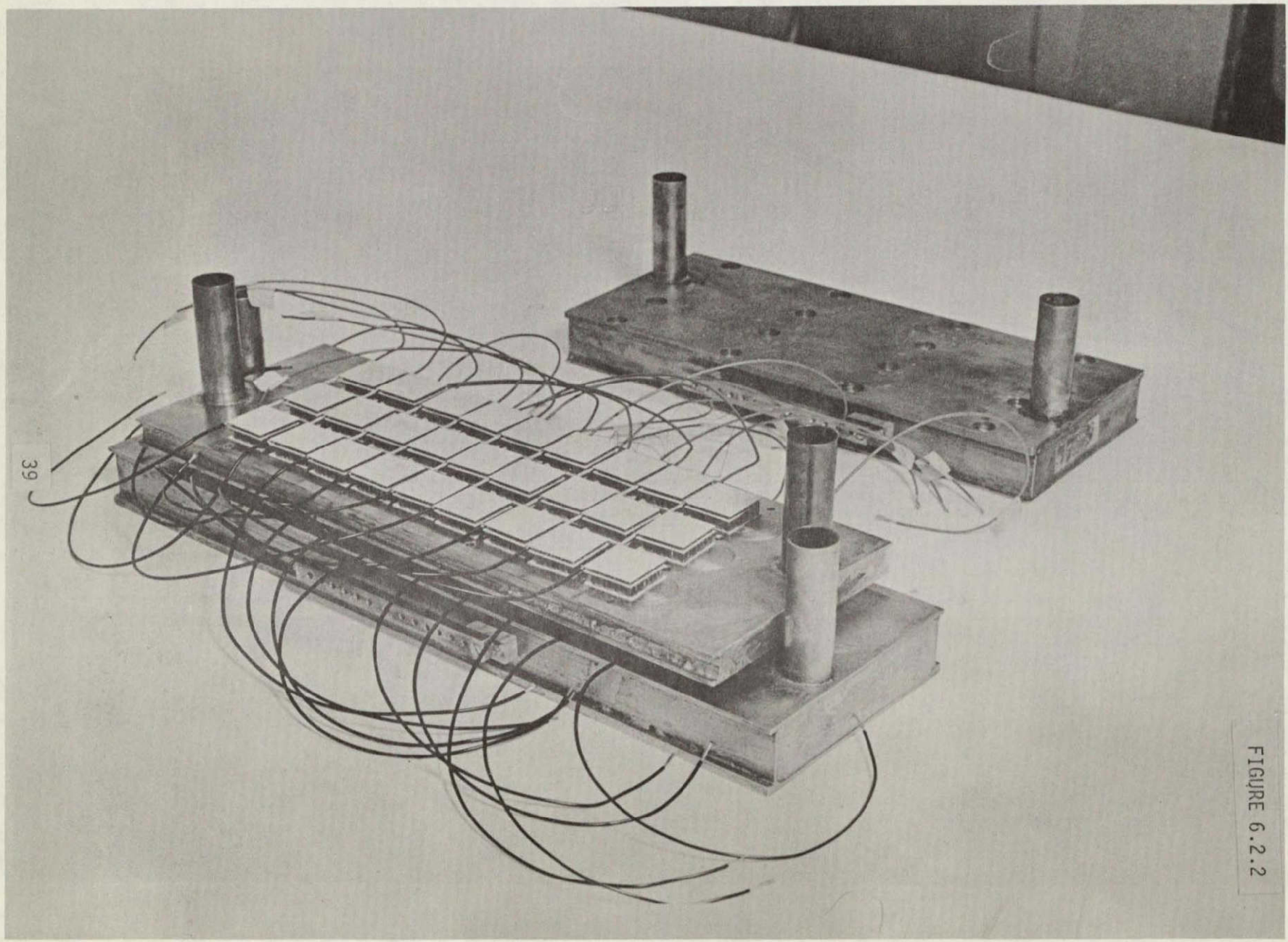


FIGURE 6.2.2

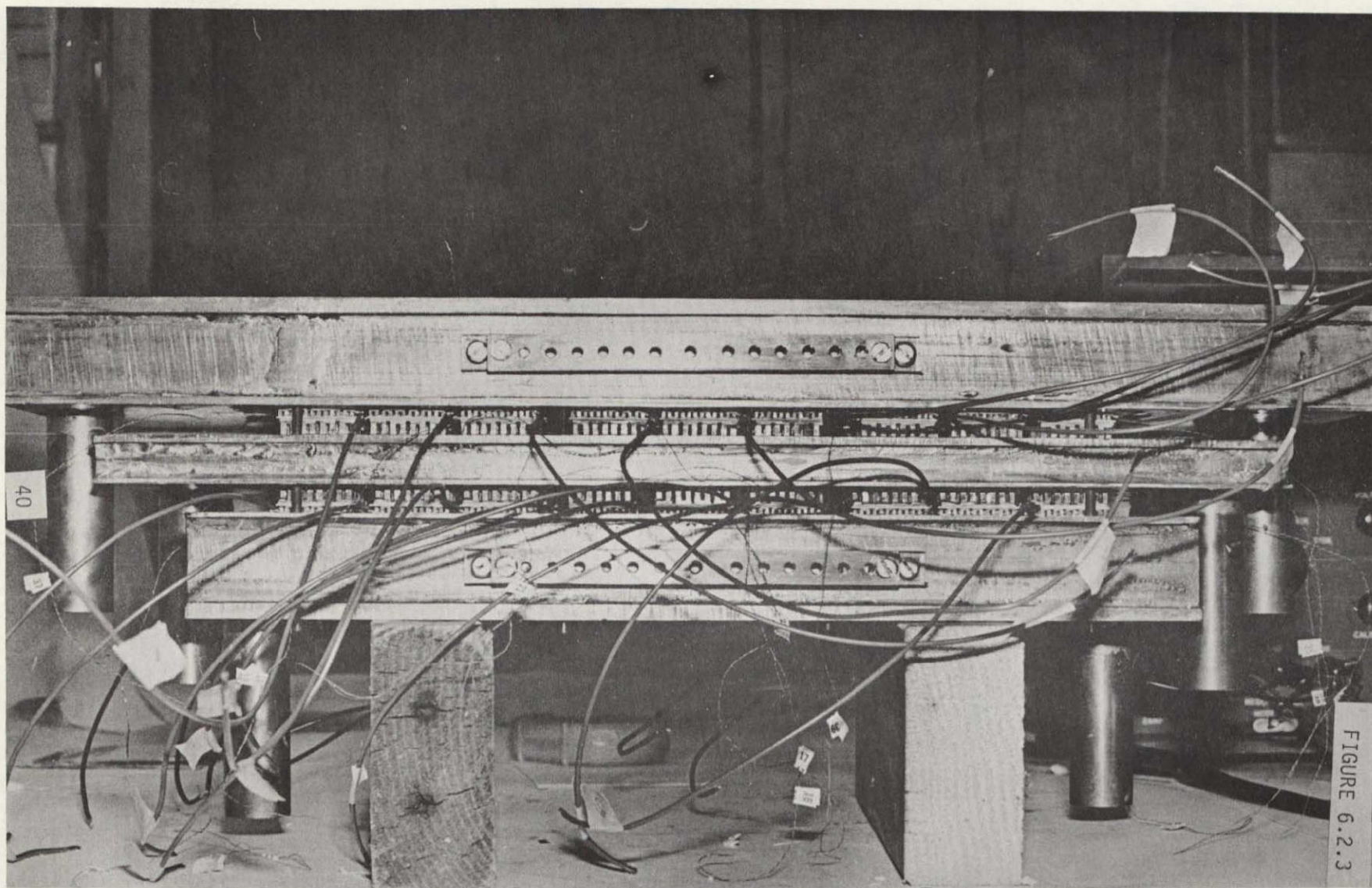
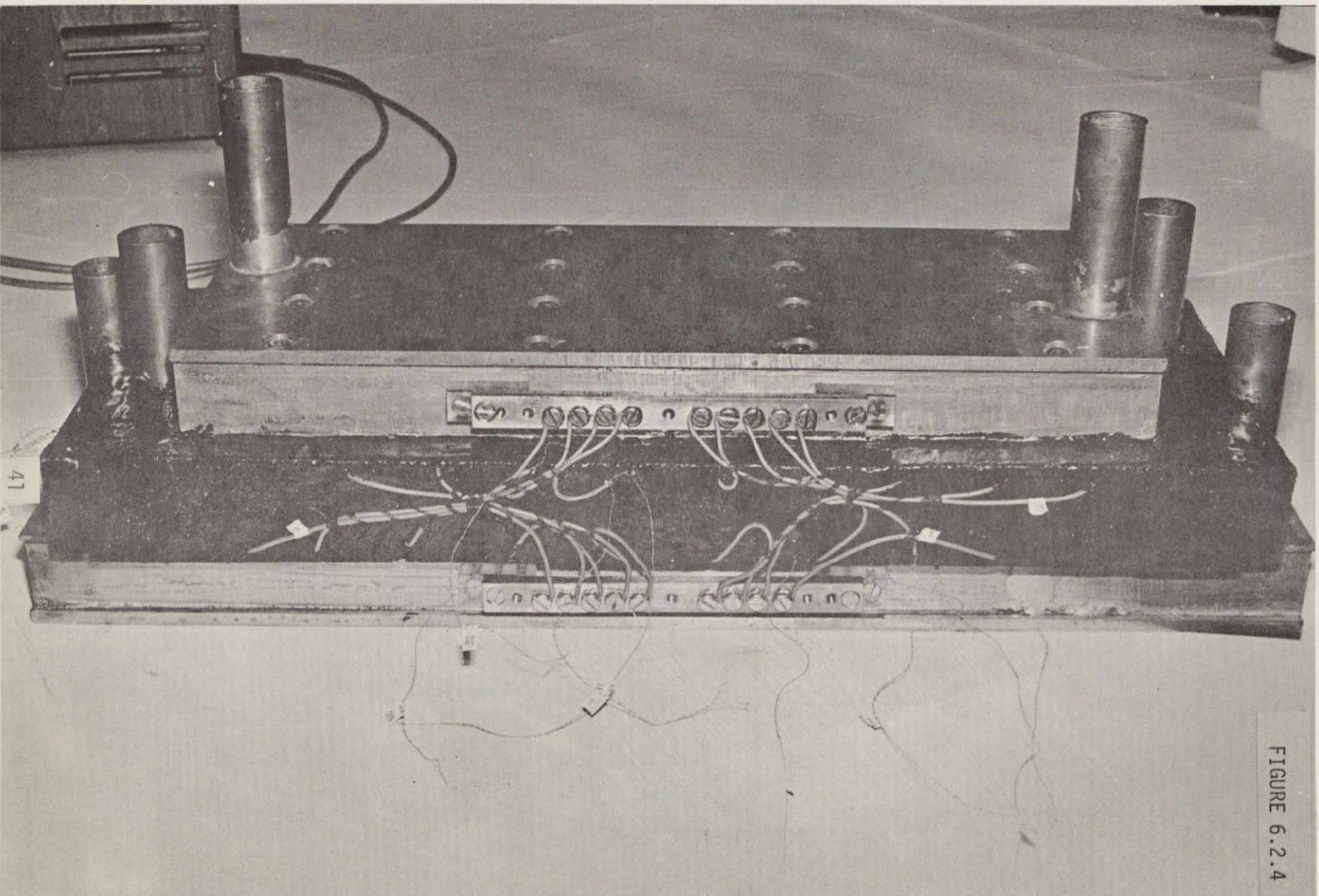


FIGURE 6.2.3

FIGURE 6.2.4



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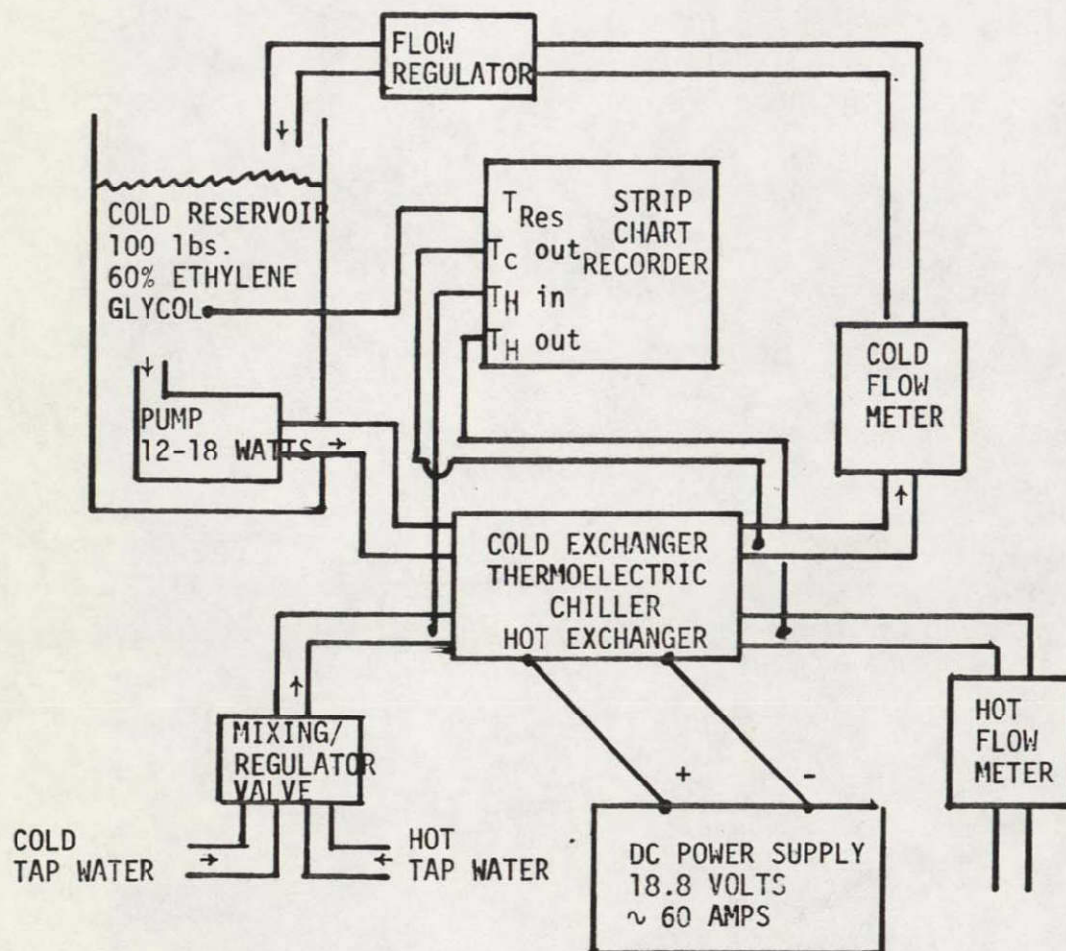
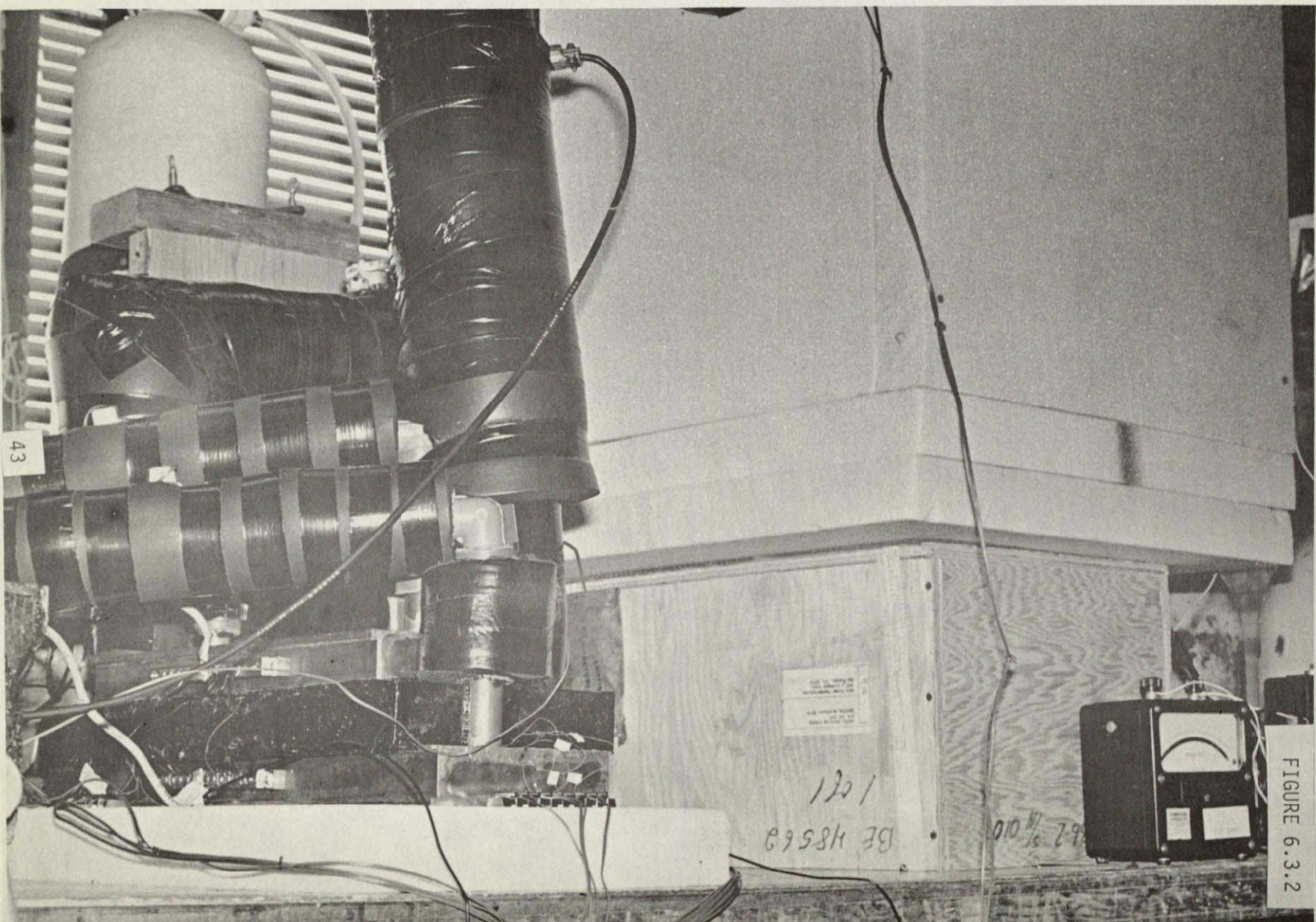


FIGURE 6.3 SCHEMATIC OF APPARATUS FOR ACCEPTANCE TEST



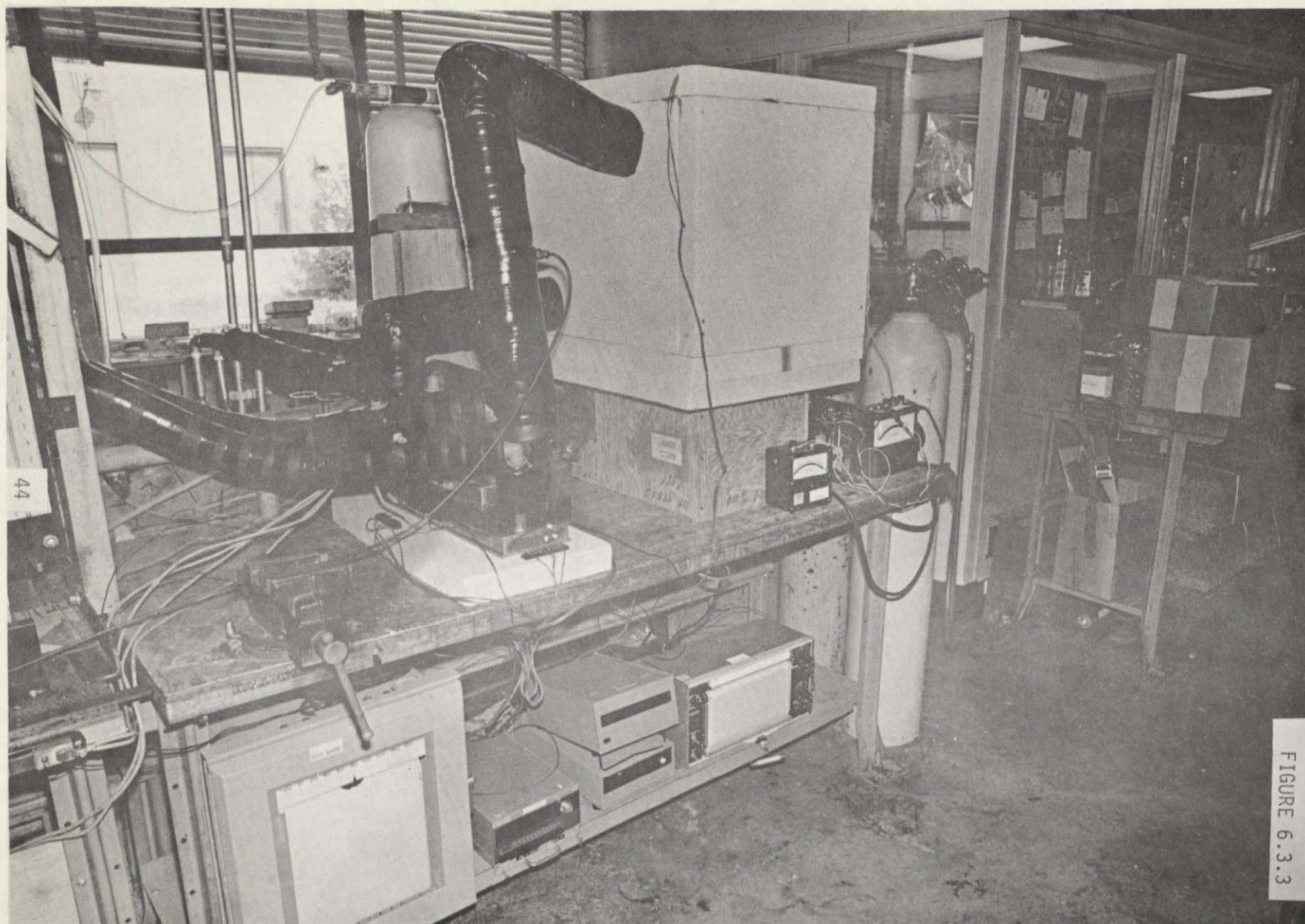


FIGURE 6.3.3

TABLE 6.3.1

NO LOAD ΔT MEASUREMENT: MARSHALL CHILLER
(1Min/Set of Readings)

Date 9/5/76

Reading	Units / Meter Position	t=0	t=5 min	t=10 min	t=15 min	t=20 min	t=25 min	t=30 min
Water Flow Upper Sink	ml/min	1890	1890	1890	1890	1890	1890	1890
Water Flow Lower Sink	ml/min	1890	1890	1890	1890	1890	1890	1890
Applied Voltage	VDC	0	19.2	19.2	19.0	19.0	19.0	19.0
Applied Current	AMP	0	62	61	61	60.8	60.8	60.8
T _H Cascade #42	°C / A1	18.8	26.3	25.3	24.8	24.6	24.5	24.5
T _H Stage #1 Cascade #42	°C / A2	20.9	25.1	26.2	26.6	26.7	26.7	26.7
T _C Cascade #42	°C / A3	19.3	-27.5	-42.7	-47.4	-49.3	-50.0	-50.2
T-Upper Sink Inlet End	°C / A4	18.9	19.4	19.0	+19.0	19.0	18.8	19.1
T-Upper Sink Middle	°C / A5	18.8	23.9	22.9	22.0	21.8	22.5	21.6
T-Cold Exch. Upper Plate Inlet End	°C / A6	19.2	-22.6	-38.7	-44.7	-46.9	-47.7	-48.0
T-Cold Exch. Upper Plate Middle	°C / A7	19.2	-28.8	-43.2	-48.4	-50.3	-51.0	-51.2
T-Cold Exch. Upper Plate Outlet End	°C / A8	19.3	-25.3	-40.9	-46.4	-48.5	-49.3	-49.5

Table 6.3.1 (Con't)
 NO LOAD ΔT MEASUREMENT: MARSHALL CHILLER
 (1 Min/Set of Readings)

Reading	Units Meter Position	t=0	t=5 min	t=10 min	t=15 min	t=20 min	t=25 min	t=30 min
T-Cold Exch. Lower Plate Inlet End	$^{\circ}\text{C}$ A9	19.3	-25.0	-40.1	-46.1	-48.3	-49.2	-49.2
T-Lower Sink Inlet End	$^{\circ}\text{C}$ A10	18.9	21.4	20.3	20.4	20.3	20.3	20.1
T-Stage Cascade #9	$^{\circ}\text{C}$ A11	18.9	27.8	26.6	26.2	26.1	26.0	25.9
T-Ethylene Glycol Outlet	$^{\circ}\text{C}$ A12							
T-Inlet H ₂ O Upper Sink	$^{\circ}\text{F}$ B1	64.5	64.1	64.1	63.9	63.8	63.7	63.6
T-Outlet H ₂ O Upper Sink	$^{\circ}\text{F}$ B2	64.5	73.1	72.9	71.6	71.3	71.1	71.1
T-Inlet H ₂ O Lower Sink	$^{\circ}\text{F}$ B3	64.6	64.3	64.3	64.1	64.0	64.0	63.8
T-Outlet H ₂ O Lower Sink	$^{\circ}\text{F}$ B4	64.8	73.2	72.4	71.9	71.6	71.6	71.5

TABLE 6.3.2
MARSHALL CHILLER PERFORMANCE TEST

Date 9/7/76

Reading	Units Meter Position	t=0	t=15 min	t=30 min	t= 1 hr.	t=2 hrs.	t=3 hrs.
Water Flow Upper Sink	ml/min	1890	1890	1890	1890	1890	1890
Water Flow Lower Sink	ml/min	1890	1890	1890	1890	1890	1890
Applied Voltage	VDC	0	18.5	18.4	18.5	18.3	18.5
Applied Current	AMPS	0	61.6	61.5	62.0	61.0	60.8
T _H Cascade #42	°C A1	18.3	27.2	27.2	26.8	26.1	26.0
T _H -Stage #1 Cascade #42	°C A2	19.6	25.7	26.6	27.0	26.3	26.0
T _C Cascade #42	°C A3	24.3	15.3	12.8	8.0	-0.2	-6.4
T-Upper Sink Inlet End	°C A4	18.2	19.5	19.6	19.6	19.0	19.1
T-Upper Sink Middle	°C A5	18.4	25.7	25.1	25.5	24.9	25.7
T-Cold Exch. Upper Plate Inlet End	°C A6	24.4	19.5	16.7	11.7	3.3	-3.3
T-Cold Exch. Upper Plate Middle	°C A7	24.3	14.7	12.2	7.4	-0.7	-6.9
T-Cold Exch. Upper Plate Outlet End	°C A8	24.3	13.8	11.5	6.7	-1.3	-7.4

Table 6.3.2 (Con't)
MARSHALL CHILLER PERFORMANCE TEST

Reading	Units Meter Position	t=0	t=15 min	t=30 min	t=1 hr.	t=2 hrs.	t=3 hrs.
T-Cold Exch. Lower Plate Inlet End	°C A9	24.8	20.0	17.2	12.0	3.4	-3.5
T-Lower Sink Inlet End	°C A10	18.4	23.5	23.6	23.5	22.6	22.2
T-Ethylene Glycol Inlet	°C A11	25.0	22.4	19.4	14.0	4.9	-2.3
T-Ethylene Glycol Outlet	°C A12	24.6	17.0	14.8	10.0	1.4	-5.6
T-Inlet H ₂ O Upper Sink	°F B1	63.0	62.4	62.2	62.3	61.3	62.2
T-Outlet H ₂ O Upper Sink	°F B2	63.1	73.0	72.7	72.1	71.4	71.8
T-Inlet H ₂ O Lower Sink	°F B3	63.5	63.3	63.1	62.8	62.5	62.8
T-Outlet H ₂ O Lower Sink	°F B4	64.0	74.0	73.7	73.0	72.4	72.5
Pump I/V	AMP/VOLT	1.27/12.0	1.26/11.9	1.26/11.9	1.30/12.0	1.35/12.0	1.36/1.20
f (Flow Meter)			250	262	258	250	248

Table 6.3.2 (Con't)
MARSHALL CHILLER PERFORMANCE TEST

Reading	Units	Meter Position	t=4 hrs.	t=5.2 hrs.	t=6 hrs.	t=7 hrs.	t=7 hrs. 53 min.	
Water Flow Upper Sink	ml/min		1890	1890	1890	1890		
Water Flow Lower Sink	ml/min		1890	1890	1890	1890		
Applied Voltage	VDC		18.3	18.3	18.6	18.6		
Applied Current	AMPS		59.6	60	60.0	60		
T _H Cascade #42	°C	A1	25.8	24.9	25.2	24.8		
T _H -Stage #1 Cascade #42	°C	A2	25.6	25.3	25.3	24.9		
T _C Cascade #42	°C	A3	-12.9	-18.7	-22.0	-25.3		
T-Upper Sink Inlet End	°C	A4	19.0	18.8	18.9	18.5		
T-Upper Sink Middle	°C	A5	25.1	23.9	24.2	23.2		
T-Cold Exch. Upper Plate Inlet End	°C	A6	-10.1	-16.1	-19.5	-22.9		
T-Cold Exch. Upper Plate Middle	°C	A7	-13.4	-19.2	-22.4	-25.7		
T-Cold Exch. Upper Plate Outlet End	°C	A8	-13.7	-19.4	-22.5	-25.6		

Table 6.3.2 (Con't)
MARSHALL CHILLER PERFORMANCE TEST

Reading	Units Meter Position	t=4 hrs.	t=5.2 hrs.	t=6 hrs.	t=7 hrs.	t=7 hrs. 53 min.	
T-Cold Exch. Lower Plate Inlet End	°C A9	-10.4	-16.3	-19.9	-23.5		
T-Lower Sink Inlet End	°C A10	21.5	21.3	21.2	20.4		
T-Ethylene Glycol Inlet	°C A11	-9.7	-16.0	-20.0	-23.6	-26.3	
T-Ethylene Glycol Outlet	°C A12	-12.5	-18.2	-21.8	-25.1	-28.6	
T-Inlet H ₂ O Upper Sink	°F B1	62.3	61.6	62.1	62.0		
T-Outlet H ₂ O Upper Sink	°F B2	71.4	70.2	71.0	69.9	loss flow in pump	
T-Inlet H ₂ O Lower Sink	°F B3	63.2	62.7	63.2	62.7		
T-Outlet H ₂ O Lower Sink	°F B4	72.2	71.3	71.7	70.5		
Pump I/V	AMP/VOLT	1.35/11.8	1.58/13.1	2.0/15.0	2.55/17.6		
f (Flow Meter)		240	220	230	215	170	

A summary of the calculated parameters at the transient condition where calculations were arbitrarily terminated is given in Tables 6.4.1 and 6.4.2 for both test simulations. The calculated results of the most important system parameters throughout the entire cool down period are shown as the solid curves in Figures 6.4.1 and 6.4.2. The actual test data are plotted onto these calculations for comparison.

It is observed that the actual current is below the predicted values. This is due to the lower calculated versus measured cascade resistance as shown in Table 6.1.1.1. However, the cool down characteristics illustrated in Figures 6.4.1 and 6.4.2 were in excellent agreement with predictions. The fact that the heat sink ran at lower than predicted temperatures was consistent with lower than calculated power consumed and exhausted into the heat sinks.

Improvements could be made in the thermal model by determining the parameters of the actual thermoelectric material composing the cascades and exchanging the resulting formulae with the "standard" formulae utilized in the model. Nevertheless, the basic thermal model is sufficiently accurate for use as a tool to study various parametrizations of the system.

6.5 Performance Predictions

Performance prediction calculations were made using the computer model verified by the acceptance tests. No attempt was made to improve or otherwise change the mathematical model, thus, the same slight discrepancy in electrical current and corresponding heat sink temperature

TABLE 6.4.1

Date 11/11/76

ZERO LOAD ΔT PERFORMANCE DESCRIPTION
 54 TE UNITS, MODEL NO. CX952-70-31
 WIRED AS 18 PARALLEL SETS OF 3 UNITS IN SERIES

Thermoelectric Device Physical Description

Height (inch) = 0.4300 Weight (grams) = 21.129
 Resistance (ohm) = 1.579
 Cold MCP per TE Device (Watt sec/C) = 0.37528E+02
 Hot MCP per TE Device (Watt sec/C) = 0.26973E+03
 Cold, Hot, Interstage Conductivity (Watt/Sq in/C) = 6.3, 6.3, 20.0

<u>TE Material</u>	<u>COMM</u>	<u>COMM</u>
Couple Distribution	70.00	31.00
Pellet Width (inch)	0.0550	0.0850
Pellet Length (inch)	0.0630	0.1510
Stage Area (Square inch)	1.4400	1.4400
Alumina Thickness (inch)	0.0300	0.0300
Bus Bar Thickness (inch)	0.0150	0.0320

<u>Exchanger Factor</u>	<u>Hot</u>	<u>Cold</u>
Fluid Type	Water	
Plate Temperature ($^{\circ}$ K)	297.54	219.83
Inlet Temperature ($^{\circ}$ K)	291.00	
Outlet Temperature ($^{\circ}$ K)	295.64	219.83
Heat Out of Fluid (Watt)	-1223.49	0.00
Plate to Plate Loss (Watt)	15.22	15.22
Prandtl Number	7.07451	
Reynolds Number	58.63	0.00
Nusselt Number	6.4963	
Heat Transfer Coefficient	127.9112	
Pressure Drop (in H ₂ O)	0.1087	0.0000
Bend Factor (in H ₂ O)	0.0013	0.0000
Flow Rate (lb/hr)	500.00	0.00
Exchanger Weight (Lb)	41.75	11.60
Fin Length (inches)	12.000	12.000
Fin Height (inches)	1.205	0.510
Fin Thickness (inches)	0.015	0.015
Number of Fins	32.00	32.00
Fin Epsilon	0.70948	

TABLE 6.4.2

11/12/76

SYSTEM PERFORMANCE DESCRIPTION
 54 TE UNITS, MODEL NO. CX952-70-31
 WIRED AS 18 PARALLEL SETS OF 3 UNITS IN SERIES

Thermoelectric Device Physical Description

Height (inch) = 0.4300, Weight (gram) = 21.129, Resistance (ohm): = 1.579
 Cold MCP per TE Device (Watt Sec/C) = 0.55448E+02
 Hot MCP per TE Device (Watt Sec/C) = 0.26973E+03
 Cold, Hot, Interstage Conductivity (Watt/sq in/C) = 6.3, 6.3, 20.0

<u>TE Material</u>	<u>COMM</u>	<u>COMM</u>
Couple Distribution	70.00	31.00
Pellet Width (inch)	0.0550	0.0850
Pellet Length (inch)	0.0630	0.1510
Stage Area (Square inch)	1.4400	1.4400
Alumina Thickness (inch)	0.0300	0.0300
Bus Bar Thickness (inch)	0.0150	0.0320

<u>Exchanger Factor</u>	<u>Hot</u>	<u>Cold</u>
Fluid Type	Water	60%EG
Plate Temperature (°K)	297.46	237.41
Inlet Temperature (°K)	290.40	239.54
Outlet Temperature (°K)	295.41	237.88
Heat Out of Fluid (Watt)	-1321.18	140.84
Plate to Plate Loss (Watt)	11.76	11.76
Prandtl Number	7.15441	634.70589
Reynolds Number	58.04	1.30
Nusselt Number	6.4964	5.1570
Heat Transfer Coefficient	127.7608	77.8288
Pressure Drop (in H ₂ O)	0.1098	24.1420
Bend Factor (in H ₂ O)	0.0013	0.0065
Flow Rate (lb/hr)	500.00	250.00
Exchanger Weight (lb)	41.75	11.60
Fin Length (inches)	12.000	12.000
Fin Height (inches)	1.205	0.510
Fin Thickness (inches)	0.015	0.015
Number of Fins	32.00	32.00
Fin Epsilon	0.70924	0.77939

(Table 6.4.2 Continued)

Table 6.4.2 (Continued)

Reservoir Characteristics

Ambient Temperature (°K)	297.09
Res Temperature (°K)	239.54
MCP (Watt Min/C)	2650.0
Loss Cond (Watt/C)	0.4859
Q Loss (Watt)	43.56

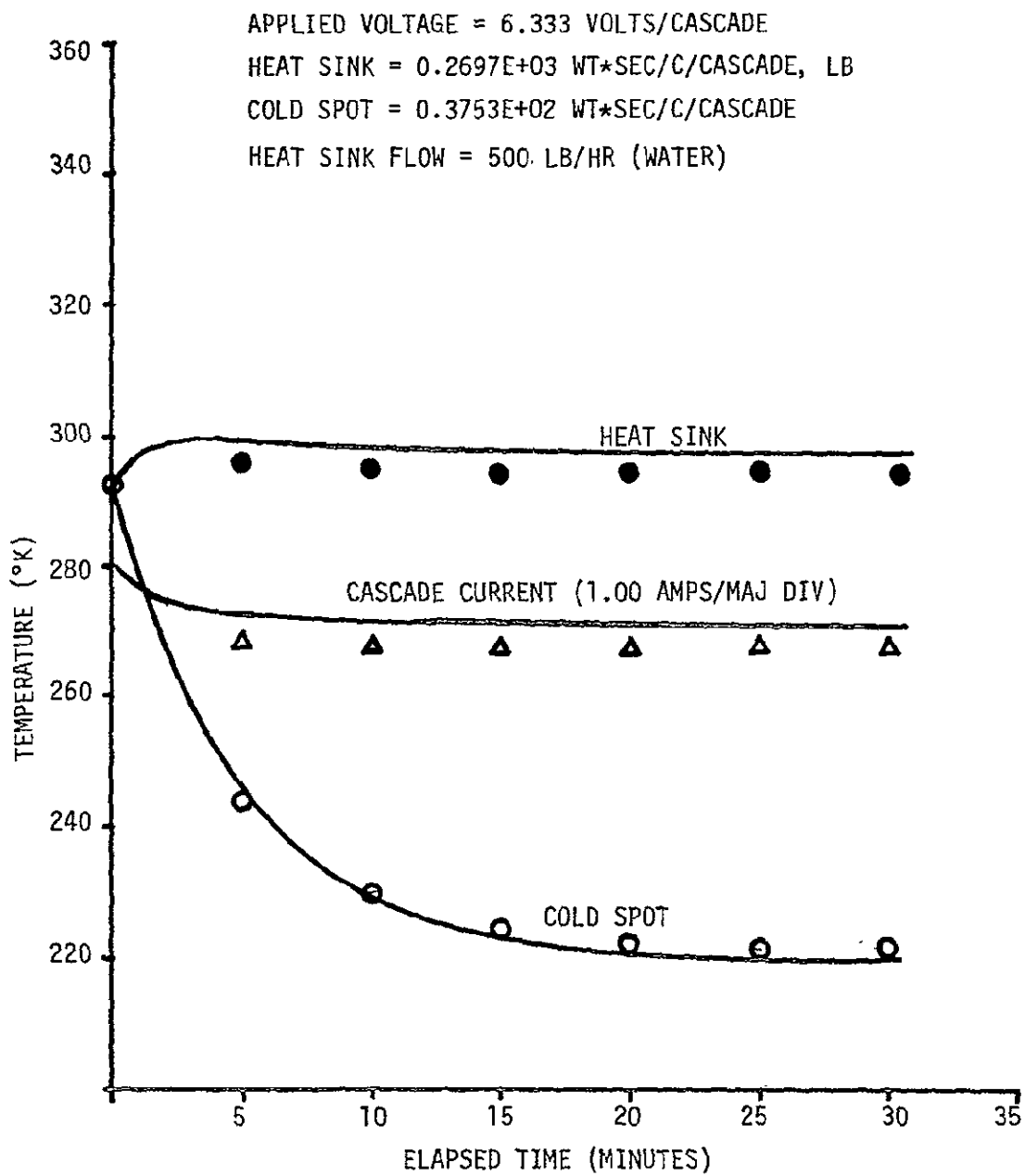


FIGURE 6.4.1 Zero Load ΔT Transient Performance

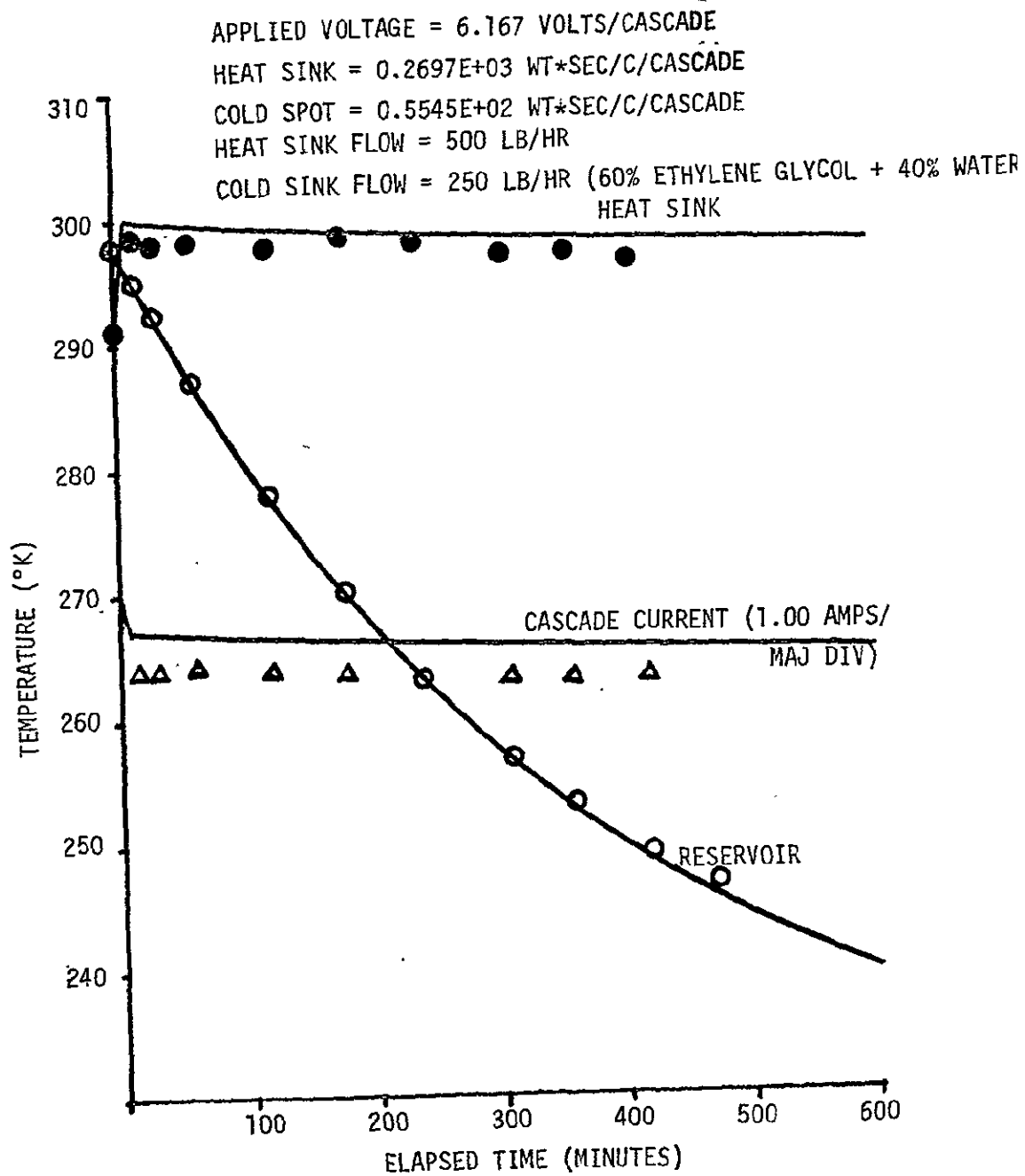


FIGURE 6.4.2 System Transient Performance

is expected to be present in these calculations.

The following set of input parameters were used:

Cold Inlet Temperature: 0°F (255.4°K), 35°F (274.9°K),
70°F (294.3°K)

Hot Flow Rate: 100, 300, 500 lb/hr.

Cold Flow Rate: 100, 150, 200, 300, 500, 700 lb/hr.

Hot Inlet Temperature: 70°F (294.3°K)

Applied Voltage: 18.5 volts

The resulting system parameters for each combination of the above input parameters are given in the appendix. The most important output parameter, cold outlet temperature, is illustrated in Figure 6.5.1. It was plotted as the inverse of the cold flow rate because of the nearly linear dependence, as observed.

PERFORMANCE PREDICTION

Heat Sink Inlet = 70°F

Applied Voltage = 18.5 volts

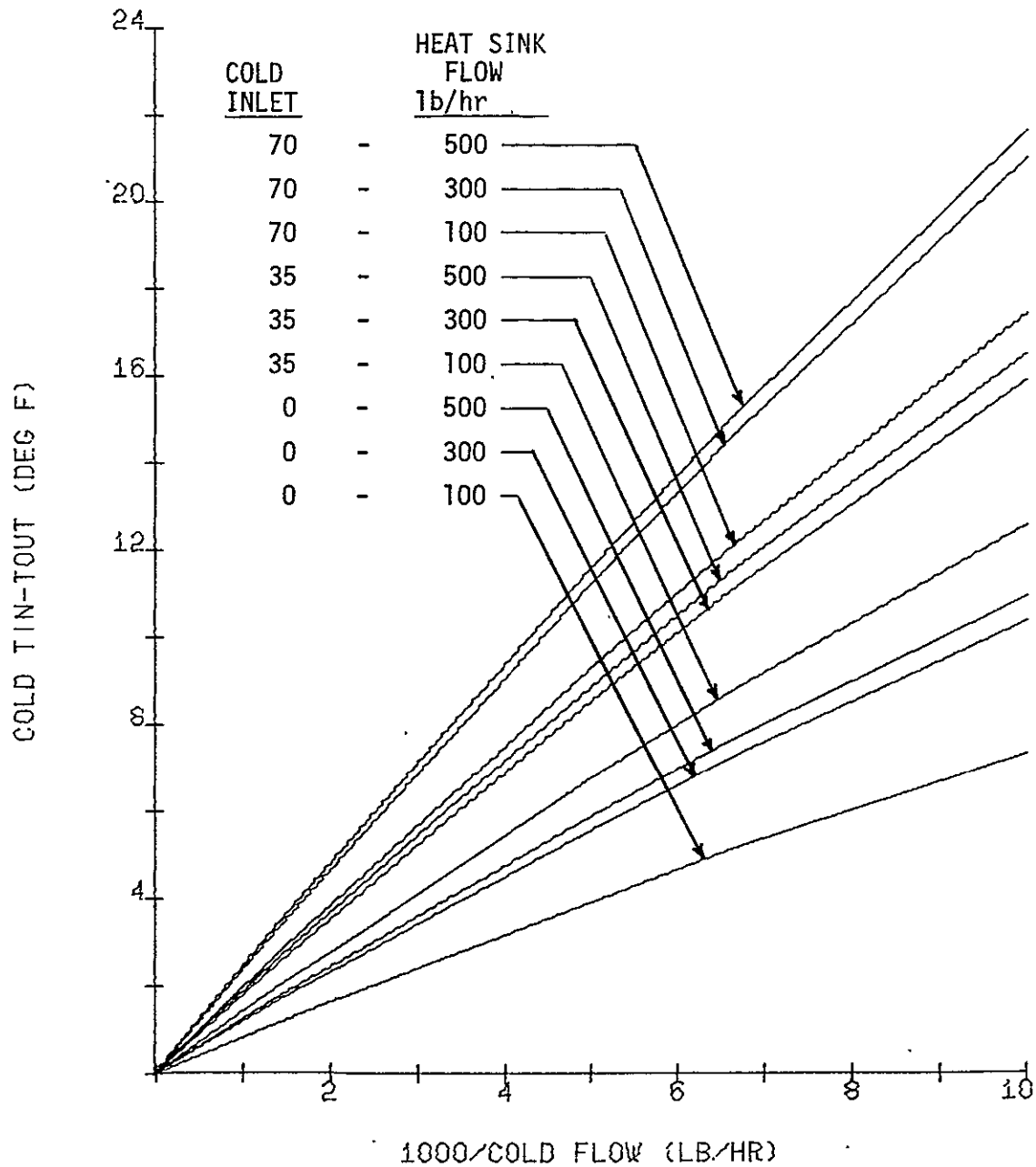


FIGURE 6.5.1

7. CONCLUSIONS

The first three conceptual phases of this program involved the thermal control of a cloud physics expansion chamber by attaching thermoelectric devices directly onto the chamber. The initial concept was based on a very optimistic input from the space lab heat exchanger. The subsequent two concepts both provided technically feasible system approaches to meet all contractual requirements. Indeed, the third phase was initiated as per MSFC's request solely for the purpose of improving cool down speed -- a parameter not specified in the contract.

Near the conclusion of the third phase, BWTE was informed of a specific transient requirement. Simple calculations using this requirement plus the mass of the chamber revealed that the transient condition could not be met on a real time basis using thermoelectrics. The limiting factor was the thermoelectric cooling "efficiency" limitation at the specified temperature conditions. This resulted in a complete redirection of effort and a fourth and final phase of activity.

A thermoelectric chiller unit was proposed to cool a liquid reservoir to -30°C for use in the thermal control of the chamber. A corresponding contract modification was made altering the scope of effort. A thermal model was generated and the resulting chiller was designed, fabricated, tested, and delivered. This unit met all requirements of the modified contract. The accuracy of the thermal model was validated and used for parametric study of system variables.

APPENDIX

Performance Calculations

(See Figure 6.5)

Input Parameters Varied:

- 1) Cold Inlet Temperature: 0°F (255.4°K), 35°F (274.9°K), 70°F (294.3°K)
- 2) Hot Flow Rate: 100, 300, 500 lb/hr.
- 3) Cold Flow Rate: 100, 150, 200, 300, 500, 700 lb/hr.

Input Parameters Constant:

- 1) Hot Inlet Temperature: 70°F (294.3°K)
- 2) Applied Voltage: 18.5 volts

$$T_C \text{ inlet} = 0^\circ\text{F} = 255.4^\circ\text{K}$$

SYSTEM PERFORMANCE DESCRIPTION DATE 01/20/77 1237
 54 TE UNITS MODEL NO. CX952-70-31
 WIRED AS 18 PARALLEL SETS OF 3 UNITS IN SERIES

THERMOELECTRIC DEVICE PHYSICAL DESCRIPTION
 HEIGHT (INCH)= 0.4300 WEIGHT (GRAM)= 21.129 RESISTANCE (OHM)= 1.579
 COLD M*CP PER TE DEVICE (WATT*SEC/C)= 0.55448E+02
 HOT M*CP PER TE DEVICE (WATT*SEC/C)= 0.26973E+03
 COLD, HOT, INTERSTAGE CONDUCTIVITY (WATT/SQ IN/C)= 6.3, 6.3, 20.0

T.E. MATERIAL	COMM	COMM
COUPLE DISTRIBUTION=	70.00	31.00
PELLET WIDTH (INCH)=	0.0550	0.0850
PELLET LENGTH (INCH)=	0.0630	0.1510
STAGE AREA (SQ INCH)=	1.4400	1.4400
ALUMINA THKNS (INCH)=	0.0300	0.0300
SUS-BAR THKNS (INCH)=	0.0150	0.0320

EXCHANGER FACTOR	HOT	COLD
FLUID TYPE	WATER	60%EG
PLATE TEMP (K)	316.77	251.19
INLET TEMP (K)	294.30	255.40
OUTLET TEMP (K)	316.73	251.32
HT OUT OF FLO(W)	-1182.91	144.88
PLT-PLT LOSS (W)	12.85	12.85
PRANDTL NUMBER	5.22663	231.32274
REYNOLDS NUMBER	15.33	1.50
NUSSELT NUMBER	6.3440	5.0151
HEAT TRANSF COEF	129.3271	75.0602
PRSS DRP(IN H2O)	0.0165	3.3508
BEND FRIC(IN H2O)	0.0001	0.0010
FLOW RT (LB/HR)	100.00	100.00
EXCH HEIGHT (LR)	41.75	11.60
FIN LENGTH (IN)	12.000	12.000
LVR LENGTH (IN)	12.000	12.000
FIN HEIGHT (IN)	1.205	0.510
FIN THKNSS (IN)	0.015	0.015
NUMBER OF FINS	32.00	32.00
FIN EPSILON	0.99801	0.96944

STEADY-STATE SYSTEM POWER CONDITIONS
 VOLTAGE (VOLTS) 18.500
 CURRENT (AMPS) 56.408
 POWER (WATTS) 1043.539

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SYSTEM PERFORMANCE DESCRIPTION DATE 01/21/77 1240
 54 TE UNITS MODEL NO. CX952-70-31
 AIRED AS 1A PARALLEL SETS OF 3 UNITS IN SERIES

THERMoeLECTRIC DEVICE PHYSICAL DESCRIPTION
 HEIGHT (INCH)= 0.4300 WEIGHT (GRAM)= 21.129 RESISTANCE (OHM)= 1.579
 COLD M*CP PER TE DEVICE (WATT*SEC/C)= 0.55448E+02
 HOT M*CP PER TE DEVICE (WATT*SEC/C)= 0.26973E+03
 COLD, HOT, INTERSTAGE CONDUCTIVITY (WATT/SQ IN/C)= 6.3, 6.3, 20.0

T.E. MATERIAL	COMM	COMM
COUPLE DISTRIBUTION=	70.00	31.00
PELLET WIDTH (INCH)=	0.0550	0.0850
PELLET LENGTH (INCH)=	0.0630	0.1510
STAGE AREA (SQ INCH)=	1.4400	1.4400
ALUMINA THKNS (INCH)=	0.0300	0.0300
BUS-BAR THKNS (INCH)=	0.0150	0.0320

EXCHANGER FACTOR	HOT	COLD
FLUID TYPE	WATER	60%EG
PLATE TEMP (K)	316.72	252.21
INLET TEMP (K)	294.30	255.40
OUTLET TEMP (K)	316.67	252.52
HT OUT OF FLD (W)	-1179.86	153.72
PLT-PLT LOSS (W)	12.63	12.63
PRANDTL NUMBER	5.23048	223.14757
REYNOLDS NUMBER	15.32	2.34
NUSSELT NUMBER	6.3440	5.0759
HEAT TRANSF COEF	129.3156	75.9451
PRSS DRP (IN H2O)	0.0165	4.8408
REND FTR (IN H2O)	0.0001	0.0024
FLOW RT (LR/HK)	100.00	150.00
EXCH WEIGHT (LB)	41.75	11.60
FIN LENGTH (IN)	12.000	12.000
LVK LENGTH (IN)	12.000	12.000
FIN HEIGHT (IN)	1.205	0.510
FIN THKNSS (IN)	0.015	0.015
NUMBER OF FINS	32.00	32.00
FIN EPSILON	0.99801	0.90437

STEADY-STATE SYSTEM POWER CONDITIONS
 VOLTAGE (VOLTS) 19.500
 CURRENT (AMPS) 56.529
 POWER (WATTS) 1045.785

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SYSTEM PERFORMANCE DESCRIPTION DATE 01/20/77 1237
 54 TE UNITS MODEL NO. CX952-70-31
 WIRED AS 18 PARALLEL SETS OF 3 UNITS IN SERIES

THERMOELECTRIC DEVICE PHYSICAL DESCRIPTION .
 HEIGHT (INCH)= 0.4300 WEIGHT (GRAM)= 21.129 RESISTANCE (OHM)= 1.579
 COLD M*CP PER TE DEVICE (WATT*SEC/C)= 0.55448E+02
 HOT M*CP PER TE DEVICE (WATT*SEC/C)= 0.26973E+03
 COLD, HOT, INTERSTAGE CONDUCTIVITY (WATT/SQ IN/C)= 6.3, 6.3, 20.0

T.E. MATERIAL	COMM	COMM
COUPLE DISTRIBUTION=	70.00	31.00
PELLET WIDTH (INCH)=	0.0550	0.0850
PELLET LENGTH (INCH)=	0.0630	0.1510
STAGE AREA (SQ INCH)=	1.4400	1.4400
ALUMINA THKNS (INCH)=	0.0300	0.0300
AUS-HAR THKNS (INCH)=	0.0150	0.0320

EXCHANGER FACTOR	HOT	COLD
FLUID TYPE	WATER	60%EG
PLATE TEMP (K)	317.22	252.79
INLET TEMP (K)	294.30	255.40
OUTLET TEMP (K)	317.17	253.23
HT OUT OF FLD (K)	-1206.15	154.25
PLT-PLT LOSS (W)	12.62	12.62
PRANDTL NUMBER	5.19916	218.43366
REYNOLDS NUMBER	15.40	3.19
NUSSELT NUMBER	6.3440	5.1347
HEAT TRANSFER COEF	129.4096	76.8111
PRSS DRP (IN H2O)	0.0164	6.3125
REND FTR (IN H2O)	0.0001	0.0042
FLOW HT (LB/HR)	100.00	200.00
EXCH WEIGHT (LB)	41.75	11.60
FIN LENGTH (IN)	12.000	12.000
LVR LENGTH (IN)	12.000	12.000
FIN HEIGHT (IN)	1.205	0.510
FIN THKNSS (IN)	0.015	0.015
NUMBER OF FINS	32.00	32.00
FIN EPSILON	0.99401	0.83095

STEADY-STATE SYSTEM POWER CONDITIONS
 VOLTAGE (VOLTS) 18.500
 CURRENT (AMPS) 56.373
 POWER (WATTS) 1042.897

SYSTEM PERFORMANCE DESCRIPTION DATE 01/20/77 1237
 54 TE UNITS MODEL NO. CX952-70-31
 WIRED AS 18 PARALLEL SETS OF 3 UNITS IN SERIES

THERMOELECTRIC DEVICE PHYSICAL DESCRIPTION
 HEIGHT (INCH)= 0.4300 WEIGHT (GRAM)= 21.129 RESISTANCE (OHM)= 1.579
 COLD M*CP PER TE DEVICE (WATT*SEC/C)= 0.55448E+02
 HOT M*CP PER TE DEVICE (WATT*SEC/C)= 0.26973E+03
 COLD, HOT, INTERSTAGE CONDUCTIVITY (WATT/SQ IN/C)= 6.3, 6.3, 20.0

T.E. MATERIAL	COMM	COMM
COUPLE DISTRIBUTION=	70.00	31.00
PELLET WIDTH (INCH)=	0.0550	0.0850
PELLET LENGTH (INCH)=	0.0630	0.1510
STAGE AREA (SQ INCH)=	1.4400	1.4400
ALUMINA THKNS (INCH)=	0.0300	0.0300
BUS-BAR THKNS (INCH)=	0.0150	0.0320

EXCHANGER FACTOR	HOT	COLD
FLUID TYPE	WATER	60%EG
PLATE TEMP (K)	316.99	253.27
INLET TEMP (K)	294.30	255.40
OUTLET TEMP (K)	316.94	253.90
HT OUT OF FLO(W)	-1194.14	159.91
PLT-PLT LOSS (W)	12.48	12.48
PRANDTL NUMBER	5.21386	214.15346
REYNOLDS NUMBER	15.36	4.88
NUSSELT NUMBER	6.3440	5.2464
HEAT TRANSF COEF	129.3654	78.4666
PRSS DRP(IN H2O)	0.0165	9.2773
BEND FRIC(IN H2O)	0.0001	0.0094
FLUX RT (LB/HR)	100.00	300.00
EXCH WEIGHT (LB)	41.75	11.60
FIN LENGTH (IN)	12.000	12.000
LVR LENGTH (IN)	12.000	12.000
FIN HEIGHT (IN)	1.205	0.510
FIN THKNSS (IN)	0.015	0.015
NUMBER OF FINS	32.00	32.00
FIN EPSILON	0.99A01	0.70118

STEADY-STATE SYSTEM POWER CONDITIONS
 VOLTAGE (VOLTS) 18.500
 CURRENT (AMPS) 56.516
 POWER (WATTS) 1045.539

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SYSTEM PERFORMANCE DESCRIPTION DATE 01/20/77 1237
 S4 TE UNITS MODEL NO. CX952-70-31
 AINED AS 1A PARALLEL SETS OF 3 UNITS IN SERIES

THERMOELECTRIC DEVICE PHYSICAL DESCRIPTION
 HEIGHT (INCH)= 0.4300 WEIGHT (GRAM)= 21.129 RESISTANCE (OHM)= 1.579
 COLD M*CP PER TE DEVICE (WATT*SEC/C)= 0.55448E+02
 HOT M*CP PER TE DEVICE (WATT*SEC/C)= 0.26973E+03
 COLD,HOT,INTERSTAGE CONDUCTIVITY (WATT/SQ IN/C)= 6.3, 6.3, 20.0

T.E. MATERIAL	COMM	COMM
COUPLE DISTRIBUTION=	70.00	31.00
PELLET WIDTH (INCH)=	0.0550	0.0850
PELLET LENGTH (INCH)=	0.0630	0.1510
STAGE AREA (SQ INCH)=	1.4400	1.4400
ALUMINA THKNS (INCH)=	0.0300	0.0300
BUS-BAR THKNS (INCH)=	0.0150	0.0320

EXCHANGER FACTOR	HOT	COLD
FLUID TYPE	WATER	50%EG
PLATE TEMP (K)	317.01	253.67
INLET TEMP (K)	294.30	255.40
OUTLET TEMP (K)	316.96	254.49
HT OUT OF FLO(W)	-1195.22	162.95
PLT=PLT LOSS (W)	12.41	12.41
PRANDTL NUMBER	5.21262	210.51846
REYNOLDS NUMBER	15.37	8.29
NUSSELT NUMBER	6.3440	5.4488
HEAT TRANSF CUEF	129.3691	81.4804
PRSS DWP(IN H2O)	0.0165	15.1972
BEND FTR(IN H2O)	0.0001	0.0261
FLOW RT (LP/HK)	100.00	500.00
EXCH HEIGHT (LH)	41.75	11.60
FIN LENGTH (IN)	12.000	12.000
LVR LENGTH (IN)	12.000	12.000
FIN HEIGHT (IN)	1.205	0.510
FIN THKNSS (IN)	0.015	0.015
NUMBER OF FINS	32.00	32.00
FIN EPSILON	0.99801	0.52775

STEADY-STATE SYSTEM POWER CONDITIONS
 VOLTAGE (VOLTS) 18.500
 CURRENT (AMPS) 56.545
 POWER (WATTS) 1046.079

SYSTEM PERFORMANCE DESCRIPTION DATE 01/20/77 1237
 54 TE UNITS MODEL NO. CX952-70-31
 WIRED AS 1A PARALLEL SETS OF 3 UNITS IN SERIES

THERMOELECTRIC DEVICE PHYSICAL DESCRIPTION
 HEIGHT (INCH)= 0.4300 WEIGHT (GRAM)= 21.129 RESISTANCE (OHM)= 1.579
 COLD M*CP PER TE DEVICE (WATT*SEC/C)= 0.55448E+02
 HOT M*CP PER TE DEVICE (WATT*SEC/C)= 0.26973E+03
 COLD, HOT, INTERSTAGE CONDUCTIVITY (WATT/SQ IN/C)= 6.3, 6.3, 20.0

T.E. MATERIAL	COMM	COMM
COUPLE DISTRIBUTION=	70.00	31.00
PELLET WIDTH (INCH)=	0.0550	0.0450
PELLET LENGTH (INCH)=	0.0630	0.1510
STAGE AREA (SQ INCH)=	1.4400	1.4400
ALUMINA THKNS (INCH)=	0.0300	0.0300
BUS-BAR THKNS (INCH)=	0.0150	0.0320

EXCHANGER FACTOR	HOT	COLD
FLUID TYPE	WATER	60%EG
PLATE TEMP (K)	317.39	253.87
INLET TEMP (K)	294.30	255.40
OUTLET TEMP (K)	317.39	254.75
HT OUT OF FLO (K)	1215.47	161.81
PLT-PLT LOSS (W)	12.44	12.44
PRANDTL NUMBER	5.18865	208.88056
REYNOLDS NUMBER	15.43	11.70
NUSSELT NUMBER	6.3400	5.6295
HEAT TRANSF CUEF	129.4413	84.1771
PRSS DWP (IN H2O)	0.0164	21.1168
BEND FTR (IN H2O)	0.0001	0.0512
FLOW RT (LR/HK)	100.00	700.00
EXCH WEIGHT (LB)	41.75	11.60
FIN LENGTH (IN)	12.000	12.000
LVR LENGTH (IN)	12.000	12.000
FIN HEIGHT (IN)	1.205	0.510
FIN THKNSS (IN)	0.015	0.015
NUMBER OF FINS	32.00	32.00
FIN EPSILON	0.99902	0.42437

STEADY-STATE SYSTEM POWER CONDITIONS

VOLTAGE (VOLTS) 18.500
 CURRENT (AMPS) 56.403
 POWER (WATTS) 1043.447

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SYSTEM PERFORMANCE DESCRIPTION DATE 01/20/77 1237
 54 TE UNITS MODEL NO. CX952-70-31
 WIRED AS 14 PARALLFL SETS OF 3 UNITS IN SERIES

THERMoeLECTRIC DEVICE PHYSICAL DESCRIPTION
 HEIGHT (INCH)= 0.4300 WEIGHT (GRAM)= 21.129 RESISTANCE (OHM)= 1.579
 COLD M*CP PER TE DEVICE (WATT*SEC/C)= 0.55448E+02
 HOT M*CP PER TE DEVICE (WATT*SEC/C)= 0.26973E+03
 COLD, HOT, INTERSTAGE CONDUCTIVITY (WATT/SQ IN/C)= 6.3, 6.3, 20.0

T.E. MATERIAL	COMM	COMM
COUPLE DISTRIBUTION=	70.00	31.00
PELLET WIDTH (INCH)=	0.0550	0.0450
PELLET LENGTH (INCH)=	0.0630	0.1510
STAGE AREA (SQ INCH)=	1.4400	1.4400
ALUMINA THKNS (INCH)=	0.0300	0.0300
BUS-BAR THKNS (INCH)=	0.0150	0.0320

EXCHANGER FACTOR	HOT	COLD
FLUID TYPE	WATER	60XEG
PLATE TEMP (K)	304.05	249.47
INLET TEMP (K)	294.30	255.40
OUTLET TEMP (K)	302.42	249.65
HT OUT OF FLO(W)	-1347.35	203.67
PLT-PLT LUSS (W)	10.69	10.69
PRANDTL NUMBER	6.17518	243.35095
REYNOLDS NUMBER	39.70	1.42
NUSSSELT NUMBER	6.4173	5.0145
HEAT TRANSFR COEF	128.2758	75.0859
PRSS DRP(IN H2O)	0.0576	3.5338
BEND FR(IN H2O)	0.0005	0.0010
FLUX RT (LR/HK)	300.00	100.00
EXCH HEIGHT (LR)	41.75	11.60
FIN LENGTH (IN)	12.000	12.000
LVR LENGTH (IN)	12.000	12.000
FIN HEIGHT (IN)	1.205	0.510
FIN THKNSS (IN)	0.015	0.015
NUMBER OF FINS	52.00	32.00
FIN EPSILON	0.87299	0.96972

STEADY-STATE SYSTEM POWER CONDITIONS
 VOLTAGE (VOLTS) 18.500
 CURRENT (AMPS) 61.564
 POWER (WATTS) 1138.932

SYSTEM PERFORMANCE DESCRIPTION DATE 01/21/77 1240
 54 TE UNITS MODEL NO. CX952-70-31
 WIRED AS 18 PARALLEL SETS OF 3 UNITS IN SERIES

THEMEOLECTRIC DEVICE PHYSICAL DESCRIPTION
 HEIGHT (INCH)= 0.4300 WEIGHT (GRAM)= 21.129 RESISTANCE (OHM)= 1.579
 COLD M*CP PER TE DEVICE (WATT*SEC/C)= 0.55448E+02
 HOT M*CP PER TE DEVICE (WATT*SEC/C)= 0.26973E+03
 COLD, HOT, INTERSTAGE CONDUCTIVITY (WATT/SQ IN/C)= 6.3, 6.3, 20.0

T.E. MATERIAL	COMM	COMM
COUPLE DISTRIBUTION=	70.00	31.00
PELLET WIDTH (INCH)=	0.0550	0.0850
PELLET LENGTH (INCH)=	0.0630	0.1510
STAGE AREA (SQ INCH)=	1.4400	1.4400
ALUMINA THKNS (INCH)=	0.0300	0.0300
BUS-BAR THKNS (INCH)=	0.0150	0.0320

EXCHANGER FACTOR	HOT	COLD
FLUID TYPE	WATER	60XEG
PLATE TEMP (K)	304.10	250.92
INLET TEMP (K)	294.30	255.40
OUTLET TEMP (K)	302.85	251.35
HT OUT OF FLO(W)	-1353.25	215.63
PLT-PLT LOSS (W)	10.41	10.41
PRANDTL NUMBER	6.17334	231.11472
REYNOLDS NUMBER	39.71	2.25
NUSSELT NUMBER	6.4173	5.0752
HEAT TRANSF COEF	128.2801	75.9597
PRSS DRP(IN H2O)	0.0576	5.0222
BEND FTR(IN H2O)	0.0005	0.0024
FLOW RT (LR/HR)	400.00	150.00
EXCH WEIGHT (LB)	41.75	11.60
FIN LENGTH (IN)	12.000	12.000
LVR LENGTH (IN)	12.000	12.000
FIN HEIGHT (IN)	1.205	0.510
FIN THKNSS (IN)	0.015	0.015
NUMBER OF FINS	32.00	32.00
FIN EPSILON	0.87300	0.90477

STEADY-STATE SYSTEM POWER CONDITIONS
 VOLTAGE (VOLTS) 18.500
 CURRENT (AMPS) 61.681
 POWER (WATTS) 1141.103

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SYSTEM PERFORMANCE DESCRIPTION DATE 01/20/77 1237
 54 TE UNITS MODEL NO. CX952-70-31
 WIRED AS 18 PARALLEL SETS OF 3 UNITS IN SERIES

THEMMELECTRIC DEVICE PHYSICAL DESCRIPTION
 HEIGHT (INCH)= 0.4300 WEIGHT (GRAM)= 21.129 RESISTANCE (OHM)= 1.579
 COLD M*CP PER TE DEVICE (WATT*SEC/C)= 0.5544E+02
 HOT M*CP PER TE DEVICE (WATT*SEC/C)= 0.26973E+03
 COLD,HOT,INTERSTAGE CONDUCTIVITY (WATT/SQ IN/C)= 6.3, 6.3, 20.0

I.E. MATERIAL	COMM	COMM
COUPLE DISTRIBUTION=	70.00	31.00
PELLET WIDTH (INCH)=	0.0550	0.0850
PELLET LENGTH (INCH)=	0.0630	0.1510
STAGE AREA (SQ INCH)=	1.4400	1.4400
ALUMINA THKNS (INCH)=	0.0300	0.0300
BUS-BAR THKNS (INCH)=	0.0150	0.0320

EXCHANGER FACTOR	HOT	COLD
FLUID TYPE	WATER	60%EG
PLATE TEMP (K)	304.19	251.67
INLET TEMP (K)	294.30	255.40
OUTLET TEMP (K)	302.94	252.30
HT OUT OF FLD(W)	-1366.53	220.51
PLT-PLT LOSS (W)	10.29	10.29
PRANDTL NUMBER	6.16579	224.59921
REYNOLDS NUMBER	39.75	3.10
NUSSELT NUMBER	6.4173	5.1340
HEAT TRANSF COEF	128.2978	76.8199
PRSS DRP(IN H2O)	0.0575	6.4995
BEND FTR(IN H2O)	0.0005	0.0042
FLOW RT (LB/HR)	300.00	200.00
EXCH WEIGHT (LB)	41.75	11.60
FIN LENGTH (IN)	12.000	12.000
LVR LENGTH (IN)	12.000	12.000
FIN HEIGHT (IN)	1.205	0.510
FIN THKNS (IN)	0.015	0.015
NUMBER OF FINS	32.00	32.00
FIN EPSILON	0.87302	0.83137

STEADY-STATE SYSTEM POWER CONDITIONS
 VOLTAGE (VOLTS) 18.500
 CURRENT (AMPS) 61.700
 POWER (WATTS) 1141.447

SYSTEM PERFORMANCE DESCRIPTION DATE 01/20/77 1237
 54 TE UNITS MODEL NO. CX952-70-31
 WIRED AS 1A PARALLEL SETS OF 3 UNITS IN SERIES

THERMOELECTRIC DEVICE PHYSICAL DESCRIPTION *
 HEIGHT (INCH)= 0.4300 WEIGHT (GRAM)= 21.129 RESISTANCE (OHM)= 1.579
 COLD M*CP PER TE DEVICE (WATT*SEC/C)= 0.55448E+02
 HOT M*CP PER TE DEVICE (WATT*SEC/C)= 0.26973E+03
 COLD, HOT, INTERSTAGE CONDUCTIVITY (WATT/SQ IN/C)= 6.3, 6.3, 20.0

T.E. MATERIAL	COMM	COMM
COUPLE DISTRIBUTION=	70.00	31.00
PELLET WIDTH (INCH)=	0.0550	0.0850
PELLET LENGTH (INCH)=	0.0630	0.1510
STAGE AREA (SQ INCH)=	1.4400	1.4400
ALUMINA THKNS (INCH)=	0.0300	0.0300
BUS-RAW THKNS (INCH)=	0.0150	0.0320

EXCHANGER FACTOR	HOT	COLD
FLUID TYPE	WATER	60%EG
PLATE TEMP (K)	304.24	252.38
INLET TEMP (K)	294.30	255.40
OUTLET TEMP (K)	302.98	253.28
HT OUT OF FLO (W)	-1372.79	225.92
PLT-PLT LOSS (W)	10.16	10.16
PRANDTL NUMBER	6.16278	218.10911
REYNOLDS NUMBER	39.77	4.79
NUSSELT NUMBER	6.4173	5.2456
HEAT TRANSF COEF	128.4048	78.4685
PRSS DRP (IN H2O)	0.0575	9.4570
BEND FTR (IN H2O)	0.0005	0.0094
FLOW WT (LB/HK)	300.00	300.00
EXCH WEIGHT (LB)	41.75	11.60
FIN LENGTH (IN)	12.000	12.000
LVR LENGTH (IN)	12.000	12.000
FIN HEIGHT (IN)	1.205	0.510
FIN THKNSS (IN)	0.015	0.015
NUMBER OF FINS	32.00	32.00
FIN EPSILON	0.87503	0.70149

STEADY-STATE SYSTEM POWER CONDITIONS
 VOLTAGE (VOLTS) 18.500
 CURRENT (AMPS) 61.745
 POWER (WATTS) 1142.277

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SYSTEM PERFORMANCE DESCRIPTION DATE 01/20/77 1237
 54 IE UNITS MODEL NO. CX952-70-31
 WIRED AS 18 PARALLEL SETS OF 3 UNITS IN SERIES

THERMOELECTRIC DEVICE PHYSICAL DESCRIPTION
 HEIGHT (INCH)= 0.4300 WEIGHT (GRAM)= 21.129 RESISTANCE (OHM)= 1.579
 COLD M*CP PER IE DEVICE (WATT*SEC/C)= 0.55448E+02
 HOT M*CP PER IE DEVICE (WATT*SEC/C)= 0.26973E+03
 COLD,HOT, INTERSTAGE CONDUCTIVITY (WATT/SQ IN/C)= 6.3, 6.3, 20.0

T.E. MATERIAL	COMM	COMM
COUPLE DISTRIBUTION=	70.00	31.00
PELLET WIDTH (INCH)=	0.0550	0.0850
PELLET LENGTH (INCH)=	0.0630	0.1510
STAGE AREA (SQ INCH)=	1.4400	1.4400
ALUMINA THKNS (INCH)=	0.0300	0.0300
BUS-BAR THKNS (INCH)=	0.0150	0.0320

EXCHANGER FACTOR	HOT	COLD
FLUID TYPE	WATER	60%EG
PLATE TEMP (K)	304.27	252.95
INLET TEMP (K)	294.30	255.40
OUTLET TEMP (K)	303.00	254.11
HT OUT OF FLO(W)	-1376.98	230.41
PLT=PLT LOSS (A)	10.05	10.05
PRANDTL NUMBER	6.16077	212.87389
REYNOLDS NUMBER	39.78	-8.19
NUSSELT NUMBER	6.4173	5.4480
HEAT TRANSF COEF	128.3095	81.4769
PRSS DRP(IN H2O)	0.0575	15.3754
REND FR(IN H2O)	0.0005	0.0261
FLOW RT (LH/HR)	300.00	500.00
EXCH WEIGHT (LH)	41.75	11.60
FIN LENGTH (IN)	12.000	12.000
LVR LENGTH (IN)	12.000	12.000
FIN HEIGHT (IN)	1.205	0.510
FIN THKNSS (IN)	0.015	0.015
NUMBER OF FINS	52.00	32.00
FIN EPSILON	0.87303	0.52792

STEADY-STATE SYSTEM POWER CONDITIONS
 VOLTAGE (VOLTS) 18.500
 CURRENT (AMPS) 61.782
 POWER (WATTS) 1142.973

SYSTEM PERFORMANCE DESCRIPTION DATE 01/20/77 1237
 54 TE UNITS MODEL NO. CX952-70-31
 WIRED AS 18 PARALLEL SETS OF 3 UNITS IN SERIES

THEMMELECTRIC DEVICE PHYSICAL DESCRIPTION
 HEIGHT (INCH)= 0.4300 WEIGHT (GRAM)= 21.129 RESISTANCE (OHM)= 1.579
 COLD M*CP PER TE DEVICE (WATT*SEC/C)= 0.55448E+02
 HOT M*CP PER TE DEVICE (WATT*SEC/C)= 0.26973E+03
 COLD, HOT, INTERSTAGE CONDUCTIVITY (WATT/SQ IN/C)= 6.3, 6.3, 20.0

T.E. MATERIAL	COMM	COMM
COUPLE DISTRIBUTION=	70.00	31.00
PELLET WIDTH (INCH)=	0.0550	0.0850
PELLET LENGTH (INCH)=	0.0630	0.1510
STAGE AREA (SQ INCH)=	1.4400	1.4400
ALUMINA THKNS (INCH)=	0.0300	0.0300
BUS-BAR THKNS (INCH)=	0.0150	0.0320

EXCHANGER FACTOR	HOT	COLD
FLUID TYPE	WATER	60%EG
PLATE TEMP (K)	304.29	253.21
INLET TEMP (K)	294.30	255.40
OUTLET TEMP (K)	303.02	254.47
HT OUT OF FLD(W)	-1379.84	232.20
PLT-PLT LOSS (W)	10.01	10.01
PRANDTL NUMBER	6.15935	210.61755
REYNOLDS NUMBER	39.79	11.60
NUSSELT NUMBER	6.4173	5.6287
HEAT TRNSFR COEF	128.3129	84.1711
PRSS DRP(IN H2O)	0.0575	21.3007
BEND FIR(IN H2O)	0.0005	0.0512
FLOW RT (LR/HR)	300.00	700.00
EXCH WEIGHT (LB)	41.75	11.60
FIN LENGTH (IN)	12.000	12.000
LVR LENGTH (IN)	12.000	12.000
FIN HEIGHT (IN)	1.205	0.510
FIN THKNSS (IN)	0.015	0.015
NUMBER OF FINS	32.00	32.00
FIN EPSILON	0.87304	0.42447

STEADY-STATE SYSTEM POWER CONDITIONS

VOLTAGE (VOLTS)	18.500
CURRENT (AMPS)	61.796
POWER (WATTS)	1143.224

SYSTEM PERFORMANCE DESCRIPTION DATE 01/20/77 1708
 54 TE UNITS MODEL NO. CX952-70-31
 WIRED AS 1A PARALLEL SETS OF 3 UNITS IN SERIES

THERMOELECTRIC DEVICE PHYSICAL DESCRIPTION*
 HEIGHT (INCH)= 0.4300 HEIGHT (GRAM)= 21.129 RESISTANCE (OHM)= 1.579
 COLD M*CP PER TE DEVICE (WATT*SEC/C)= 0.55448E+02
 HOT M*CP PER TE DEVICE (WATT*SEC/C)= 0.26973E+03
 COLD HOT INTERSTAGE CONDUCTIVITY (WATT/SQ IN/C)= 6.3, 6.3, 20.0

T.E. MATERIAL	COMM	COMM
COUPLE DISTRIBUTION=	70.00	31.00
PELLET WIDTH (INCH)=	0.0550	0.0850
PELLET LENGTH (INCH)=	0.0630	0.1510
STAGE AREA (SQ INCH)=	1.4400	1.4400
ALUMINA THKNS (INCH)=	0.0300	0.0300
BUS-BAR THKNS (INCH)=	0.0150	0.0320

EXCHANGER FACTOR	HOT	COLD
FLUID TYPE	WATER	60%EG
PLATE TEMP (K)	301.64	249.14
INLET TEMP (K)	294.30	255.40
OUTLET TEMP (K)	299.52	249.33
HT OUT OF FLD(W)	1377.02	214.77
PLT-PLT LOSS (W)	10.28	10.28
PRANDTL NUMBER	6.43784	245.72185
REYNOLDS NUMBER	63.76	1.41
NUSSELT NUMBER	6.4947	5.0144
HEAT TRANSF COEF	129.2126	75.0908
PRSS DRP(IN H2O)	0.1000	3.5699
BEND FTR(IN H2O)	0.0013	0.0010
FLOW RT (L4/HR)	500.00	100.00
EXCH WEIGHT (LB)	41.75	11.60
FIN LENGTH (IN)	12.000	12.000
LVR LENGTH (IN)	12.000	12.000
FIN HEIGHT (IN)	1.205	0.510
FIN THKNSS (IN)	0.015	0.015
NUMBER OF FINS	32.00	32.00
FIN EPSILON	0.71157	0.96978

STEADY-STATE SYSTEM POWER CONDITIONS
 VOLTAGE (VOLTS) 18.500
 CURRENT (AMPS) 62.581
 POWER (WATTS) 1157.754

SYSTEM PERFORMANCE DESCRIPTION DATE 01/21/77 1240
 54 TE UNITS MODEL NO. CX952-70-31
 WIRED AS 1A PARALLEL SETS OF 3 UNITS IN SERIES

THERMoeLECTRIC DEVICE PHYSICAL DESCRIPTION
 HEIGHT (INCH)= 0.4300 WEIGHT (GRAM)= 21.129 RESISTANCE (OHM)= 1.579
 COLD M*CP PER TE DEVICE (WATT*SEC/C)= 0.55448E+02
 HOT M*CP PER TE DEVICE (WATT*SEC/C)= 0.26973E+03
 COLD, HOT, INTERSTAGE CONDUCTIVITY (WATT/SQ IN/C)= 6.3, 6.3, 20.0

I.E. MATERIAL	COMM	COMM
COUPLE DISTRIBUTION=	70.00	31.00
PELLET WIDTH (INCH)=	0.0550	0.0850
PELLET LENGTH (INCH)=	0.0630	0.1510
STAGE AREA (SQ INCH)=	1.4400	1.4400
ALUMINA THKNS (INCH)=	0.0300	0.0300
BUS-BAR THKNS (INCH)=	0.0150	0.0320

EXCHANGER FACTOR	HOT	COLD
FLUID TYPE	WATER	60XEG
PLATE TEMP (K)	301.69	250.67
INLET TEMP (K)	294.30	255.40
OUTLET TEMP (K)	299.56	251.12
HT OUT OF FLO(W)	-1386.90	227.67
PLT-PLT LOSS (W)	9.99	9.99
PRANDTL NUMBER	6.43520	232.72792
REYNOLDS NUMBER	63.78	2.24
NUSSELT NUMBER	6.4947	5.0751
HEAT TRANSF COEF	129.2184	75.9625
PRSS DRP(IN H2O)	0.1000	5.0590
BEND FTR(IN H2O)	0.0013	0.0024
FLO* RT (LH/HK)	500.00	150.00
EXCH HEIGHT (LH)	41.75	11.60
FIN LENGTH (IN)	12.000	12.000
LVR LENGTH (IN)	12.000	12.000
FIN HEIGHT (IN)	1.205	0.510
FIN THKNSS (IN)	0.015	0.015
NUMBER OF FINS	32.00	32.00
FIN EPSILON	0.71158	0.90485

STEADY-STATE SYSTEM POWER CONDITIONS
 VOLTAGE (VOLTS) 18.500
 CURRENT (AMPS) 62.697
 POWER (WATTS) 1159.889

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SYSTEM PERFORMANCE DESCRIPTION DATE 01/21/77 1109
 54 TE UNITS MODEL NO. CX952-70-31
 REQ AS 18 PARALLEL SETS OF 3 UNITS IN SERIES

THERMoeLECTRIC DEVICE PHYSICAL DESCRIPTION*

IGHT (INCH)= 0.4300 WEIGHT (GRAM)= 21.129 RESISTANCE (OHM)= 1.579
 LD *CP PER TE DEVICE (WATT*SEC/C)= 0.55448E+02
 UT *CP PER TE DEVICE (WATT*SEC/C)= 0.26973E+03
 LD, HOT, INTERSTAGE CONDUCTIVITY (WATT/SQ IN/C)= 6.3, 6.3, 20.0

T.E. MATERIAL	COMM	COMM
COUPLE DISTRIBUTION=	70.00	31.00
ELLET WIDTH (INCH)=	0.0550	0.0850
ELLET LENGTH (INCH)=	0.0630	0.1510
TAGE AREA (SQ INCH)=	1.4400	1.4400
LUMINA THKNS (INCH)=	0.0300	0.0300
US-BAR THKNS (INCH)=	0.0150	0.0320

CHANGER FACTOR	HOT	COLD
FLUID TYPE	WATER	60ZEG
PLATE TEMP (K)	301.73	251.46
INLET TEMP (K)	294.30	255.40
UTLET TEMP (K)	299.59	252.12
OUT OF FLD(*)	-1394.99	233.10
T-PLT LOSS (*)	9.85	9.85
PRANDTL NUMBER	6.43274	225.83087
EYNOLDS NUMBER	63.81	3.08
NUSSELT NUMBER	6.4947	5.1339
AT TRANSF CUEF	129.2238	76.8216
SS DRP(IN H2O)	0.1000	6.5369
ND FTR(IN H2O)	0.0013	0.0042
LUM RT (LB/HR)	500.00	200.00
CH WEIGHT (LB)	41.75	11.60
IN LENGTH (IN)	12.000	12.000
VR LENGTH (IN)	12.000	12.000
IN HEIGHT (IN)	1.205	0.510
IN THKNSS (IN)	0.015	0.015
NUMBER OF FINS	32.00	32.00
FIN EPSILON	0.71159	0.83145

STEADY-STATE SYSTEM POWER CONDITIONS

VOLTAGE (VOLTS) 18.500
 CURRENT (AMPS) 62.748
 POWER (WATTS) 1160.831

SYSTEM PERFORMANCE DESCRIPTION DATE 01/20/77 1237
 54 TE UNITS MODEL NO. CX952-70-31
 WIRED AS 14 PARALLEL SETS OF 3 UNITS IN SERIES

THERMOELECTRIC DEVICE PHYSICAL DESCRIPTION
 HEIGHT (INCH)= 0.4300 HEIGHT (GRAM)= 21.129 RESISTANCE (OHM)= 1.579
 COLD M*CP PER TE DEVICE (WATT*SEC/C)= 0.55448E+02
 HOT M*CP PER TE DEVICE (WATT*SEC/C)= 0.26973E+03
 COLD, HOT, INTERSTAGE CONDUCTIVITY (WATT/SQ IN/C)= 6.3, 6.3, 20.0

T.E. MATERIAL	COMM	COMM
COUPLE DISTRIBUTION=	70.00	31.00
PELLET WIDTH (INCH)=	0.0550	0.0850
PELLET LENGTH (INCH)=	0.0630	0.1510
STAGE AREA (SQ INCH)=	1.4400	1.4400
ALUMINA THKNS (INCH)=	0.0300	0.0300
RUS-BAR THKNS (INCH)=	0.0150	0.0320

EXCHANGER FACTOR	HOT	COLD
FLUID TYPE	WATER	60%EG
PLATE TEMP (K)	301.78	252.22
INLET TEMP (K)	294.30	255.40
OUTLET TEMP (K)	299.63	253.17
HT OUT OF FLD (K)	-1404.42	238.48
PLT-PLT LOSS (K)	9.71	9.71
PRANDTL NUMBER	6.42933	218.87123
REYNOLDS NUMBER	63.84	4.77
NUSSELT NUMBER	6.4947	5.2455
HEAT TRANSF COEF	129.2313	78.4689
PRSS DRP (IN H2O)	0.0999	9.4917
BEND FR (IN H2O)	0.0013	0.0094
FLOW RT (LH/HR)	500.00	300.00
EXCH WEIGHT (LB)	41.75	11.60
FIN LENGTH (IN)	12.000	12.000
LVR LENGTH (IN)	12.000	12.000
FIN HEIGHT (IN)	1.205	0.510
FIN THKNSS (IN)	0.015	0.015
NUMBER OF FINS	32.00	32.00
FIN EPSILON	0.71160	0.70155

STEADY-STATE SYSTEM POWER CONDITIONS
 VOLTAGE (VOLTS) 18.500
 CURRENT (AMPS) 62.790
 POWER (WATTS) 1161.607

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SYSTEM PERFORMANCE DESCRIPTION DATE 01/20/77 1237
 54 TE UNITS MODEL NO. CX952-70-31
 WIRED AS 18 PARALLEL SETS OF 3 UNITS IN SERIES

THERMOELECTRIC DEVICE PHYSICAL DESCRIPTION
 HEIGHT (INCH)= 0.4300 HEIGHT (GRAM)= 21.129 RESISTANCE (OHM)= 1.579
 COLD M*CP PER TE DEVICE (WATT*SEC/C)= 0.55448E+02
 HOT M*CP PER TE DEVICE (WATT*SEC/C)= 0.26973E+03
 COLD, HOT, INTERSTAGE CONDUCTIVITY (WATT/SQ IN/C)= 6.3, 6.3, 20.0

I.E. MATERIAL	COMM	COMM
COUPLE DISTRIBUTION=	70.00	31.00
PELLET WIDTH (INCH)=	0.0550	0.0450
PELLET LENGTH (INCH)=	0.0630	0.1510
STAGE AREA (SQ INCH)=	1.4400	1.4400
ALUMINA THKNS (INCH)=	0.0300	0.0300
BUS-BAR THKNS (INCH)=	0.0150	0.0320

EXCHANGER FACTOR	HOT	COLD
FLUID TYPE	WATER	60%EG
PLATE TEMP (K)	301.81	252.82
INLET TEMP (K)	294.30	255.40
OUTLET TEMP (K)	299.65	254.04
HT OUT OF FLD(W)	-1409.57	243.04
PLT-PLT LOSS (W)	9.60	9.60
PRANDTL NUMBER	6.42774	213.31674
REYNOLDS NUMBER	63.85	8.17
NUSSELT NUMBER	6.4947	5.4478
HEAT TRANSFER COEF	129.2348	81.4762
PRSS DRP (IN H2O)	0.0999	15.4089
BEND FR (IN H2O)	0.0013	0.0261
FLOW RT (LB/HR)	500.00	500.00
EXCH WEIGHT (LB)	41.75	11.60
FIN LENGTH (IN)	12.000	12.000
LVR LENGTH (IN)	12.000	12.000
FIN HEIGHT (IN)	1.205	0.510
FIN THKNSS (IN)	0.015	0.015
NUMBER OF FINS	32.00	32.00
FIN EPSILON	0.71160	0.52795

STEADY-STATE SYSTEM POWER CONDITIONS
 VOLTAGE (VOLTS) 1A.500
 CURRENT (AMPS) 62.831
 POWER (WATTS) 1162.369

SYSTEM PERFORMANCE DESCRIPTION DATE 01/20/77 1237
 54 TE UNITS MODEL NO. CX952-70-31
 WIRED AS 18 PARALLEL SETS OF 3 UNITS IN SERIES

THERMOELECTRIC DEVICE PHYSICAL DESCRIPTION
 HEIGHT (INCH)= 0.4300 WEIGHT (GRAM)= 21.129 RESISTANCE (OHM)= 1.579
 COLD M*CP PER TE DEVICE (WATT*SEC/C)= 0.55444E+02
 HOT M*CP PER TE DEVICE (WATT*SEC/C)= 0.26973E+03
 COLD, HOT, INTERSTAGE CONDUCTIVITY (WATT/SQ IN/C)= 6.3, 6.3, 20.0

T.E. MATERIAL	COMM	COMM
COUPLE DISTRIBUTION=	70.00	31.00
PELLET WIDTH (INCH)=	0.0550	0.0950
PELLET LENGTH (INCH)=	0.0630	0.1510
STAGE AREA (SQ INCH)=	1.4400	1.4400
ALUMINA THKNS (INCH)=	0.0300	0.0300
BUS-BAR THKNS (INCH)=	0.0150	0.0320

EXCHANGER FACTOR	HOT	COLD
FLUID TYPE	WATER	60%EG
PLATE TEMP (K)	301.83	253.08
INLET TEMP (K)	294.30	255.40
OUTLET TEMP (K)	299.66	254.42
HT OUT OF FLD(W)	-1412.64	245.27
PLT-PLT LOSS (W)	9.55	9.55
PRANDTL NUMBER	6.42661	210.93621
REYNOLDS NUMBER	63.86	11.58
NUSSELT NUMBER	6.4947	5.6285
HEAT TRANSF COEF	129.2373	84.1700
PRSS DRP(IN H2O)	0.0999	21.3344
REND FTR(IN H2O)	0.0013	0.0512
FLOW RT (LB/HR)	500.00	700.00
EXCH WEIGHT (LB)	41.75	11.60
FIN LENGTH (IN)	12.000	12.000
LVR LENGTH (IN)	12.000	12.000
FIN HEIGHT (IN)	1.205	0.510
FIN THKNSS (IN)	0.015	0.015
NUMBER OF FINS	32.00	32.00
FIN EPSILON	0.71161	0.42449

STEADY-STATE SYSTEM POWER CONDITIONS
 VOLTAGE (VOLTS) 18.500
 CURRENT (AMPS) 62.846
 POWER (WATTS) 1162.652

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$$T_c \text{ inlet} = 35^\circ\text{F} = 274.9^\circ\text{K}$$

SYSTEM PERFORMANCE DESCRIPTION DATE 01/20/77 1237
 54 TE UNITS MODEL NO. CX952-70-31
 WIRED AS 18 PARALLEL SETS OF 3 UNITS IN SERIES

THERMoeLECTRIC DEVICE PHYSICAL DESCRIPTION
 HEIGHT (INCH)= 0.4300 WEIGHT (GRAM)= 21.129 RESISTANCE (OHM)= 1.579
 COLD M*CP PER TE DEVICE (WATT*SEC/C)= 0.55448E+02
 HOT M*CP PER TE DEVICE (WATT*SEC/C)= 0.26973E+03
 COLD, HOT, INTERSTAGE CONDUCTIVITY (WATT/SQ IN/C)= 6.3, 6.3, 20.0

T.E. MATERIAL	COMM	COMM
COUPLE DISTRIBUTION=	70.00	31.00
PELLET WIDTH (INCH)=	0.0550	0.0850
PELLET LENGTH (INCH)=	0.0630	0.1510
STAGE AREA (SQ INCH)=	1.4400	1.4400
ALUMINA THKNS (INCH)=	0.0300	0.0300
BUS-BAR THKNS (INCH)=	0.0150	0.0320

EXCHANGER FACTOR	HOT	COLD
FLUID TYPE	WATER	60%EG
PLATE TEMP (K)	319.22	267.66
INLET TEMP (K)	294.30	274.90
OUTLET TEMP (K)	319.17	267.92
HT OUT OF FLO(W)	-1311.66	259.25
PLT-PLT LOSS (W)	10.10	10.10
PRANDTL NUMBER	5.08255	91.38002
REYNOLDS NUMBER	15.71	4.01
NUSSELT NUMBER	6.3439	5.0266
HEAT TRANSFER COEF	129.7670	74.4676
PRSS DRP(IN H2O)	0.0161	1.2625
BEND FR(IN H2O)	0.0001	0.0011
FLOW RT (LB/HR)	100.00	100.00
EXCH HEIGHT (LB)	41.75	11.60
FIN LENGTH (IN)	12.000	12.000
LVR LENGTH (IN)	12.000	12.000
FIN HEIGHT (IN)	1.205	0.510
FIN THKNSS (IN)	0.015	0.015
NUMBER OF FINS	32.00	32.00
FIN EPSILON	0.99803	0.96346

STEADY-STATE SYSTEM POWER CONDITIONS
 VOLTAGE (VOLTS) 18.500
 CURRENT (AMPS) 56.962
 POWER (WATTS) 1053.802

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SYSTEM PERFORMANCE DESCRIPTION DATE 01/21/77 1240
 54 TE UNITS MODEL NO. CX952-70-31
 WIRED AS 1A PARALLEL SETS OF 3 UNITS IN SERIES

THEMEOELECTRIC DEVICE PHYSICAL DESCRIPTION
 HEIGHT (INCH)= 0.4300 WEIGHT (GRAM)= 21.129 RESISTANCE (OHM)= 1.579
 COLD M*CP PER TE DEVICE (WATT*SEC/C)= 0.5544E+02
 HOT M*CP PER TE DEVICE (WATT*SEC/C)= 0.26973E+03
 COLD, HOT, INTERSTAGE CONDUCTIVITY (WATT/SQ IN/C)= 6.3, 6.3, 20.0

T.F. MATERIAL	COMM	COMM
COUPLE DISTRIBUTION=	70.00	31.00
PELLET WIDTH (INCH)=	0.0550	0.0850
PELLET LENGTH (INCH)=	0.0630	0.1510
STAGE AREA (SQ INCH)=	1.4400	1.4400
ALUMINA THKNS (INCH)=	0.0300	0.0300
BUS-BAR THKNS (INCH)=	0.0150	0.0320

EXCHANGER FACTOR	HOT	COLD
FLUID TYPE	WATER	60%EG
PLATE TEMP (K)	319.22	269.41
INLET TEMP (K)	294.30	274.90
OUTLET TEMP (K)	319.18	270.00
HT OUT OF FLO (K)	-1311.94	273.43
PLT-PLT LOSS (K)	9.76	9.76
PRANDTL NUMBER	5.08261	87.36576
REYNOLDS NUMBER	15.71	6.32
NUSSLEI NUMBER	6.3439	5.0949
HEAT TRANSF COEF	129.7668	75.4309
PRSS DRP (IN H2O)	0.0161	1.8068
BEND FTR (IN H2O)	0.0001	0.0024
FLOW RT (LR/HK)	100.00	150.00
EXCH WEIGHT (LB)	41.75	11.60
FIN LENGTH (IN)	12.000	12.000
LVR LENGTH (IN)	12.000	12.000
FIN HEIGHT (IN)	1.205	0.510
FIN THKNSS (IN)	0.015	0.015
NUMBER OF FINS	32.00	32.00
FIN EPSILON	0.99803	0.89224

STEADY-STATE SYSTEM POWER CONDITIONS
 VOLTAGE (VOLTS) 18.500
 CURRENT (AMPS) 57.125
 POWER (WATTS) 1056.811

SYSTEM PERFORMANCE DESCRIPTION DATE 01/21/77 1109
 54 TE UNITS MODEL NO. CX952-70-31
 WIRING AS 18 PARALLEL SETS OF 3 UNITS IN SERIES

THERMOELECTRIC DEVICE PHYSICAL DESCRIPTION
 HEIGHT (INCH)= 0.4300 WEIGHT (GRAM)= 21.129 RESISTANCE (OHM)= 1.579
 COLD M*CP PER TE DEVICE (WATT*SEC/C)= 0.55448E+02
 HOT M*CP PER TE DEVICE (WATT*SEC/C)= 0.26973E+03
 COLD, HOT, INTERSTAGE CONDUCTIVITY (WATT/SQ IN/C)= 6.3, 6.3, 20.0

T.E. MATERIAL	COMM	COMM
COUPLE DISTRIBUTION=	70.00	31.00
PELLET WIDTH (INCH)=	0.0550	0.0850
PELLET LENGTH (INCH)=	0.0630	0.1510
STAGE AREA (SQ INCH)=	1.4400	1.4400
ALUMINA THKNS (INCH)=	0.0300	0.0300
BUS-BAR THKNS (INCH)=	0.0150	0.0320

EXCHANGER FACTOR	HOT	COLD
FLUID TYPE	WATER	60%EG
PLATE TEMP (K)	319.30	270.30
INLET TEMP (K)	294.30	274.90
OUTLET TEMP (K)	319.25	271.15
HT OUT OF FLO(W)	-1315.74	279.81
PLT-PLT LOSS (W)	9.60	9.60
PRANDTL NUMBER	5.07845	85.26795
REYNOLDS NUMBER	15.72	8.65
NUSSLELT NUMBER	6.3439	5.1615
HEAT TRANSF COEF	129.7798	76.3905
PRSS DRP(IN H2O)	0.0161	2.3491
3END FTR(IN H2O)	0.0001	0.0042
FLOW RT (L4/HR)	100.00	200.00
EXCH WEIGHT (LB)	41.75	11.60
FIN LENGTH (IN)	12.000	12.000
LVR LENGTH (IN)	12.000	12.000
FIN HEIGHT (IN)	1.205	0.510
FIN THKNSS (IN)	0.015	0.015
NUMBER OF FINS	32.00	32.00
FIN EPSILON	0.99803	0.81525

STEADY-STATE SYSTEM POWER CONDITIONS

VOLTAGE (VOLTS)	18.500
CURRENT (AMPS)	57.178
POWER (WATTS)	1057.795

SYSTEM PERFORMANCE DESCRIPTION DATE 01/20/77 1237
 54 TE UNITS MODEL NO. CX952-70-31
 WIRED AS 18 PARALLEL SETS OF 3 UNITS IN SERIES

THERMOELECTRIC DEVICE PHYSICAL DESCRIPTION
 HEIGHT (INCH)= 0.4300 WEIGHT (GRAM)= 21.129 RESISTANCE (OHM)= 1.579
 COLD M*CP PER TE DEVICE (WATT*SEC/C)= 0.55448E+02
 HOT M*CP PER TE DEVICE (WATT*SEC/C)= 0.26973E+03
 COLD, HOT, INTERSTAGE CONDUCTIVITY (WATT/SQ IN/C)= 6.3, 6.3, 20.0

T.E. MATERIAL	COMM	COMM
COUPLE DISTRIBUTION=	70.00	31.00
PELLET WIDTH (INCH)=	0.0550	0.0850
PELLET LENGTH (INCH)=	0.0630	0.1510
STAGE AREA (SQ INCH)=	1.4400	1.4400
ALUMINA THKNS (INCH)=	0.0300	0.0300
BUSBAR THKNS (INCH)=	0.0150	0.0320

EXCHANGER FACTOR	HOT	COLD
FLUID TYPE	WATER	60XEG
PLATE TEMP (K)	319.54	271.18
INLET TEMP (K)	294.30	274.90
OUTLET TEMP (K)	319.49	272.36
HT OUT OF FLO(W)	-1324.70	284.78
PLT-PLT LOSS (W)	9.47	9.47
PRANDTL NUMBER	5.06395	83.12581
REYNOLDS NUMBER	15.76	13.33
NUSSELT NUMBER	6.3439	5.2891
HEAT TRANSF COEF	129.4251	78.2492
PRSS DRP(IN H2O)	0.0161	3.4333
BEND FTR(IN H2O)	0.0001	0.0095
FLOW RT (L4/HR)	100.00	300.00
EXCH HEIGHT (LB)	41.75	11.60
FIN LENGTH (IN)	12.000	12.000
LVR LENGTH (IN)	12.000	12.000
FIN HEIGHT (IN)	1.205	0.510
FIN THKNSS (IN)	0.015	0.015
NUMBER OF FINS	32.00	32.00
FIN EPSILON	0.49804	0.68337

STEADY-STATE SYSTEM POWER CONDITIONS
 VOLTAGE (VOLTS) 18.500
 CURRENT (AMPS) 57.160
 POWER (WATTS) 1057.468

SYSTEM PERFORMANCE DESCRIPTION DATE 01/20/77 1237
 54 TE UNITS MODEL NO. CX952-70-31
 WIRED AS 14 PARALLEL SETS OF 3 UNITS IN SERIES

THERMoeLECTRIC DEVICE PHYSICAL DESCRIPTION
 HEIGHT (INCH)= 0.4300 WEIGHT (GRAM)= 21.129 RESISTANCE (OHM)= 1.579
 COLD M*CP PER TE DEVICE (WATT*SFC/C)= 0.55448E+02
 HOT M*CP PER TE DEVICE (WATT*SEC/C)= 0.26973E+03
 COLD, HOT, INTERSTAGE CONDUCTIVITY (WATT/SQ IN/C)= 6.3, 6.3, 20.0

T.E. MATERIAL	COMM	COMM
CUMPLE DTSIMBUTION=	70.00	31.00
PELLET WIDTH (INCH)=	0.0550	0.0850
PELLET LENGTH (INCH)=	0.0630	0.1510
STAGE AREA (SQ INCH)=	1.4400	1.4400
ALUMINA THKNS (INCH)=	0.0300	0.0300
BUS-BAR THKNS (INCH)=	0.0150	0.0320

EXCHANGER FACTOR	HOT	COLD
FLUID TYPE	WATER	60XEG
PLATE TEMP (K)	319.96	271.90
INLET TEMP (K)	294.30	274.90
OUTLET TEMP (K)	319.91	273.36
HT OUT OF FLU(W)	-1350.79	287.26
PLT-PLT LUSS (W)	9.42	9.42
PRANDTL NUMBER	5.03919	81.40005
REYNOLDS NUMBER	15.83	22.71
NUSSELT NUMBER	6.3439	5.5231
HEAT TRANSFER COEF	129.9030	81.6866
PRSS DRP (IN H2O)	0.0160	5.6073
BEND FTR (IN H2O)	0.0001	0.0264
FLOW RT (LR/HK)	100.00	500.00
EXCH WEIGHT (LB)	41.75	11.60
FIN LENGTH (IN)	12.000	12.000
LVR LENGTH (IN)	12.000	12.000
FIN HEIGHT (IN)	1.205	0.510
FIN THKNSS (IN)	0.015	0.015
NUMBER OF FINS	32.00	32.00
FIN EPSILON	0.99804	0.51209

STEADY-STATE SYSTEM POWER CONDITIONS
 VOLTAGE (VOLTS) 18.500
 CURRENT (AMPS) 57.057
 POWER (WATTS) 1059.553

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SYSTEM PERFORMANCE DESCRIPTION DATE 01/20/77 1237
 54 TE UNITS MODEL NO. CX952-70-31
 WIRED AS 14 PARALLEL SETS OF 3 UNITS IN SERIES

THERMOELECTRIC DEVICE PHYSICAL DESCRIPTION
 HEIGHT (INCH)= 0.4300 WEIGHT (GRAM)= 21.129 RESISTANCE (OHM)= 1.579
 COLD M*CP PER TE DEVICE (WATT*SEC/C)= 0.55448E+02
 HOT M*CP PER TE DEVICE (WATT*SEC/C)= 0.26473E+03
 COLD, HOT, INTERSTAGE CONDUCTIVITY (WATT/SQ IN/C)= 6.3, 6.3, 20.0

I.F. MATERIAL	COMM	COMM
COUPLE DISTRIBUTION=	70.00	31.00
PELLET WIDTH (INCH)=	0.0550	0.0850
PELLET LENGTH (INCH)=	0.0630	0.1510
STAGE AREA (SQ INCH)=	1.4400	1.4400
ALUMINA THKNS (INCH)=	0.0300	0.0300
RUS-BAR THKNS (INCH)=	0.0150	0.0320

EXCHANGER FACTOR	HOT	COLD
FLUID TYPE	WATER	60%FG
PLATE TEMP (K)	319.64	272.19
INLET TEMP (K)	294.30	274.90
OUTLET TEMP (K)	319.59	273.78
HT OUT OF FLO (K)	-1333.98	291.95
PLT-PLT LOSS (K)	9.29	9.29
PRANDTL NUMBER	5.05821	80.69266
REYNOLDS NUMBER	15.78	32.09
NUSSELT NUMBER	6.3439	5.7344
HEAT TRANSF COEFF	129.8431	84.7991
PRSS DRP (IN H2O)	0.0160	7.7926
BEND FR (IN H2O)	0.0001	0.0517
FLOW RT (LX/HX)	100.00	700.00
EXCH WEIGHT (LB)	41.75	11.60
FIN LENGTH (IN)	12.000	12.000
LVR LENGTH (IN)	12.000	12.000
FIN HEIGHT (IN)	1.205	0.510
FIN THKNSS (IN)	0.015	0.015
NUMBER OF FINS	32.00	32.00
FIN EPSILON	0.49804	0.41175

STEADY-STATE SYSTEM POWER CONDITIONS

VOLTAGE (VOLTS) 18.500
 CURRENT (AMPS) 57.214
 POWER (WATTS) 1058.467

SYSTEM PERFORMANCE DESCRIPTION DATE 01/20/77 1237
 54 IE UNITS MODEL NO. CX952-70-31
 WIRED AS 18 PARALLEL SETS OF 3 UNITS IN SERIES

THERMoeLECTRIC DEVICE PHYSICAL DESCRIPTION
 HEIGHT (INCH)= 0.4300 WEIGHT (GRAM)= 21.129 RESISTANCE (OHM)= 1.579
 COLD W*CP PER IE DEVICE (WATT*SEC/C)= 0.55448E+02
 HOT W*CP PER IE DEVICE (WATT*SEC/C)= 0.26973E+03
 COLD, HOT, INTERSTAGE CONDUCTIVITY (WATT/SG IN/C)= 6.3, 6.3, 20.0

T.E. MATERIAL	COMM	COMM
COUPLE DISTRIBUTION=	70.00	31.00
PELLET WIDTH (INCH)=	0.0550	0.0850
PELLET LENGTH (INCH)=	0.0630	0.1510
STAGE AREA (SG INCH)=	1.4400	1.4400
ALUMINA THKNS (INCH)=	0.0300	0.0300
BUS-BAR THKNS (INCH)=	0.0150	0.0320

EXCHANGER FACTOR	HOT	COLD
FLUID TYPE	WATER	60%EG
PLATE TEMP (K)	305.08	265.75
INLET TEMP (K)	294.30	274.90
OUTLET TEMP (K)	303.71	266.08
HT OUT OF FLUID	-1444.61	326.90
PLT-PLT LOSS (W)	7.70	7.70
PRANDTL NUMBER	6.10671	95.13932
REYNOLDS NUMBER	40.09	3.85
NUSSELI NUMBER	6.4172	5.0261
HEAT TRANSFER COEF	128.4371	74.5006
PRSS DRP (IN H2O)	0.0571	1.3173
BEND FR (IN H2O)	0.0005	0.0011
FLOW RT (LR/HR)	300.00	100.00
EXCH WEIGHT (LB)	41.75	11.60
FIN LENGTH (IN)	12.000	12.000
LVR LENGTH (IN)	12.000	12.000
FIN HEIGHT (IN)	1.205	0.510
FIN THKNSS (IN)	0.015	0.015
NUMBER OF FINS	32.00	32.00
FIN EPSILON	0.67314	0.96376

STEADY-STATE SYSTEM POWER CONDITIONS
 VOLTAGE (VOLTS) 18.500
 CURRENT (AMPS) 62.572
 POWER (WATTS) 1157.590

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SYSTEM PERFORMANCE DESCRIPTION DATE 01/21/77 1240
 54 TE UNITS MODEL NO. CX952-70-31
 WIRED AS 18 PARALLEL SETS OF 3 UNITS IN SERIES

THERMOELECTRIC DEVICE PHYSICAL DESCRIPTION
 HEIGHT (INCH)= 0.4300 WEIGHT (GRAM)= 21.129 RESISTANCE (OHM)= 1.579
 COLD M*CP PER TE DEVICE (WATT*SEC/C)= 0.5544E+02
 HOT M*CP PER TE DEVICE (WATT*SEC/C)= 0.26973E+03
 COLD, HOT, INTERSTAGE CONDUCTIVITY (WATT/SQ IN/C)= 6.3, 6.3, 20.0

T.E. MATERIAL	COMM	COMM
COUPLE DISTRIBUTION=	70.00	31.00
PELLET WIDTH (INCH)=	0.0550	0.0850
PELLET LENGTH (INCH)=	0.0630	0.1510
STAGE AREA (SQ INCH)=	1.4400	1.4400
ALUMINA THKNS (INCH)=	0.0300	0.0300
BUS-BAR THKNS (INCH)=	0.0150	0.0320

EXCHANGER FACTOR	HOT	COLD
FLUID TYPE	WATER	60XEG
PLATE TEMP (K)	305.13	267.97
INLET TEMP (K)	294.30	274.90
OUTLET TEMP (K)	305.76	268.72
HT OUT OF FLO(W)	-1496.48	344.83
PLT-PLT LOSS (W)	7.28	7.28
PRANDTL NUMBER	6.10404	89.81890
REYNOLDS NUMBER	40.11	6.13
NUSSELT NUMBER	6.4172	5.0943
HEAT TRANSF COEF	128.4435	75.4514
PRSS DRP(IN H2O)	0.0570	1.8604
BEND FTR(IN H2O)	0.0005	0.0024
FLOW RT (LB/HR)	300.00	150.00
EXCH WEIGHT (LB)	41.75	11.60
FIN LENGTH (IN)	12.000	12.000
LVR LENGTH (IN)	12.000	12.000
FIN HEIGHT (IN)	1.205	0.510
FIN THKNSS (IN)	0.015	0.015
NUMBER OF FINS	32.00	32.00
FIN EPSILON	0.87319	0.89265

STEADY-STATE SYSTEM POWER CONDITIONS
 VOLTAGE (VOLTS) 1A.500
 CURRENT (AMPS) 62.751
 POWER (WATTS) 1160.888

SYSTEM PERFORMANCE DESCRIPTION DATE 01/20/77 1237
 54 TE UNITS MODEL NO. CX952-70-31
 WIRED AS 18 PARALLEL SETS OF 3 UNITS IN SERIES

THERMOELECTRIC DEVICE PHYSICAL DESCRIPTION
 HEIGHT (INCH)= 0.4300 WEIGHT (GRAM)= 21.129 RESISTANCE (OHM)= 1.579
 COLD M*CP PER TE DEVICE (WATT*SEC/C)= 0.55448E+02
 HOT M*CP PER TE DEVICE (WATT*SEC/C)= 0.26973E+03
 COLD, HOT, INTERSTAGE CONDUCTIVITY (WATT/SQ IN/C)= 6.3, 6.3, 20.0

I.E. MATERIAL	COMM	COMM
COUPLE DISTRIBUTION=	70.00	31.00
PELLET WIDTH (INCH)=	0.0550	0.0850
PELLET LENGTH (INCH)=	0.0630	0.1510
STAGE AREA (SQ INCH)=	1.4400	1.4400
ALUMINA THKNS (INCH)=	0.0300	0.0300
BUS-BAR THKNS (INCH)=	0.0150	0.0320

EXCHANGER FACTOR	HOT	COLD
FLUID TYPE	WATER	60XEG
PLATE TEMP (K)	305.28	269.10
INLET TEMP (K)	294.30	274.90
OUTLET TEMP (K)	303.89	270.17
HT OUT OF FLD(W)	-1517.03	352.06
PLT-PLT LOSS (W)	7.09	7.09
PRANDTL NUMBER	6.09313	87.04656
REYNOLDS NUMBER	40.17	8.46
NUSSELT NUMBER	6.4172	5.1609
HEAT TRNSFR COEF	128.4694	76.4038
PRSS DRP(IN H2O)	0.0569	2.4008
BEND FR(IN H2O)	0.0005	0.0042
FLOW RT (LB/HR)	300.00	200.00
EXCH WEIGHT (LB)	41.75	11.60
FIN LENGTH (IN)	12.000	12.000
LVR LENGTH (IN)	12.000	12.000
FIN HEIGHT (IN)	1.205	0.510
FIN THKNSS (IN)	0.015	0.015
NUMBER OF FINS	32.00	32.00
FIN EPSILON	0.87322	0.81564

STEADY-STATE SYSTEM POWER CONDITIONS
 VOLTAGE (VOLTS) 18.500
 CURRENT (AMPS) 62.782
 POWER (WATTS) 1161.460

SYSTEM PERFORMANCE DESCRIPTION DATE 01/20/77 1237
 54 TE UNITS MODEL NO. CX952-70-31
 WIRED AS 18 PARALLEL SETS OF 3 UNITS IN SERIES

THERMoeLECTRIC DEVICE PHYSICAL DESCRIPTION
 HEIGHT (INCH)= 0.4300 WEIGHT (GRAM)= 21.129 RESISTANCE (OHM)= 1.579
 COLD M*CP PER TE DEVICE (WATT*SEC/C)= 0.55448E+02
 HOT M*CP PER TE DEVICE (WATT*SEC/C)= 0.26973E+03
 COLD, HOT, INTERSTAGE CONDUCTIVITY (WATT/SQ IN/C)= 6.3, 6.3, 20.0

T.E. MATERIAL	COMM	COMM
COUPLE DISTRIBUTION=	70.00	31.00
PELLET WIDTH (INCH)=	0.0550	0.0850
PELLET LENGTH (INCH)=	0.0630	0.1510
STAGE AREA (SQ INCH)=	1.4400	1.4400
ALUMINA THKNS (INCH)=	0.0300	0.0300
BUS-BAR THKNS (INCH)=	0.0150	0.0320

EXCHANGER FACTOR	HOT	COLD
FLUID TYPE	WATER	60%EG
PLATE TEMP (K)	305.35	270.19
INLET TEMP (K)	294.30	274.90
OUTLET TEMP (K)	303.95	271.68
HT OUT OF FLO(W)	-1526.28	360.24
PLT-PLT LOSS (W)	6.89	6.89
PRANDTL NUMBER	6.08971	84.31083
REYNOLDS NUMBER	40.20	13.13
NUSSELT NUMBER	6.4172	5.2884
HEAT TRANSF COEF	128.4800	78.2552
PRSS DROP (IN H2O)	0.0569	3.4850
BEND FR (IN H2O)	0.0005	0.0095
FLOW RT (LR/HK)	300.00	300.00
EXCH HEIGHT (LH)	41.75	11.60
FIN LENGTH (IN)	12.000	12.000
LVR LENGTH (IN)	12.000	12.000
FIN HEIGHT (IN)	1.205	0.510
FIN THKNSS (IN)	0.015	0.015
NUMBER OF FINS	32.00	32.00
FIN EPSILON	0.67323	0.68367

STEADY-STATE SYSTEM POWER CONDITIONS
 VOLTAGE (VOLTS) 18.500
 CURRENT (AMPS) 62.849
 POWER (WATTS) 1162.705

SYSTEM PERFORMANCE DESCRIPTION DATE 01/20/77 1237
 54 TE UNITS MODEL NO. CX952-70-31
 WIRED AS 14 PARALLEL SETS OF 3 UNITS IN SERIES

THERMoeLECTRIC DEVICE PHYSICAL DESCRIPTION
 HEIGHT (INCH)= 0.4300 WEIGHT (GRAM)= 21.129 RESISTANCE (OHM)= 1.579
 COLD M*CP PER TE DEVICE (WATT*SEC/C)= 0.55448E+02
 HOT M*CP PER TE DEVICE (WATT*SEC/C)= 0.26973E+03
 COLD, HOT, INTERSTAGE CONDUCTIVITY (WATT/SQ IN/C)= 6.3, 6.3, 20.0

T.E. MATERIAL	COMM	COMM
COUPLE DISTRIBUTION=	70.00	31.00
PELLET WIDTH (INCH)=	0.0550	0.0850
PELLET LENGTH (INCH)=	0.0630	0.1510
STAGE AREA (SQ INCH)=	1.4400	1.4400
ALUMINA THKNS (INCH)=	0.0300	0.0300
BUS-BAR THKNS (INCH)=	0.0150	0.0320

EXCHANGER FACTOR	HOT	COLD
FLUID TYPE	WATER	60%EG
PLATE TEMP (K)	305.40	271.07
INLET TEMP (K)	294.30	274.90
OUTLET TEMP (K)	303.99	272.94
HT OUT OF FLO(W)	-1533.84	366.64
PLT-PLT LOSS (W)	6.73	6.73
PRANDTL NUMBER	6.04511	42.12351
REYNOLDS NUMBER	40.22	22.50
NUSSELT NUMBER	6.4172	5.5224
HEAT TRANSFER COEFF	124.4886	81.6862
PRSS DRP (IN H2O)	0.0569	5.6597
BEND FRIC (IN H2O)	0.0005	0.0264
FLUX RT (L2/H2)	300.00	500.00
EXCH HEIGHT (L2)	41.75	11.60
FIN LENGTH (IN)	12.000	12.000
LVR LENGTH (IN)	12.000	12.000
FIN HEIGHT (IN)	1.205	0.510
FIN THKNSS (IN)	0.015	0.015
NUMBER OF FINS	32.00	32.00
FIN EPSILON	0.87324	0.51225

STEADY-STATE SYSTEM POWER CONDITIONS
 VOLTAGE (VOLTS) 18.500
 CURRENT (AMPS) 62.903
 POWER (WATTS) 1163.707

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SYSTEM PERFORMANCE DESCRIPTION DATE 01/20/77 1237
 54 TE UNITS MODEL NO. CX952-70-31
 WIRED AS 18 PARALLEL SETS OF 3 UNITS IN SERIES

THERMoeLECTRIC DEVICE PHYSICAL DESCRIPTION
 HEIGHT (INCH)= 0.4300 WEIGHT (GRAM)= 21.129 RESISTANCE (OHM)= 1.579
 COLD M*CP PER TE DEVICE (WATT*SEC/C)= 0.55448E+02
 HOT M*CP PER TE DEVICE (WATT*SEC/C)= 0.26973E+03
 COLD, HOT, INTERSTAGE CONDUCTIVITY (WATT/SQ IN/C)= 6.3, 6.3, 20.0

T.E. MATERIAL	COMM	COMM
COUPLE DISTRIBUTION=	70.00	31.00
PELLET WIDTH (INCH)=	0.0550	0.0850
PELLET LENGTH (INCH)=	0.0630	0.1510
STAGE AREA (SQ INCH)=	1.4400	1.4400
ALUMINA THKNS (INCH)=	0.0300	0.0300
BUS-BAR THKNS (INCH)=	0.0150	0.0320

EXCHANGER FACTOR	HOT	COLD
FLUID TYPE	WATER	60%EG
PLATE TEMP (K)	305.42	271.47
INLET TEMP (K)	294.30	274.90
OUTLET TEMP (K)	304.01	273.49
HT OUT OF FLU(W)	-1536.86	369.52
PLT-PLT LOSS (W)	6.65	6.65
PRANDTL NUMBER	6.08372	81.18763
REYNOLDS NUMBER	40.22	31.89
NUSSELT NUMBER	6.4172	5.7336
HEAT TRANSF COEF	128.4919	44.7960
PRSS DRP (IN H2O)	0.0569	7.8428
BEND FR (IN H2O)	0.0005	0.0517
FLOW RT (LB/HR)	300.00	700.00
EXCH WEIGHT (LB)	41.75	11.60
FIN LENGTH (IN)	12.000	12.000
LVR LENGTH (IN)	12.000	12.000
FIN HEIGHT (IN)	1.205	0.510
FIN THKNSS (IN)	0.015	0.015
NUMBER OF FINS	32.00	32.00
FIN EPSILON	0.87325	0.41184

STEADY-STATE SYSTEM POWER CONDITIONS
 VOLTAGE (VOLTS) 18.500
 CURRENT (AMPS) 62.929
 POWER (WATTS) 1164.193

SYSTEM PERFORMANCE DESCRIPTION DATE 01/20/77 1237
 54 TE UNITS MODEL NO. CX952-70-31
 WIRED AS 18 PARALLEL SETS OF 3 UNITS IN SERIES

THERMOELECTRIC DEVICE PHYSICAL DESCRIPTION
 HEIGHT (INCH)= 0.4300 WEIGHT (GRAM)= 21.129 RESISTANCE (OHM)= 1.579
 COLD M*CP PER TE DEVICE (WATT*SEC/C)= 0.55448E+02
 HOT M*CP PER TE DEVICE (WATT*SEC/C)= 0.26973E+03
 COLD, HOT, INTERSTAGE CONDUCTIVITY (WATT/SQ IN/C)= 6.3, 6.3, 20.0

T.E. MATERIAL	COMM	COMM
COUPLE DISTRIBUTION=	70.00	31.00
PELLET WIDTH (INCH)=	0.0550	0.0850
PELLET LENGTH (INCH)=	0.0630	0.1510
STAGE AREA (SQ INCH)=	1.4400	1.4400
ALUMINA THKNS (INCH)=	0.0300	0.0300
BUS-BAR THKNS (INCH)=	0.0150	0.0320

EXCHANGER FACTOR	HOT	COLD
FLUID TYPE	WATER	60%EG
PLATE TEMP (K)	302.40	265.39
INLET TEMP (K)	294.30	274.90
OUTLET TEMP (K)	300.07	265.74
HT OUT OF FLO(W)	-1520.96	539.50
PLT-PLT LOSS (W)	7.25	7.25
PRANDTL NUMBER	6.39313	95.86483
REYNOLDS NUMBER	64.16	3.82
NUSSELT NUMBER	6.4946	5.0260
HEAT TRANSFR COEF	129.3114	74.5067
PRSS DRP (IN H2O)	0.0994	1.3279
BEND FTR (IN H2O)	0.0013	0.0011
FLOW RT (L3/HR)	500.00	100.00
EXCH WEIGHT (LB)	41.75	11.60
FIN LENGTH (IN)	12.000	12.000
LVR LENGTH (IN)	12.000	12.000
FIN HEIGHT (IN)	1.205	0.510
FIN THKNSS (IN)	0.015	0.015
NUMBER OF FINS	32.00	32.00
FIN EPSILON	0.71173	0.96382

STEADY-STATE SYSTEM POWER CONDITIONS
 VOLTAGE (VOLTS) 18.500
 CURRENT (AMPS) 63.678
 POWER (WATTS) 1178.037

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SYSTEM PERFORMANCE DESCRIPTION DATE 01/21/77 1240
 54 TE UNITS MODEL NO. CX952-70-31
 WIRED AS 18 PARALLEL SETS OF 3 UNITS IN SERIES

THERMOELECTRIC DEVICE PHYSICAL DESCRIPTION
 HEIGHT (INCH)= 0.4300 WEIGHT (GRAM)= 21.129 RESISTANCE (OHM)= 1.579
 COLD M*CP PER TE DEVICE (WATT*SEC/C)= 0.5544E+02
 HOT M*CP PER TE DEVICE (WATT*SEC/C)= 0.26973E+03
 COLD, HOT, INTERSTAGE CONDUCTIVITY (WATT/SQ IN/C)= 6.3, 6.3, 20.0

T.E. MATERIAL	COMM	COMM
COUPLE DISTRIBUTION=	70.00	31.00
PELLET WIDTH (INCH)=	0.0550	0.0850
PELLET LENGTH (INCH)=	0.0630	0.1510
STAGE AREA (SQ INCH)=	1.4400	1.4400
ALUMINA THKNS (INCH)=	0.0300	0.0300
BUS-BAR THKNS (INCH)=	0.0150	0.0320

EXCHANGER FACTUM	HOT	COLD
FLUID TYPE	WATER	60XEG
PLATE TEMP (K)	302.47	267.71
INLET TEMP (K)	294.30	274.90
OUTLET TEMP (K)	300.11	268.48
HT OUT OF FLD(W)	-1532.90	357.82
PLT-PLT LOSS (W)	6.81	6.81
PRANDTL NUMBER	6.59057	90.27314
REYNOLDS NUMBER	64.18	6.10
NUSSELT NUMBER	6.4946	5.0941
HEAT TRANSFR COEF	129.3176	75.4551
PRSS DRP (IN H2O)	0.0994	1.8703
BEND FTR (IN H2O)	0.0013	0.0024
FLOW RT (LM/HR)	500.00	150.00
EXCH WEIGHT (LR)	41.75	11.60
FIN LENGTH (IN)	12.000	12.000
LVR LENGTH (IN)	12.000	12.000
FIN HEIGHT (IN)	1.205	0.510
FIN THKNSS (IN)	0.015	0.015
NUMBER OF FINS	32.00	32.00
FIN EPSILON	0.71174	0.89273

STEADY-STATE SYSTEM POWER CONDITIONS
 VOLTAGE (VOLTS) 18.500
 CURRENT (AMPS) 63.859
 POWER (WATTS) 1181.397

SYSTEM PERFORMANCE DESCRIPTION DATE 01/20/77 1237
 54 TE UNITS MODEL NO. CX952-70-31
 WIRED AS 18 PARALLEL SETS OF 3 UNITS IN SERIES

THERMOELECTRIC DEVICE PHYSICAL DESCRIPTION
 HEIGHT (INCH)= 0.4300 WEIGHT (GRAM)= 21.129 RESISTANCE (OHM)= 1.579
 COLD M*CP PER TE DEVICE (WATT*SEC/C)= 0.5544E+02
 HOT M*CP PER TE DEVICE (WATT*SEC/C)= 0.26973E+03
 COLD,HOT,INTERSTAGE CONDUCTIVITY (WATT/SQ IN/C)= 6.3, 6.3, 20.0

T.E. MATERIAL	COMM	COMM
COUPLE DISTRIBUTION=	70.00	31.00
PELLET WIDTH (INCH)=	0.0550	0.0850
PELLET LENGTH (INCH)=	0.0630	0.1510
STAGE AREA (SQ INCH)=	1.4400	1.4400
ALUMINA THKNS (INCH)=	0.0300	0.0300
BUS-BAR THKNS (INCH)=	0.0150	0.0320

EXCHANGER FACTOR	HOT	COLD
FLUID TYPE	WATER	60%EG
PLATE TEMP (K)	302.57	268.87
INLET TEMP (K)	294.30	274.90
OUTLET TEMP (K)	300.18	269.98
HT OUT OF FLD(W)	-1551.5A	366.00
PLT-PLT LOSS (W)	6.60	6.60
PRANDTL NUMBER	6.38367	87.39790
REYNOLDS NUMBER	64.24	8.42
NUSSELT NUMBER	6.4946	5.1607
HEAT TRANSF COEF	129.3325	76.4063
PRSS DRP(IN H2O)	0.0993	2.4110
BEND FR(IN H2O)	0.0013	0.0042
FLW RT (LR/HR)	500.00	200.00
EXCH WEIGHT (LB)	41.75	11.60
FIN LENGTH (IN)	12.000	12.000
LVR LENGTH (IN)	12.000	12.000
FIN HEIGHT (IN)	1.205	0.510
FIN THKNS (IN)	0.015	0.015
NUMBER OF FINS	32.00	32.00
FIN EPSILON	0.71176	0.81572

STEADY-STATE SYSTEM POWER CONDITIONS
 VOLTAGE (VOLTS) 18.500
 CURRENT (AMPS) 63.912
 POWER (WATTS) 1182.368

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SYSTEM PERFORMANCE DESCRIPTION DATE 01/20/77 1237
 S4 TE UNITS MODEL NO. CX952-70-31

WIRED AS 18 PARALLEL SETS OF 3 UNITS IN SERIES

THERMOELECTRIC DEVICE PHYSICAL DESCRIPTION

HEIGHT (INCH)= 0.4300 WEIGHT (GRAM)= 21.129 RESISTANCE (OHM)= 1.579
 COLD M*CP PER TE DEVICE (WATT*SEC/C)= 0.55448E+02
 HOT M*CP PER TE DEVICE (WATT*SEC/C)= 0.26973E+03
 COLD, HOT, INTERSTAGE CONDUCTIVITY (WATT/SQ IN/C)= 6.3, 6.3, 20.0

T.E. MATERIAL	COMM	COMM
COUPLE DISTRIBUTION=	70.00	31.00
PELLET WIDTH (INCH)=	0.0550	0.0850
PELLET LENGTH (INCH)=	0.0630	0.1510
STAGE AREA (SQ INCH)=	1.4400	1.4400
ALUMINA THKNS (INCH)=	0.0300	0.0300
BUS-BAR THKNS (INCH)=	0.0150	0.0320

EXCHANGER FACTOR	HOT	COLD
FLUID TYPE	WATER	60%EG
PLATE TEMP (K)	302.62	270.01
INLET TEMP (K)	294.30	274.90
OUTLET TEMP (K)	300.22	271.55
HT OUT OF FLD(W)	-1561.23	374.43
PLT-PLT LOSS (W)	6.39	6.39
PRANDTL NUMBER	6.38079	84.53804
REYNOLDS NUMBER	64.27	13.09
NUSSELT NUMBER	6.4946	5.2882
HEAT TRANSFR COEF	129.3389	78.2563
PRSS DRP(IN H2O)	0.0992	3.4949
BEND FR(IN H2O)	0.0013	0.0095
FLOW RT (LB/HK)	500.00	300.00
EXCH HEIGHT (LB)	41.75	11.60
FIN LENGTH (IN)	12.000	12.000
LVR LENGTH (IN)	12.000	12.000
FIN HEIGHT (IN)	1.205	0.510
FIN THKNSS (IN)	0.015	0.015
NUMBER OF FINS	32.00	32.00
FIN EPSILON	0.71177	0.68373

STEADY-STATE SYSTEM POWER CONDITIONS

VOLTAGE (VOLTS) 18.500
 CURRENT (AMPS) 63.989
 POWER (WATTS) 1183.792

SYSTEM PERFORMANCE DESCRIPTION DATE 01/20/77 1237
 54 TE UNITS MODEL NO. CX952-70-31
 WIRED AS 18 PARALLEL SETS OF 3 UNITS IN SERIES

TELEMOELECTRIC DEVICE PHYSICAL DESCRIPTION*
 HEIGHT (INCH)= 0.4300 WEIGHT (GRAM)= 21.129 RESISTANCE (OHM)= 1.579
 COLD M*CP PER TE DEVICE (WATT*SFC/C)= 0.55448E+02
 HOT M*CP PER TE DEVICE (WATT*SEC/C)= 0.26973E+03
 COLD, HOT, INTERSTAGE CONDUCTIVITY (WATT/SQ IN/C)= 6.3, 6.3, 20.0

T.E. MATERIAL	COMM	COMM
COUPLE DISTRIBUTION=	70.00	31.00
PELLET WIDTH (INCH)=	0.0550	0.0850
PELLET LENGTH (INCH)=	0.0630	0.1510
STAGE AREA (SQ INCH)=	1.4400	1.4400
ALUMINA THKNS (INCH)=	0.0300	0.0300
BUS-BAR THKNS (INCH)=	0.0150	0.0320

EXCHANGER FACTOR	HOT	COLD
FLUID TYPE	WATER	60%EG
PLATE TEMP (K)	302.66	270.91
INLET TEMP (K)	294.30	274.90
OUTLET TEMP (K)	300.25	272.86
HT OUT OF FLO(W)	-1569.40	381.23
PLT-PLT LOSS (W)	6.22	6.22
PRANDTL NUMBER	6.37821	82.25760
REYNOLDS NUMBER	64.29	22.46
NUSSELT NUMBER	6.4946	5.5222
HEAT TRANSF COEF	129.3447	81.6861
PRSS DMP(IN H2O)	0.0992	5.6695
BEND FTR(IN H2O)	0.0013	0.0264
FLOW K1 (LB/HR)	500.00	500.00
EXCH HEIGHT (LB)	41.75	11.60
FIN LENGTH (IN)	12.000	12.000
LVR LENGTH (IN)	12.000	12.000
FIN HEIGHT (IN)	1.205	0.510
FIN THKNSS (IN)	0.015	0.015
NUMBER OF FINS	32.00	32.00
FIN EPSILON	0.71174	0.51228

STEADY-STATE SYSTEM POWER CONDITIONS
 VOLTAGE (VOLTS) 18.500
 CURRENT (AMPS) 64.048
 POWER (WATTS) 1184.895

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SYSTEM PERFORMANCE DESCRIPTION DATE 01/20/77 1237
 54 TE UNITS MODEL NO. CX952-70-31
 WIRED AS 1A PARALLEL SETS OF 3 UNITS IN SERIES

THERMUELECTRIC DEVICE PHYSICAL DESCRIPTION
 HEIGHT (INCH)= 0.4300 WEIGHT (GRAM)= 21.129 RESISTANCE (OHM)= 1.579
 COLD M*CP PER TE DEVICE (WATT*SEC/C)= 0.55448E+02
 HOT M*CP PER TE DEVICE (WATT*SEC/C)= 0.26973E+03
 COLD,HOT,INTERSTAGE CONDUCTIVITY (WATT/SQ IN/C)= 6.3, 6.3, 20.0

T.E. MATERIAL	COMM	COMM
COUPLE DISTRIBUTION=	70.00	31.00
PELLET WIDTH (INCH)=	0.0550	0.0850
PELLET LENGTH (INCH)=	0.0630	0.1510
STAGE AREA (SQ INCH)=	1.4400	1.4400
ALUMINA THKNS (INCH)=	0.0300	0.0300
BUS-BAR THKNS (INCH)=	0.0150	0.0320

EXCHANGER FACTOR	HOT	COLD
FLUID TYPE	WATER	60XEG
PLATE TEMP (K)	302.68	271.33
INLET TEMP (K)	294.30	274.90
OUTLET TEMP (K)	300.27	273.43
HT OUT OF FLD(W)	-1572.99	384.51
PLT-PLT LOSS (W)	6.14	6.14
PRANDTL NUMBER	6.37709	81.28581
REYNOLDS NUMBER	64.30	31.84
NUSSELT NUMBER	6.4946	5.7335
HEAT TRANSFR COEF	129.3472	84.7954
PRSS DRP(IN H2O)	0.0992	7.8528
BEND FR(IN H2O)	0.0013	0.0517
FLOW RT (LH/HR)	500.00	700.00
EXCH WEIGHT (LB)	41.75	11.60
FIN LENGTH (IN)	12.000	12.000
LVR LENGTH (IN)	12.000	12.000
FIN HEIGHT (IN)	1.205	0.510
FIN THKNSS (IN)	0.015	0.015
NUMBER OF FINS	32.00	32.00
FIN EPSILON	0.71178	0.41186

STEADY-STATE SYSTEM POWER CONDITIONS
 VOLTAGE (VOLTS) 1A.500
 CURRENT (AMPS) 64.077
 POWER (WATTS) 1185.419

$$T_c \text{ inlet} = 70^\circ\text{F} = 294.3^\circ\text{K}$$

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SYSTEM PERFORMANCE DESCRIPTION DATE 01/20/77 1237
 54 TE UNITS MODEL NO. CX952-70-31
 ... WIRED AS 18 PARALLEL SETS OF 3 UNITS IN SERIES

THERMOELECTRIC DEVICE PHYSICAL DESCRIPTION
 HEIGHT (INCH)= 0.4300 WEIGHT (GRAM)= 21.129 RESISTANCE (OHM)= 1.579
 COLD M*CP PER TE DEVICE (WATT*SEC/C)= 0.55448E+02
 HOT M*CP PER TE DEVICE (WATT*SEC/C)= 0.26973E+03
 COLD, HOT, INTERSTAGE CONDUCTIVITY (WATT/SQ IN/C)= 6.3, 6.3, 20.0

T.E. MATERIAL	COMM	COMM
COUPLE DISTRIBUTION=	70.00	31.00
PELLET WIDTH (INCH)=	0.0550	0.0850
PELLET LENGTH (INCH)=	0.0630	0.1510
STAGE AREA (SQ INCH)=	1.4400	1.4400
ALUMINA THKNS (INCH)=	0.0300	0.0300
BUS-BAR THKNS (INCH)=	0.0150	0.0320

EXCHANGER FACTOR	HOT	COLD
FLUID TYPE	WATER	60%EG
PLATE TEMP (K)	321.47	284.20
INLET TEMP (K)	294.30	294.30
OUTLET TEMP (K)	321.42	284.63
HT OUT OF FLD(A)	-1430.31	373.32
PLT-PLT LOSS (K)	7.30	7.30
PRANDTL NUMBER	4.95492	46.45921
REYNOLDS NUMBER	16.06	8.31
NUSSELT NUMBER	6.3438	5.0371
HEAT TRANSFER COEF	130.1722	73.7017
PRSS DRP (IN H2O)	0.0158	0.6158
HEAD FTR (IN H2O)	0.0001	0.0011
FLUX HT (LH/HK)	100.00	100.00
EXCH WEIGHT (LB)	41.75	11.60
FIN LENGTH (IN)	12.000	12.000
LVR LENGTH (IN)	12.000	12.000
FIN HEIGHT (IN)	1.205	0.510
FIN THKNSS (IN)	0.015	0.015
NUMBER OF FINS	32.00	32.00
FIN EPSILON	0.99806	0.95727

STEADY-STATE SYSTEM POWER CONDITIONS
 VOLTAGE (VOLTS) 18.500
 CURRENT (AMPS) 57.603
 POWER (WATTS) 1065.647

SYSTEM PERFORMANCE DESCRIPTION DATE 01/21/77 1240
 54 TE UNITS MODEL NO. CX952-70-31
 WIRED AS 18 PARALLEL SETS OF 3 UNITS IN SERIES

THEMEOELECTRIC DEVICE PHYSICAL DESCRIPTION
 HEIGHT (INCH)= 0.4300 WEIGHT (GRAM)= 21.129 RESISTANCE (OHM)= 1.579
 COLD M*CP PER TE DEVICE (WATT*SEC/C)= 0.55448E+02
 HOT M*CP PER TE DEVICE (WATT*SEC/C)= 0.26973E+03
 COLD, HOT, INTERSTAGE CONDUCTIVITY (WATT/SQ IN/C)= 6.3, 6.3, 20.0

T.E. MATERIAL	COMM	COMM
COUPLE DISTRIBUTION=	70.00	31.00
PELLET WIDTH (INCH)=	0.0550	0.0850
PELLET LENGTH (INCH)=	0.0630	0.1510
STAGE AREA (SQ INCH)=	1.4400	1.4400
ALUMINA THKNS (INCH)=	0.0300	0.0300
BUS-BAR THKNS (INCH)=	0.0150	0.0320

EXCHANGER FACTOR	HOT	COLD
FLUID TYPE	WATER	60XEG
PLATE TEMP (K)	321.62	286.64
INLET TEMP (K)	294.30	294.30
OUTLET TEMP (K)	321.57	287.56
HT OUT OF FLD(K)	-1438.09	391.44
PLT-PLT LOSS (K)	6.85	6.85
PRANDTL NUMBER	4.94702	44.32809
REYNOLDS NUMBER	16.09	13.12
NUSSELT NUMBER	6.3438	5.1121
HEAT TRANSF COEF	130.1977	74.7098
PRSS DRP (IN H2O)	0.0157	0.8792
BEND FR (IN H2O)	0.0001	0.0024
FLOW RT (LB/HR)	100.00	150.00
EXCH WEIGHT (LB)	41.75	11.60
FIN LENGTH (IN)	12.000	12.000
LVR LENGTH (IN)	12.000	12.000
FIN HEIGHT (IN)	1.205	0.510
FIN THKNSS (IN)	0.015	0.015
NUMBER OF FINS	32.00	32.00
FIN EPSILON	0.99806	0.88027

STEADY-STATE SYSTEM POWER CONDITIONS
 VOLTAGE (VOLTS) 18.500
 CURRENT (AMPS) 57.770
 POWER (WATTS) 1068.753

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SYSTEM PERFORMANCE DESCRIPTION DATE 01/21/77 1109
 54 TE UNITS MODEL NO. CX952-70-31
 WIRED AS 14 PARALLEL SETS OF 3 UNITS IN SERIES

THERMOELECTRIC DEVICE PHYSICAL DESCRIPTION
 HEIGHT (INCH)= 0.4300 WEIGHT (GRAM)= 21.129 RESISTANCE (OHM)= 1.579
 COLD M*CP PER TE DEVICE (WATT*SEC/C)= 0.55448E+02
 HOT M*CP PER TE DEVICE (WATT*SEC/C)= 0.26973E+03
 COLD,HOT, INTERSTAGE CONDUCTIVITY (WATT/SQ IN/C)= 6.3, 6.3, 20.0

T.E. MATERIAL	COMM	COMM
COUPLE DISTRIBUTION=	70.00	31.00
PELLET WIDTH (INCH)=	0.0550	0.0850
PELLET LENGTH (INCH)=	0.0630	0.1510
STAGE AREA (SQ INCH)=	1.4400	1.4400
ALUMINA THKNS (INCH)=	0.0300	0.0300
BUS-BAR THKNS (INCH)=	0.0150	0.0320

EXCHANGER FACTOR	HOT	COLD
FLUID TYPE	WATER	50%EG
PLATE TEMP (K)	321.73	287.86
INLET TEMP (K)	294.30	294.30
OUTLET TEMP (K)	321.68	289.14
HT OUT OF FLD(W)	-1443.79	399.98
PLT-PLT LOSS (W)	6.63	6.63
PRANDTL NUMBER	4.94102	43.23466
REYNOLDS NUMBER	16.10	17.98
NUSSELT NUMBER	6.3438	5.1856
HEAT TRANSF COEF	130.2172	75.7349
PRSS DRP(IN H2O)	0.0157	1.1423
SENO FTR(IN H2O)	0.0001	0.0043
FLOW RT (LW/MW)	100.00	200.00
EXCH WEIGHT (LB)	41.75	11.60
FIN LENGTH (IN)	12.000	12.000
LVR LENGTH (IN)	12.000	12.000
FIN HEIGHT (IN)	1.205	0.510
FIN THKNS (IN)	0.015	0.015
NUMBER OF FINS	32.00	32.00
FIN EPSILON	0.99806	0.80008

STEADY-STATE SYSTEM POWER CONDITIONS
 VOLTAGE (VOLTS) 18.500
 CURRENT (AMPS) 57.839
 POWER (WATTS) 1070.018

SYSTEM PERFORMANCE DESCRIPTION DATE 01/21/77 1240
 54 TE UNITS MODEL NO. CX952-70-31
 WIRED AS 18 PARALLEL SETS OF 3 UNITS IN SERIES

THEMEOELECTRIC DEVICE PHYSICAL DESCRIPTION
 HEIGHT (INCH)= 0.4300 HEIGHT (GRAM)= 21.129 RESISTANCE (OHM)= 1.579
 COLD M*CP PER TE DEVICE (WATT*SEC/C)= 0.55448E+02
 HOT M*CP PER TE DEVICE (WATT*SEC/C)= 0.26973E+03
 COLD, HOT, INTERSTAGE CONDUCTIVITY (WATT/SQ IN/C)= 6.3, 6.3, 20.0

T.E. MATERIAL	COMM	COMM
COUPLE DISTRIBUTION=	70.00	31.00
PELLET WIDTH (INCH)=	0.0550	0.0850
PELLET LENGTH (INCH)=	0.0630	0.1510
STAGE AREA (SQ INCH)=	1.4400	1.4400
ALUMINA THKNS (INCH)=	0.0300	0.0300
BUS-BAR THKNS (INCH)=	0.0150	0.0320

EXCHANGER FACTOR	HOT	COLD
FLUID TYPE	WATER	60%EG
PLATE TEMP (K)	321.88	289.05
INLET TEMP (K)	294.30	294.30
OUTLET TEMP (K)	321.82	290.80
HT OUT OF FLD(W)	-1451.62	408.02
PLT-PLT LOSS (W)	6.43	6.43
PRANDTL NUMBER	4.93279	42.13168
REYNOLDS NUMBER	16.13	27.74
NUSSELT NUMBER	6.3438	5.3271
HEAT TRANSFER COEF	130.2440	77.7485
PRSS DRP(IN H2O)	0.0157	1.6697
BEND FR(IN H2O)	0.0001	0.0096
FLOW RT (LH/HK)	100.00	300.00
EXCH WEIGHT (LB)	41.75	11.60
FIN LENGTH (IN)	12.000	12.000
LVR LENGTH (IN)	12.000	12.000
FIN HEIGHT (IN)	1.205	0.510
FIN THKNSS (IN)	0.015	0.015
NUMBER OF FINS	32.00	32.00
FIN EPSILON	0.99806	0.66651

STEADY-STATE SYSTEM POWER CONDITIONS
 VOLTAGE (VOLTS) 18.500
 CURRENT (AMPS) 57.890
 POWER (WATTS) 1070.963

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SYSTEM PERFORMANCE DESCRIPTION DATE 01/21/77 1240
 54 TE UNITS MODEL NO. CX952-70-31
 WIRED AS 1A PARALLEL SETS OF 3 UNITS IN SERIES

THERMOELECTRIC DEVICE PHYSICAL DESCRIPTION
 HEIGHT (INCH)= 0.4300 WHEIGHT (GRAM)= 21.129 RESISTANCE (OHM)= 1.579
 COLD M*CP PER TE DEVICE (WATT*SEC/C)= 0.5544E+02
 HOT M*CP PER TE DEVICE (WATT*SEC/C)= 0.26973E+03
 COLD, HOT, INTERSTAGE CONDUCTIVITY (WATT/SQ IN/C)= 6.3, 6.3, 20.0

I.E. MATERIAL	COMM	COMM
COUPLE DISTRIBUTION=	70.00	31.00
PELLET WIDTH (INCH)=	0.0550	0.0850
PELLET LENGTH (INCH)=	0.0630	0.1510
STAGE AREA (SQ INCH)=	1.4400	1.4400
ALUMINA THKNS (INCH)=	0.0300	0.0300
BUS-BAR THKNS (INCH)=	0.0150	0.0320

EXCHANGER FACTOR	HOT	COLD
FLUID TYPE	WATER	60%EG
PLATE TEMP (K)	321.89	290.01
INLET TEMP (K)	294.30	294.30
OUTLET TEMP (K)	321.83	292.17
HT OUT OF FLO(W)	1452.13	415.33
PLT-PLT LOSS (%)	6.24	6.24
PRANDTL NUMBER	4.93226	41.25178
REYNOLDS NUMBER	16.13	47.32
NUSSELT NUMBER	6.3438	5.5888
HEAT TRANSFR COEF	130.2457	81.5211
PRSS DRP(IN H2O)	0.0157	2.7308
BEND FTR(IN H2O)	0.0001	0.0266
FLOW RT (LP/HR)	100.00	500.00
EXCH WEIGHT (LB)	41.75	11.60
FIN LENGTH (IN)	12.000	12.000
LVR LENGTH (IN)	12.000	12.000
FIN HEIGHT (IN)	1.205	0.510
FIN THKNSS (IN)	0.015	0.015
NUMBER OF FINS	32.00	32.00
FIN EPSILON	0.99806	0.49736

STEADY-STATE SYSTEM POWER CONDITIONS
 VOLTAGE (VOLTS) 18.500
 CURRENT (AMPS) 57.973
 POWER (WATTS) 1072.504

SYSTEM PERFORMANCE DESCRIPTION DATE 01/21/77 1240
 54 TE UNITS MODEL NO. CX952-70-31
 WIRED AS 18 PARALLEL SETS OF 3 UNITS IN SERIES

THERMUELECTRIC DEVICE PHYSICAL DESCRIPTION
 HEIGHT (INCH)= 0.4300 WEIGHT (GRAM)= 21.129 RESISTANCE (OHM)= 1.579
 COLD M*CP PER TE DEVICE (WATT*SEC/C)= 0.55448E+02
 HOT M*CP PER TE DEVICE (WATT*SEC/C)= 0.26973E+03
 COLD, HOT, INTERSTAGE CONDUCTIVITY (WATT/SQ IN/C)= 6.3, 6.3, 20.0

I.E. MATERIAL	COMM	COMM
COUPLE DISTRIBUTION=	70.00	31.00
PELLET WIDTH (INCH)=	0.0550	0.0850
PELLET LENGTH (INCH)=	0.0630	0.1510
STAGE AREA (SQ INCH)=	1.4400	1.4400
ALUMINA THKNS (INCH)=	0.0300	0.0300
BUS-BAR THKNS (INCH)=	0.0150	0.0320

EXCHANGER FACTOR	HOT	COLD
FLUID TYPE	WATER	60XEG
PLATE TEMP (K)	321.96	290.46
INLET TEMP (K)	294.30	294.30
OUTLET TEMP (K)	321.90	292.77
HT OUT OF FLD (W)	1455.75	418.27
PLT-PLT LOSS (W)	6.17	6.17
PRANDTL NUMBER	4.42846	40.87334
REYNOLDS NUMBER	16.14	66.91
NUSSELT NUMBER	6.3438	5.8267
HEAT TRANSF COEFF	150.2581	84.9683
PRSS DRP (IN H2O)	0.0157	3.8001
BEND FTR (IN H2O)	0.0001	0.0522
FLOW RT (LR/HK)	100.00	700.00
EXCH WEIGHT (LB)	41.75	11.60
FIN LENGTH (IN)	12.000	12.000
LVR LENGTH (IN)	12.000	12.000
FIN HEIGHT (IN)	1.205	0.510
FIN THKNSS (IN)	0.015	0.015
NUMBER OF FINS	32.00	32.00
FIN EPSILON	0.99806	0.39978

STEADY-STATE SYSTEM POWER CONDITIONS
 VOLTAGE (VOLTS) 14.500
 CURRENT (AMPS) 57.988
 POWER (WATTS) 1072.776

SYSTEM PERFORMANCE DESCRIPTION DATE 01/20/77 1237
 54 IE UNITS MODEL NO. CX952-70-31
 WIRED AS 14 PARALLEL SETS OF 3 UNITS IN SERIES

THERMOELECTRIC DEVICE PHYSICAL DESCRIPTION
 HEIGHT (INCH)= 0.4300 WEIGHT (GRAM)= 21.129 RESISTANCE (OHM)= 1.579
 COLD M*CP PER IE DEVICE (WATT*SEC/C)= 0.55448E+02
 HOT M*CP PER IE DEVICE (WATT*SEC/C)= 0.26973E+03
 COLD, HOT, INTERSTAGE CONDUCTIVITY (WATT/SQ IN/C)= 6.3, 6.3, 20.0

T.E. MATERIAL	COMM	COMM
COUPLE DISTRIBUTION=	70.00	31.00
PELLET WIDTH (INCH)=	0.0550	0.0850
PELLET LENGTH (INCH)=	0.0630	0.1510
STAGE AREA (SQ INCH)=	1.4400	1.4400
ALUMINA THKNS (INCH)=	0.0300	0.0300
BUS-BAR THKNS (INCH)=	0.0150	0.0320

EXCHANGER FACTOR	HOT	COLD
FLUID TYPE	WATER	60XEG
PLATE TEMP (K)	306.08	282.14
INLET TEMP (K)	294.30	294.30
OUTLET TEMP (K)	304.59	282.65
HT OUT OF FLO(W)	-1627.70	448.75
PLT-PLT LOSS (A)	4.69	4.69
PRANDTL NUMBER	6.04052	47.97960
REYNOLDS NUMBER	40.48	8.03
NUSSELT NUMBER	6.4171	5.0366
HEAT TRANSF COEF	128.5457	73.7506
PRSS DROP (IN H2O)	0.0565	0.6374
BEND FTR (IN H2O)	0.0005	0.0011
FLOW RT (L/H/HR)	300.00	100.00
EXCH WEIGHT (LB)	41.75	11.60
FIN LENGTH (IN)	12.000	12.000
LVR LENGTH (IN)	12.000	12.000
FIN HEIGHT (IN)	1.205	0.510
FIN THKNSS (IN)	0.015	0.015
NUMBER OF FINS	32.00	32.00
FIN EPSILON	0.47337	0.95763

STEADY-STATE SYSTEM POWER CONDITIONS
 VOLTAGE (VOLTS) 18.500
 CURRENT (AMPS) 63.586
 POWER (WATTS) 1176.341

SYSTEM PERFORMANCE DESCRIPTION DATE 01/21/77 1240
 54 TE UNITS MODEL NO. CX952-70-31
 WIRED AS 18 PARALLEL SETS OF 3 UNITS IN SERIES

THERMUELECTRIC DEVICE PHYSICAL DESCRIPTION
 HEIGHT (INCH)= 0.4300 WEIGHT (GRAM)= 21.129 RESISTANCE (OHM)= 1.579
 COLD M*CP PER TE DEVICE (WATT*SEC/C)= 0.55448E+02
 HOT M*CP PER TE DEVICE (WATT*SEC/C)= 0.26973E+03
 COLD, HOT, INTERSTAGE CONDUCTIVITY (WATT/SQ IN/C)= 6.3, 6.3, 20.0

T.E. MATERIAL	COMM	COMM
COUPLE DISTRIBUTION=	70.00	31.00
PELLET WIDTH (INCH)=	0.0550	0.0450
PELLET LENGTH (INCH)=	0.0630	0.1510
STAGE AREA (SQ INCH)=	1.4400	1.4400
ALUMINA THKNS (INCH)=	0.0300	0.0300
BUS-BAR THKNS (INCH)=	0.0150	0.0320

EXCHANGER FACTOR	HOT	COLD
FLUID TYPE	WATER	60%EG
PLATE TEMP (K)	306.18	285.07
INLET TEMP (K)	294.30	294.30
OUTLET TEMP (K)	304.58	286.17
HT OUT OF FLD(W)	-1641.70	471.44
PLT-PLT LOSS (W)	4.13	4.13
PRAYUTL NUMREN	6.03497	45.32122
REYNOLDS NUMBER	40.51	12.81
NUSSELT NUMBER	6.4171	5.1114
HEAT TRANSFR COEF	128.6091	74.7425
PRSS DRP(IN H2O)	0.0565	0.9003
3END FTR(IN H2O)	0.0005	0.0024
FLOW RT (LB/HR)	300.00	150.00
EXCH WEIGHT (LB)	41.75	11.60
FIN LENGTH (IN)	12.000	12.000
LVR LENGTH (IN)	12.000	12.000
FIN HEIGHT (IN)	1.205	0.510
FIN THKNSS (IN)	0.015	0.015
NUMBER OF FINS	32.00	32.00
FIN EPSILON	0.87338	0.86074

STEADY-STATE SYSTEM POWER CONDITIONS
 VOLTAGE (VOLTS) 18.500
 CURRENT (AMPS) 63.804
 POWER (WATTS) 1180.380

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SYSTEM PERFORMANCE DESCRIPTION DATE 01/21/77 1109
 54 IE UNITS MODEL NO. CX952-70-31
 WIRING AS 18 PARALLEL SETS OF 3 UNITS IN SERIES

THERMoeLECTRIC DEVICE PHYSICAL DESCRIPTION
 HEIGHT (INCH)= 0.4300 HEIGHT (MM)= 21.129 RESISTANCE (OHM)= 1.579
 COLD HPC PER IE DEVICE (WATT*SEC/C)= 0.55444E+02
 HOT HPC PER IE DEVICE (WATT*SEC/C)= 0.26973E+03
 COLD,HOT,INTERSTAGE CONDUCTIVITY (WATT/SQ IN/C)= 6.3, 6.3, 20.0

	T.E. MATERIAL	COMM	COMM
	COUPLE DISTANCE (INCH)=	70.00	31.00
	PELLET WIDTH (INCH)=	0.0550	0.0450
	PELLET LENGTH (INCH)=	0.0630	0.1510
-----	STAGE AREA (SQ INCH)=	1.4400	1.4400
	ALUMINA THKNS (INCH)=	0.0300	0.0300
	SUS-BAR THKNS (INCH)=	0.0150	0.0320

	HOT	COLD
EXCHANGER FACTOR		
FLUID TYPE	WATER	60%EG

PLATE TEMP (K)	306.27	286.53
INLET TEMP (K)	294.30	294.30
OUTLET TEMP (K)	304.75	288.08
HT OUT OF FLD (K)	-1654.08	481.99
PLT-PLT LOSS (K)	3.46	3.85
PRANDTL NUMBER	6.02414	43.96572
REYNOLDS NUMBER	40.45	17.65
NUSSELT NUMBER	6.4171	5.1849
HEAT TRANSFER COEF	125.5232	75.7582
PRSS DRP (IN H2O)	0.0564	1.1630
SEND FTR (IN H2O)	0.0005	0.0043
FLOW RT (L2/HK)	400.00	200.00
EXCH HEIGHT (LR)	41.75	11.60
FIN LENGTH (IN)	12.000	12.000
LVR LENGTH (IN)	12.000	12.000
FIN HEIGHT (IN)	1.205	0.510
FIN THKNS (IN)	0.015	0.015
NUMBER OF FINS	32.00	32.00
FIN EPSILON	0.87340	0.80053

STEADY-STATE SYSTEM POWER CONDITIONS
 VOLTAGE (VOLTS) 18.500
 CURRENT (AMPS) 63.493
 POWER (WATTS) 1182.029

SYSTEM PERFORMANCE DESCRIPTION DATE 01/21/77 1109
 54 TE UNITS MODEL NO. CX952-70-31
 WIRED AS 18 PARALLEL SETS OF 3 UNITS IN SERIES

THERMOELECTRIC DEVICE PHYSICAL DESCRIPTION
 HEIGHT (INCH)= 0.4300 WEIGHT (GRAM)= 21.129 RESISTANCE (OHM)= 1.579
 COLD M*CP PER TE DEVICE (WATT*SEC/C)= 0.55448E+02
 HOT M*CP PER TE DEVICE (WATT*SEC/C)= 0.26973E+03
 COLD, HOT, INTERSTAGE CONDUCTIVITY (WATT/SQ IN/C)= 6.3, 6.3, 20.0

T.E. MATERIAL	COMM	COMM
COUPLE DISTRIBUTION=	70.00	31.00
PELLET WIDTH (INCH)=	0.0550	0.0850
PELLET LENGTH (INCH)=	0.0630	0.1510
STAGE AREA (SQ INCH)=	1.4400	1.4400
ALUMINA THKNS (INCH)=	0.0300	0.0300
BUS-BAR THKNS (INCH)=	0.0150	0.0320

EXCHANGER FACTOR	HOT	COLD
FLUID TYPE	WATER	50%EG
PLATE TEMP (K)	306.36	287.96
INLET TEMP (K)	294.30	294.30
OUTLET TEMP (K)	304.84	290.07
HT OUT OF FLD(K)	-1667.23	492.25
PLT-PLT LOSS (W)	3.60	3.60
PRANDTL NUMBER	6.02295	42.61038
REYNOLDS NUMBER	40.58	27.40
NUSSELT NUMBER	6.4171	5.3264
HEAT TRANSFER COEF	128.6582	77.7616
PRSS DROP (IN H2O)	0.0564	1.6900
3END FTR (IN H2O)	0.0005	0.0096
FLOW RT (L2/HR)	300.00	300.00
EXCH WEIGHT (LB)	41.75	11.60
FIN LENGTH (IN)	12.000	12.000
LVR LENGTH (IN)	12.000	12.000
FIN HEIGHT (IN)	1.205	0.510
FIN THKNSS (IN)	0.015	0.015
NUMBER OF FINS	52.00	32.00
FIN EPSILON	0.47342	0.66685

STEADY-STATE SYSTEM POWER CONDITIONS

VOLTAGE (VOLTS)	18.500
CURRENT (AMPS)	63.978
POWER (WATTS)	1183.590

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SYSTEM PERFORMANCE DESCRIPTION DATE 01/21/77 1240
 54 TE UNITS MODEL NO. CX952-70-31
 WIRED AS 18 PARALLEL SETS OF 3 UNITS IN SERIES

THEMEOELECTRIC DEVICE PHYSICAL DESCRIPTION
 HEIGHT (INCH)= 0.4300 WEIGHT (GRAM)= 21.129 RESISTANCE (OHM)= 1.579
 COLD M*CP PER TE DEVICE (WATT*SEC/C)= 0.55448E+02
 HOT M*CP PER TE DEVICE (WATT*SEC/C)= 0.26973E+03
 COLD, HOT, INTERSTAGE CONDUCTIVITY (WATT/SQ IN/C)= 6.3, 6.3, 20.0

T.E. MATERIAL	COMM	COMM
COUPLE DISTRIBUTION=	70.00	31.00
PELLET WIDTH (INCH)=	0.0550	0.0850
PELLET LENGTH (INCH)=	0.0630	0.1510
STAGE AREA (SQ INCH)=	1.4400	1.4400
ALUMINA THKNS (INCH)=	0.0300	0.0300
BUS-BAR THKNS (INCH)=	0.0150	0.0320

EXCHANGER FACTOR	HOT	COLD
FLUID TYPE	WATER	60%EG
PLATE TEMP (K)	306.39	289.12
INLET TEMP (K)	294.30	294.30
OUTLET TEMP (K)	304.86	291.72
HT OUT OF FLD (K)	-1670.86	501.12
PLT-PLT LOSS (K)	5.38	3.38
PRANDTL NUMBER	6.02124	41.53388
REYNOLDS NUMBER	40.59	46.96
NUSSÉLT NUMBER	6.4171	5.5881
HEAT TRANSF COEF	124.6423	41.5254
PRSS DRP (IN H2O)	0.0564	2.7508
BEND FTR (IN H2O)	0.0005	0.0266
FLOW RT (LB/HK)	300.00	500.00
EXCH WEIGHT (LB)	41.75	11.60
FIN LENGTH (IN)	12.000	12.000
LVR LENGTH (IN)	12.000	12.000
FIN HEIGHT (IN)	1.205	0.510
FIN THKNSS (IN)	0.015	0.015
NUMBER OF FINS	32.00	32.00
FIN EPSILON	0.47342	0.49754

STEADY-STATE SYSTEM POWER CONDITIONS
 VOLTAGE (VOLTS) 18.500
 CURRENT (AMPS) 64.067
 POWER (WATTS) 1185.237

SYSTEM PERFORMANCE DESCRIPTION DATE 01/21/77 1240
54 TE UNITS MODEL NO. CX952-70-31
WIRED AS 18 PARALLEL SETS OF 3 UNITS IN SERIES

THEMEOELECTRIC DEVICE PHYSICAL DESCRIPTION
HEIGHT (INCH)= 0.4300 WEIGHT (GRAM)= 21.129 RESISTANCE (OHM)= 1.579
COLD M*CP PER TE DEVICE (WATT*SEC/C)= 0.55448E+02
HOT M*CP PER TE DEVICE (WATT*SEC/C)= 0.26973E+03
COLD, HOT, INTERSTAGE CONDUCTIVITY (WATT/SQ IN/C)= 6.3, 6.3, 20.0

T.E. MATERIAL	COMM	COMM
COUPLE DISTRIBUTION=	70.00	31.00
PELLET WIDTH (INCH)=	0.0550	0.0850
PELLET LENGTH (INCH)=	0.0630	0.1510
STAGE AREA (SQ INCH)=	1.4400	1.4400
ALUMINA THKNS (INCH)=	0.0300	0.0300
BUS-BAR THKNS (INCH)=	0.0150	0.0320

EXCHANGER FACTOR	HOT	COLD
FLUID TYPE	WATER	60%EG
PLATE TEMP (K)	306.42	289.66
INLET TEMP (K)	294.30	294.30
OUTLET TEMP (K)	304.89	292.45
HT OUT OF FLD(W)	-1675.27	505.06
PLT-PLT LOSS (A)	3.28	3.28
PRANDTL NUMBER	6.01917	41.07396
REYNOLDS NUMBER	40.61	66.56
NUSSELT NUMBER	6.4171	5.8259
HEAT TRANSF COEFF	124.6474	84.9689
PRSS DRP(IN H2O)	0.0563	3.8199
BEND FTK(IN H2O)	0.0005	0.0522
FLUX RT (LB/HK)	500.00	700.00
EXCH WEIGHT (LB)	41.75	11.60
FIN LENGTH (IN)	12.000	12.000
LVR LENGTH (IN)	12.000	12.000
FIN HEIGHT (IN)	1.205	0.510
FIN THKNSS (IN)	0.015	0.015
NUMBER OF FINS	32.00	32.00
FIN EPSILON	0.87343	0.39989

STEADY-STATE SYSTEM POWER CONDITIONS
VOLTAGE (VOLTS) 19.500
CURRENT (AMPS) 64.101
POWER (WATTS) 1185.861

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SYSTEM PERFORMANCE DESCRIPTION DATE 01/21/77 1109
 54 TE UNITS MODEL NO. CX952-70-31
 JOINED AS 14 PARALLEL SETS OF 3 UNITS IN SERIES

THERMOELECTRIC DEVICE PHYSICAL DESCRIPTION
 HEIGHT (INCH)= 0.4300 WEIGHT (GRAM)= 21.129 RESISTANCE (OHM)= 1.579
 COLD *CP PER TE DEVICE (WATT*SEC/C)= 0.55448E+02
 HOT *CP PER TE DEVICE (WATT*SEC/C)= 0.26973E+03
 COLD, HOT, INTERSTAGE CONDUCTIVITY (WATT/SQ IN/C)= 6.3, 6.3, 20.0

T.F. MATERIAL	COMM	COMM
COUPLE DISTRIBUTION=	70.00	31.00
PELLET WIDTH (INCH)=	0.0550	0.0850
PELLET LENGTH (INCH)=	0.0630	0.1510
STAGE AREA (SQ INCH)=	1.4400	1.4400
ALUMINA THKNS (INCH)=	0.0300	0.0300
BUS-BAR THKNS (INCH)=	0.0150	0.0320

	HOT	COLD
EXCHANGER FACTOR		
FLUID TYPE	WATER	60%EG
PLATE TEMP (K)	303.15	281.77
INLET TEMP (K)	294.30	294.30
OUTLET TEMP (K)	300.60	282.30
IT OUT OF FLO(W)	-1562.24	462.10
PLT=PLT LOSS (W)	4.19	4.19
PRANDTL NUMBER	6.34998	48.24938
REYNOLDS NUMBER	64.54	7.98
NUSSELT NUMBER	6.4945	5.0365
HEAT TRANSFR COEF	129.4079	73.7590
PRESS DRP(IN H2O)	0.0988	0.6412
LEND FTR(IN H2O)	0.0013	0.0011
FLOW RT (LB/HR)	500.00	100.00
EXCH WEIGHT (LB)	41.75	11.60
FIN LENGTH (IN)	12.000	12.000
LVR LENGTH (IN)	12.000	12.000
FIN HEIGHT (IN)	1.205	0.510
FIN THKNSS (IN)	0.015	0.015
NUMBER OF FINS	32.00	32.00
FIN EPSILON	0.71188	0.95769

STEADY-STATE SYSTEM POWER CONDITIONS
 VOLTAGE (VOLTS) 18.500
 CURRENT (AMPS) 64.779
 POWER (WATTS) 1198.414

SYSTEM PERFORMANCE DESCRIPTION DATE 01/21/77 1240
 SEITE UNITS MODEL NO. CX952-70-31
 WIRED AS 1A PARALLEL SFTS OF 3 UNITS IN SERIES

THERMOELECTRIC DEVICE PHYSICAL DESCRIPTION
 HEIGHT (INCH)= 0.4300 WEIGHT (GRAM)= 21.129 RESISTANCE (OHM)= 1.579
 COLD M*CP PER TE DEVICE (WATT*SEC/C)= 0.55448E+02
 HOT M*CP PER TE DEVICE (WATT*SEC/C)= 0.26973E+03
 COLD, HOT, INTERSTAGE CONDUCTIVITY (WATT/SQ IN/C)= 6.3, 6.3, 20.0

T.E. MATERIAL	COMM	COMM
COUPLE DISTRIBUTION=	70.00	31.00
PELLET WIDTH (INCH)=	0.0550	0.0850
PELLET LENGTH (INCH)=	0.0630	0.1510
STAGE AREA (SQ INCH)=	1.4400	1.4400
ALUMINA THKNS (INCH)=	0.0300	0.0300
BUS-BAR THKNS (INCH)=	0.0150	0.0320

EXCHANGER FACTOR	HOT	COLD
FLUID TYPE	WATER	60%EG
PLATE TEMP (K)	403.24	284.78
INLET TEMP (K)	294.30	294.30
OUTLET TEMP (K)	300.67	285.92
HT OUT OF FLD(W)	-1478.76	486.11
PLT-PLT LOSS (W)	3.61	3.61
PRANDTL NUMBER	6.34558	45.50531
REYNOLDS NUMBER	64.58	12.75
NUSSELT NUMBER	6.4945	5.1113
HEAT TRANSF COEF	129.4178	74.7484
PRSS DWP(IN H2O)	0.0948	0.9042
3END FTR(IN H2O)	0.0013	0.0024
FLOW RT (LH/HR)	500.00	150.00
EXCH WEIGHT (LB)	41.75	11.60
FIN LENGTH (IN)	12.000	12.000
LVR LENGTH (IN)	12.000	12.000
FIN HEIGHT (IN)	1.205	0.510
FIN THKNSS (IN)	0.015	0.015
NUMBER OF FINS	32.00	32.00
FIN EPSILON	0.71190	0.88083

STEADY-STATE SYSTEM POWER CONDITIONS
 VOLTAGE (VOLTS) 14.500
 CURRENT (AMPS) 65.004
 POWER (WATTS) 1202.575

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SYSTEM PERFORMANCE DESCRIPTION DATE 01/21/77 1109
54 TE UNITS MODEL NO. CX952-70-31
WIRED AS 1A PARALLEL SETS OF 3 UNITS IN SERIES

THERMOELECTRIC DEVICE PHYSICAL DESCRIPTION
HEIGHT (INCH)= 0.4300 WEIGHT (GRAM)= 21.129 RESISTANCE (OHM)= 1.579
COLD M*CP PER TE DEVICE (WATT*SEC/C)= 0.55448E+02
HOT M*CP PER TE DEVICE (WATT*SEC/C)= 0.26973E+03
COLD, HOT, INTERSTAGE CONDUCTIVITY (WATT/SQ IN/C)= 4.3, 6.3, 20.0

T.E. MATERIAL	COMM	COMM
COUPLE DISTRIBUTION=	70.00	31.00
PELLET WIDTH (INCH)=	0.0550	0.0850
PELLET LENGTH (INCH)=	0.0630	0.1510
STAGE AREA (SQ INCH)=	1.4400	1.4400
ALUMINA THKNS (INCH)=	0.0300	0.0300
BUS-BAR THKNS (INCH)=	0.0150	0.0320

EXCHANGER FACTOR	HOT	COLD
FLUID TYPE	WATER	60%EG
PLATE TEMP (K)	303.32	286.28
INLET TEMP (K)	294.30	294.30
OUTLET TEMP (K)	300.72	287.88
HT OUT OF FLO(W)	-1692.93	497.42
PLT-PLT LOSS (W)	3.34	3.34
PRANDTL NUMBER	6.34126	44.10556
REYNOLDS NUMBER	64.62	17.59
NUSSELT NUMBER	4.4945	5.1848
HEAT TRANSF COEFF	129.4275	75.7625
PRSS DRP(IN H2O)	0.0987	1.1669
3END FRIC(IN H2O)	0.0013	0.0043
FLOW RT (L9/HK)	500.00	200.00
EXCH WEIGHT (LB)	41.75	11.60
FIN LENGTH (IN)	12.000	12.000
LVR LENGTH (IN)	12.000	12.000
FIN HEIGHT (IN)	1.205	0.510
FIN THKNSS (IN)	0.015	0.015
NUMBER OF FINS	32.00	32.00
FIN EPSILON	0.71191	0.80061

STEADY-STATE SYSTEM POWER CONDITIONS
VOLTAGE (VOLTS) 18.500
CURRENT (AMPS) 65.100
POWER (WATTS) 1204.357

SYSTEM PERFORMANCE DESCRIPTION DATE 01/21/77 1109
 54 IF UNITS MODEL NO. CX952-70-31
 WIRED AS 18 PARALLEL SETS OF 3 UNITS IN SERIES

THEMADIELECTRIC DEVICE PHYSICAL DESCRIPTION

HEIGHT (INCH)= 0.4300 WEIGHT (GRAM)= 21.129 RESISTANCE (OHM)= 1.579
 COLD M*CP PER IE DEVICE (WATT*SEC/C)= 0.55448E+02
 HOT M*CP PER IE DEVICE (WATT*SEC/C)= 0.26973E+03
 COLD, HOT, INTERSTAGE CONDUCTIVITY (WATT/SQ IN/C)= 6.3, 6.3, 20.0

I.F. MATERIAL	COMM	COMM
COUPLE DISTRIBUTION=	70.00	31.00
PELLET WIDTH (INCH)=	0.0550	0.0850
PELLET LENGTH (INCH)=	0.0630	0.1510
STAGE AREA (SQ INCH)=	1.4400	1.4400
ALUMINA THKNS (INCH)=	0.0300	0.0300
BUS-BAR THKNS (INCH)=	0.0150	0.0320

EXCHANGER FACTOR	HOT	COLD
FLUID TYPE	WATER	60%EG
PLATE TEMP (K)	303.38	287.75
INLET TEMP (K)	294.30	294.30
OUTLET TEMP (K)	300.76	289.93
HT OUT OF FLD(K)	-1704.44	508.37
PLT-PLT LOSS (K)	3.06	3.06
PRANDTL NUMBER	6.54773	42.70318
REYNOLDS NUMBER	64.65	27.34
NUSSELT NUMBER	6.4945	5.3263
HEAT TRANSF COEFF	129.4354	77.7641
PRSS DROP (IN H2O)	0.0927	1.6939
SEND FTR (IN H2O)	0.0013	0.0096
FLUX WT (LH/HK)	500.00	300.00
EXCH WEIGHT (LB)	41.75	11.60
FIN LENGTH (IN)	12.000	12.000
LVR LENGTH (IN)	12.000	12.000
FIN HEIGHT (IN)	1.205	0.510
FIN THKNSS (IN)	0.015	0.015
NUMBER OF FINS	32.00	32.00
FIN EPSILON	0.71192	0.66691

STEADY-STATE SYSTEM POWER CONDITIONS

VOLTAJE (VOLTS) 18.500
 CURRENT (AMPS) 65.200
 POWER (WATTS) 1206.208

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SYSTEM PERFORMANCE DESCRIPTION DATE 01/21/77 1240
 54 TF UNITS MODEL NO. CX952-70-31
 WIRED AS 18 PARALLEL SETS OF 3 UNITS IN SERIES

THERMOELECTRIC DEVICE PHYSICAL DESCRIPTION
 HEIGHT (INCH)= 0.4300 WEIGHT (GRAM)= 21.129 RESISTANCE (OHM)= 1.579
 COLD M*CP PER TE DEVICE (WATT*SEC/C)= 0.5544E+02
 HOT M*CP PER TE DEVICE (WATT*SEC/C)= 0.26473E+03
 COLD, HOT, INTERSTAGE CONDUCTIVITY (WATT/SQ IN/C)= 6.3, 6.3, 20.0

T.E. MATERIAL	COMM	COMM
COUPLE DISTRIBUTION	70.00	31.00
PELLET WIDTH (INCH)	0.0550	0.0450
PELLET LENGTH (INCH)	0.0630	0.1510
STAGE AREA (SQ INCH)	1.4400	1.4400
ALUMINA THKNS (INCH)	0.0300	0.0300
BUS-BAR THKNS (INCH)	0.0150	0.0320

EXCHANGER FACTOR	HOT	COLD
FLUID TYPE	WATER	60XEG
PLATE TEMP (K)	303.44	284.95
INLET TEMP (K)	294.30	294.30
OUTLET TEMP (K)	300.80	291.54
HT DUT OF FLOW	-1715.17	517.12
PLT-PLT LOSS (K)	2.83	2.83
PRANDTL NUMBER	6.33448	41.58694
REYNOLDS NUMBER	64.68	46.90
NUSSOLT NUMBER	6.4945	5.5879
HEAT TRANSFER COEFF	129.4428	81.5263
PRSS DROP (IN H2O)	1.0986	2.7545
RENO FTR (IN H2O)	0.0013	0.0266
FLOW RT (LM/HK)	500.00	500.00
EXCH WEIGHT (LB)	41.75	11.60
FIN LENGTH (IN)	12.000	12.000
LVR LENGTH (IN)	12.000	12.000
FIN HEIGHT (IN)	1.205	0.510
FIN THKNS (IN)	0.015	0.015
NUMBER OF FINS	32.00	32.00
FIN EPSILON	0.71194	0.49757

STEADY-STATE SYSTEM POWER CONDITIONS
 VOLTAGE (VOLTS) 14.500
 CURRENT (AMPS) 65.279
 POWER (WATTS) 1207.660

SYSTEM PERFORMANCE DESCRIPTION DATE 01/21/77 1240
 54 TE UNITS MODEL NO. CX952-70-31
 WIRED AS 1A PARALLEL SETS OF 3 UNITS IN SERIES

THERMOELECTRIC DEVICE PHYSICAL DESCRIPTION
 HEIGHT (INCH)= 0.4300 WEIGHT (GRAM)= 21.129 RESISTANCE (OHM)= 1.579
 COLD M*CP PER TE DEVICE (WATT*SEC/C)= 0.5544E+02
 HOT M*CP PER TE DEVICE (WATT*SEC/C)= 0.26973E+03
 COLD, HOT, INTERSTAGE CONDUCTIVITY (WATT/SQ IN/C)= 6.3, 6.3, 20.0

T.E. MATERIAL	COMM	COMM
COUPLE DISTRIBUTION=	70.00	31.00
PELLET WIDTH (INCH)=	0.0550	0.0850
PELLET LENGTH (INCH)=	0.0630	0.1510
STAGE AREA (SQ INCH)=	1.4400	1.4400
ALUMINA THKNS (INCH)=	0.0300	0.0300
BUS-BAR THKNS (INCH)=	0.0150	0.0320

EXCHANGER FACTOR	HOT	COLD
FLUID TYPE	WATER	60%EG
PLATE TEMP (K)	303.47	289.52
INLET TEMP (K)	294.30	294.30
OUTLET TEMP (K)	300.83	292.39
HT OUT OF FLD (K)	-1720.81	521.18
PLT-PLT LOSS (K)	2.73	2.73
PRANDTL NUMBER	6.33276	41.11147
REYNOLDS NUMBER	64.70	66.49
NUSSELT NUMBER	6.4945	5.8258
HEAT TRANSF CUEF	129.4467	84.9690
PRSS DRP (IN H ₂ O)	0.0986	3.8236
BEND FTR (IN H ₂ O)	0.0013	0.0522
FLOW RT (LM/HK)	500.00	700.00
EXCH WEIGHT (LB)	41.75	11.60
FIN LENGTH (IN)	12.000	12.000
LVR LENGTH (IN)	12.000	12.000
FIN HEIGHT (IN)	1.205	0.510
FIN THKNSS (IN)	0.015	0.015
NUMBER OF FINS	32.00	32.00
FIN EPSILON	0.71194	0.39990

STEADY-STATE SYSTEM POWER CONDITIONS
 VOLTAGE (VOLTS) 18.500
 CURRENT (AMPS) 65.314
 POWER (WATTS) 1208.311

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